Volume 3: Residential Measures

Ameren Missouri TRM – Volume 3: Residential Measures Revision Log

Revision	Date	Description
1.0	05/30/2018	Initial version filed for Commission approval.
2.0	12/21/2018	Updated "Deemed Tables" with PY2017 Evaluation results per Stipulation and Agreement (File No. EO-2018-0211). Added Demand Response language per Stipulation and Agreement.
3.0	1/01/2020	Updated "Deemed Tables" with PY2018 Evaluation results. Also includes revisions to HVAC measures and multifamily measures, based on feedback from evaluation contractor. This includes updates to Volume 3 of the TRM.
4.0	10/15/2020	Updated "Deemed Tables" with PY2019 Evaluation results and other revisions to improve consistency with Deemed tables.
5.0	09/15/21	Updated "Deemed Tables" with PY2020 Evaluation results and other revisions to improve consistency with Deemed tables.
6.0	09/26/2022	Updated "Deemed Tables" with PY2021 Evaluation results and other revisions to improve consistency with Deemed Tables. Other revisions include updates to incremental costs for low flow showerheads, in-service rates for low flow showerheads and faucet aerators based on PY2021 evaluation, incorporation of SEER to SEER2 and HSPF to HSPF2 conversion factors due to upcoming Code of Federal Regulation testing procedures, and updates to PTHP and PTAC baseline code efficiencies.
7.0	10/05/2023	Addition of Pay As You Save (PAYS®) ISR's. Added language to clarify that ccASHP's must meet the majority of a home's heating needs. Updated HVAC baselines for heat pumps to CFR standards, with a TRM effective date of 1/1/2024 to allow for sell-through; Updates to lighting measures to address EISA updates to general service lamps (GSL), effective 8/1/2023. Updated deemed costs of light bulbs to reflect first year cost per bulb.

Volume 3: Residential Measures

3.1 Appliances

3.1.1 Refrigerator and Freezer Recycling

DESCRIPTION

This measure describes savings from the retirement and recycling of inefficient but operational refrigerators and freezers. Savings are provided in two ways. First, a regression equation is provided that requires the use of key inputs describing the retired unit (or population of units) and is based on a 2013 workpaper provided by Cadmus using data from a 2012 ComEd metering study and metering data from a Michigan study. The second methodology is a deemed approach based on 2011 Cadmus analysis of data from a number of evaluations.[1]

The savings are equivalent to the unit energy consumption of the retired unit and should be claimed for the assumed remaining useful life of that unit. A Part Use Factor is applied to account for those secondary units that are not in use throughout the entire year. The user should note that the regression algorithm is designed to provide an accurate portrayal of savings for the population as a whole and includes those parameters that have a significant effect on the consumption. The precision of savings for individual units will vary. This measure also includes a section accounting for the interactive effect of reduced waste heat on the heating and cooling loads.

This measure was developed to be applicable to the following program type: ERET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

N/A

DEFINITION OF BASELINE EQUIPMENT

The existing inefficient unit must be operational and have a capacity of between 10 and 30 cubic feet.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated remaining useful life of the recycling units is 8 years.[2]

DEEMED MEASURE COST

Measure cost includes the cost of pickup and recycling of the refrigerator and should be based on actual costs of running the program. If unknown, assume $140 per unit.[3]

LOADSHAPE

Refrigeration RES
Freezer RES

Algorithm

CALCULATION OF SAVINGS

ENERGY SAVINGS

Regression analysis: Refrigerators

Daily energy savings for refrigerators are based upon a linear regression model using the following coefficients:[4]

[1] Cadmus "2010 Residential Great Refrigerator Roundup Program – Impact Evaluation," 2011.
[2] KEMA "Residential Refrigerator Recycling Ninth Year Retention Study," 2004.
[3] Based on average program costs for SCE Refrigerator Appliance Recycling Program. Innovologie, "Appliance Recycling Program Retailer Trial Final Report," a report prepared for Southern California Edison, 2013.
[4] Coefficients provided in May 13, 2016, Cadmus evaluation report; Ameren Missouri Refrigerator Recycling Impact and Process Evaluation: PY2015.

Independent Variable Description	Estimate Coefficient
Intercept	0.5822
Age (years)	0.0269
Pre-1990 (=1 if manufactured pre-1990)	1.0548
Size (cubic feet)	0.0673
Dummy: Side-by-Side (= 1 if side-by-side)	1.0706
Dummy: Single Door (= 1 if single door)	-1.9767
Dummy: Primary Usage Type (in absence of the program) (= 1 if primary unit)	0.6046
Interaction: Located in Unconditioned Space x CDD/365	0.0200
Interaction: Located in Unconditioned Space x HDD/365	-0.0447

$$\Delta kWh_{Unit} = \Big[0.5822 + (Age * 0.0269) + (Pre - 1990 * 1.0548) + (Size * 0.0673) + (Side - by - side * 1.0706) + (Single - door * -1.9767) + (Primary\ Usage * 0.6046) + \left(\frac{CDD}{365} * Unconditioned * 0.0200\right) + \left(\frac{HDD}{365} * Unconditioned * -0.0447\right)\Big] * Days * Part\ Use\ Factor$$

Where:

Age = Age of retired unit
Pre-1990 = Pre-1990 dummy (=1 if manufactured pre-1990, else 0)
Size = Capacity (cubic feet) of retired unit
Side-by-Side = Side-by-side dummy (= 1 if side-by-side, else 0)
Single-Door = Single-door dummy (= 1 if single-door, else 0)
Primary Usage = Primary Usage Type (in absence of the program) dummy (= 1 if Primary, else 0. If unknown, assume 0.262.[5])
CDD = Cooling Degree Days
= 1678:[6]
Unconditioned = If unit in unconditioned space = 1, otherwise 0. If unknown, assume 0.64.[7]
HDD = Heating Degree Days
= 4486[8]
Days = Days per year
= 365
Part Use Factor = To account for those units that are not running throughout the entire year. If available, Part-Use Factor participant survey results should be used. If not available, assume 0.864.[9]

Deemed approach: Refrigerators

$$\Delta kWh_{Unit} = UEC * Part\ Use\ Factor$$

Where:

UEC = Unit Energy Consumption
= 1181 kWh[10]
Part Use Factor = To account for those units that are not running throughout the entire year. If available, Part-Use Factor participant survey results should be used. If not available, assume 0.864.[11]
ΔkWh_{Unit} = 1181 * 0.864
= 1020 kWh

[5] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.
[6] Based on climate normals CDD data, with a base temp of 65°F.
[7] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.
[8] Based on climate normals HDD data, with a base temp of 65°F.
[9] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.
[10] This value is taken from the 2016 Cadmus evaluation of Ameren Missouri Refrigerator Recycling PY2015.
[11] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.

Regression analysis: Freezers:

Daily energy savings for freezers are based upon a linear regression model using the following coefficients:[12]

Independent Variable Description	Estimate Coefficient
Intercept	-0.8918
Age (years)	0.0384
Pre-1990 (=1 if manufactured pre-1990)	0.6952
Size (cubic feet)	0.1287
Chest Freezer Configuration (=1 if chest freezer)	0.3503
Interaction: Located in Unconditioned Space x CDD	0.0695
Interaction: Located in Unconditioned Space x HDD	-0.0313

$$\Delta kWh_{Unit} = [-0.8918 + (Age * 0.0384) + (Pre-1990 * 0.6952) + (Size * 0.1287) + (Chest\ Freezer * 0.3503) + (CDD/365 * Unconditioned * 0.0695) + (HDD/365 * Unconditioned * -0.0313)] * Part\ Use\ Factor$$

Where:

Age = Age of retired unit
Pre-1990 = Pre-1990 dummy (=1 if manufactured pre-1990, else 0)
Size = Capacity (cubic feet) of retired unit
Chest Freezer = Chest Freezer dummy (= 1 if chest freezer, else 0)
CDD = Cooling Degree Days (see table in refrigerator section)
Unconditioned = If unit in unconditioned space = 1, otherwise 0. If unknown, assume 0.67.[13]
HDD = Heating Degree Days (see table in refrigerator section)
Days = Days per year = 365
Part Use Factor = To account for those units that are not running throughout the entire year. If available, Part-Use Factor participant survey results should be used. If not available, assume 0.778.[14]

Deemed approach: Freezers

$$\Delta kWh_{Unit} = UEC * Part\ Use\ Factor$$

Where:

$UEC_{Reitred}$ = Unit Energy Consumption of retired unit
= 1061 kWh[15]
Part Use Factor = To account for those units that are not running throughout the entire year. If available, Part-Use Factor participant survey results should be used. If not available, assume 0.778.[16]
ΔkWh_{Unit} = 1061 * 0.778
= 825 kWh

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh_{unit} * CF$$

Where:

ΔkWh_{unit} = Savings provided in algorithm above (not including $\Delta kWh_{wasteheat}$)
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor[17]
Refrigerators = 0.0001285253
Freezers = 0.0001285253

[12] Coefficients provided in May 13, 2016, Cadmus evaluation report; Ameren Missouri Refrigerator Recycling Impact and Process Evaluation: PY2015.
[13] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.
[14] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.
[15] This value is taken from the 2016 Cadmus evaluation of Ameren Missouri Refrigerator Recycling PY2015.
[16] Ameren Missouri Appliance Recycling Impact and Process Evaluation: PY2019.
[17] Based on Ameren Missouri 2016 Loadshape for Residential Refrigeration and Freezer End-Use.

NATURAL GAS SAVINGS

$$\Delta Therms = \Delta kWh_{Unit} * WHFeHeatGas * 0.03412$$

Where:

ΔkWh_{Unit} = kWh savings calculated from either method above, not including the $\Delta kWh_{WasteHeat}$

WHFeHeatGas = Waste Heat Factor for Energy to account for gas heating increase from removing waste heat from refrigerator/freezer
= - (HF / $\eta Heat_{Gas}$) * %GasHeat
If unknown, assume 0

HF = Heating Factor or percentage of reduced waste heat that must now be heated
= 58% for unit in heated space[18]
= 0% for unit in heated space or unknown

$\eta Heat_{Gas}$ = Efficiency of heating system
= 71%[19]

%GasHeat = Percentage of homes with gas heat – see table below.

0.03412 = Converts kWh to therms

Heating Fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65%[20]

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[18] Based on 212 days where HDD 65>0, divided by 365.25.

[19] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences. The predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the state. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

[20] Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see “HC6.9 Space Heating in Midwest Region.xls.”

3.1.2 Air Purifier/Cleaner

DESCRIPTION
An air purifier (cleaner) meeting the efficiency specifications of ENERGY STAR® is purchased and installed in place of a model meeting the current federal standard.

This measure was developed to be applicable to the following program types: TOS and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
The efficient equipment is defined as an air purifier meeting the efficiency specifications of ENERGY STAR® as provided below.

1. Must produce a minimum 50 Clean Air Delivery Rate (CADR) for Dust[21] to be considered under this specification.
2. Minimum Performance Requirement: = 2.0 CADR/Watt (Dust)
3. Standby Power Requirement: = 2.0 Watts Qualifying models that perform secondary consumer functions (e.g., clock, remote control) must meet the Standby Power Requirement.
4. UL Safety Requirement: Models that emit ozone as a byproduct of air cleaning must meet UL Standard 867 (ozone production must not exceed 50ppb)

DEFINITION OF BASELINE EQUIPMENT
The baseline equipment is assumed to be a conventional unit.[22]

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The measure life is assumed to be 9 years.[23]

DEEMED MEASURE COST
The incremental cost for this measure is $70.[24]

LOADSHAPE
HVAC RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS[25]

$$Energy\ Savings\ (kWh_{Year}) = \{CADR\ \times (1/Eff_{BL} - 1/Eff_{ES}) \times (Hr_{oper}) + (SBBL - SBES) \times (24 - Hr_{oper})\} \times 365/1000\ *\ ISR$$

Where:

CADR = Clean air recovery rate for dust
Eff_{BL} = Clean air recovery rate for dust per watt for baseline unit
Eff_{ES} = Clean air recovery rate for dust per watt for ENERGY STAR® unit
Hr_{oper} = Hours per day of operation
SBBL = Standby for baseline unit
SBES = Standby for ENERGY STAR® unit
365 = Days/year
1,000 = Conversion factor (Wh/kWh)

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[21] Measured according to the latest ANSI/AHAM AC-1 (AC-1) Standard.
[22] As defined as the average of non-ENERGY STAR® products found in EPA research, 2011, ENERGY STAR® Qualified Room Air Cleaner Calculator.
[23] ENERGY STAR® Qualified Room Air Cleaner Calculator.
[24] Ameren Missouri MEEIA 2016-18 TRM, January 1, 2018.
[25] ENERGY STAR® Qualified Room Air Cleaner Calculator.

Term	Value[26]
CADR	157.56
EFF_{BL}	1.00
EFF_{ES}	3.00
Hr_{oper}	16
SB_{BL}	1.00
SB_{ES}	0.391
ISR	94%

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Gross customer annual kWh savings for the measure
CF = 0.0004660805

NATURAL GAS SAVINGS
N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
There are no operation and maintenance cost adjustments for this measure.[27]

MEASURE CODE:

[26] Ameren Missouri Efficient Products Evaluation PY2018.
[27] Some types of room air cleaners require filter replacement or periodic cleaning, but this is likely to be true for both efficient and baseline units and so no difference in cost is assumed.

3.1.3 Clothes Dryer

DESCRIPTION

This measure relates to the installation of a residential clothes dryer meeting the ENERGY STAR® criteria. ENERGY STAR® qualified clothes dryers save energy through a combination of more efficient drying and reduced runtime of the drying cycle. More efficient drying is achieved through increased insulation, modifying operating conditions such as air flow and/or heat input rate, improving air circulation through better drum design or booster fans, and improving efficiency of motors. Reducing the runtime of dryers through automatic termination by temperature and moisture sensors is believed to have the greatest potential for reducing energy use in clothes dryers.[28] ENERGY STAR® provides criteria for both gas and electric clothes dryers.

This measure was developed to be applicable to the following program types: TOS and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Clothes dryer must meet the ENERGY STAR® criteria, as required by the program.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a clothes dryer meeting the minimum federal requirements for units manufactured on or after January 1, 2015.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 14 years.[29]

DEEMED MEASURE COST

Dryer Size	Incremental Cost[30]
Standard	$75
Compact	$105

LOADSHAPE

Miscellaneous RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \left(\frac{Load}{CEFbase} - \frac{Load}{CEFeff}\right) * Ncycles * \%Electric$$

Where:

Load = The average total weight (lbs) of clothes per drying cycle. If dryer size is unknown, assume standard.

Dryer Size	Load (lbs)[31]
Standard	8.45
Compact	3

[28] ENERGY STAR® Market & Industry Scoping Report. Residential Clothes Dryers. Table 8. November 2011. http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf

[29] Based on an average estimated range of 12-16 years. ENERGY STAR® Market & Industry Scoping Report. Residential Clothes Dryers. November 2011. http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf

[30] Cost based on ENERGY STAR® Savings Calculator for ENERGY STAR® Qualified Appliances. https://www.energystar.gov/sites/default/files/asset/document/appliance_calculator.xlsx

[31] Based on ENERGY STAR® test procedures. https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers

CEFbase = Combined energy factor (CEF) (lbs/kWh) of the baseline unit is based on existing federal standards energy factor and adjusted to CEF as performed in the ENERGY STAR® analysis.[32] If product class unknown, assume electric, standard.

Product Class	CEFbase
Vented Electric, Standard (≥ 4.4 ft^3)	3.11
Vented Electric, Compact (120V) (< 4.4	3.01
Vented Electric, Compact (240V) (<4.4	2.73
Ventless Electric, Compact (240V) (<4.4	2.13
Vented Gas	2.84[33]

CEFeff = CEF (lbs/kWh) of the ENERGY STAR® unit based on ENERGY STAR® requirements.[34] If product class unknown, assume electric, standard.

Product Class	CEFeff
Vented or Ventless Electric, Standard (≥ 4.4 ft^3)	3.93
Vented or Ventless Electric, Compact (120V) (< 4.4	3.80
Vented Electric, Compact (240V) (< 4.4 ft^3)	3.45
Ventless Electric, Compact (240V) (< 4.4 ft^3)	2.68
Vented Gas	3.48[35]

Ncycles = Number of dryer cycles per year. Use actual data if available. If unknown, use 283 cycles per year.[36]

%Electric = The percent of overall savings coming from electricity
= 100% for electric dryers, 5% for gas dryers[37]

Using defaults provided above:

Product Class	ΔkWh
Vented Electric, Standard (≥ 4.4 ft^3)	145.7
Vented Electric, Compact (120V) (< 4.4 ft^3)	53.8
Vented Electric, Compact (240V) (<4.4 ft^3)	58.9
Ventless Electric, Compact (240V) (<4.4 ft^3)	74.3
Vented Gas	7.0

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001148238

[32] ENERGY STAR® Draft 2 Version 1.0 Clothes Dryers Data and Analysis.
[33] Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms.
[34] ENERGY STAR® Clothes Dryers Key Product Criteria. https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers
[35] Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms.
[36] Appendix D to Subpart B of Part 430 – Uniform Test Method for Measuring the Energy Consumption of Dryers.
[37] One hundred percent for electric dryer accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc.). Five percent for gas dryers was determined using a ratio of the electric to total savings from gas dryers given by ENERGY STAR® Draft 2 Version 1.0 Clothes Dryers Data and Analysis. Value reported in 2015 EPA ENERGY STAR® appliance calculator.

Using defaults provided above:

Product Class	ΔkW
Vented Electric, Standard (≥ 4.4 ft^3)	0.0251
Vented Electric, Compact (120V) (< 4.4 ft^3)	0.0092
Vented Electric, Compact (240V) (<4.4 ft^3)	0.0101
Ventless Electric, Compact (240V) (<4.4 ft^3)	0.0128
Vented Gas	0.0012

NATURAL GAS ENERGY SAVINGS

Natural gas savings only apply to ENERGY STAR® vented gas clothes dryers.

$$\Delta Therm = \left(\frac{Load}{CEFbase} - \frac{Load}{CEFeff}\right) * Ncycles * Therm\_convert * \%Gas$$

Where:

Therm_convert = Conversion factor from kWh to therm
= 0.03413

%Gas = Percent of overall savings coming from gas
= 0% for electric units and 84% for gas units[38]

Using defaults provided above:

ΔTherm = (8.45/2.84 – 8.45/3.48) * 257 * 0.03413 * 0.84
= 4.03 therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[38] Zero percent for gas dryers accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc.). Eighty-four percent was determined using a ratio of the gas to total savings from gas dryers given by ENERGY STAR® Draft 2 Version 1.0 Clothes Dryers Data and Analysis.

3.1.4 Clothes Washer

DESCRIPTION

This measure relates to the installation of a clothes washer meeting the ENERGY STAR® (CEE Tier1), ENERGY STAR® Most Efficient (CEE Tier 2), or CEE Tier 3 minimum qualifications. If the Domestic Hot Water (DHW) and dryer fuels of the installations are unknown (for example through a retail program), savings are based on a weighted blend using RECS data (the resultant values (kWh, therms and gallons of water) are provided). The algorithms can also be used to calculate site-specific savings where DHW and dryer fuels are known.

This measure was developed to be applicable to the following program types: TOS and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Clothes washer must meet the ENERGY STAR® (CEE Tier1), ENERGY STAR® Most Efficient (CEE Tier 2), or CEE Tier 3 minimum qualifications (provided in the table below), as required by the program.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard-sized clothes washer meeting the minimum federal baseline as of March 2015.[39]

Efficiency Level		Top loading >2.5 Cu ft	Front Loading >2.5 Cu ft
Baseline	Federal Standard	≥1.29 IMEF, ≤8.4 IWF	≥1.84 IMEF, ≤4.7 IWF
Efficient	ENERGY STAR®, CEE Tier 1	≥2.06 IMEF, ≤4.3 IWF	≥2.38 IMEF, ≤3.7 IWF
	ENERGY STAR® Most Efficient, CEE Tier 2	≥2.76 IMEF, ≤3.5 IWF	≥2.74 IMEF, ≤3.2 IWF
	CEE Tier 3	≥2.92 IMEF, ≤3.2 IWF	

The Integrated Modified Energy Factor (IMEF) includes unit operation, standby, water heating, and drying energy use, with the higher the value the more efficient the unit: "The quotient of the cubic foot (or liter) capacity of the clothes container divided by the total clothes washer energy consumption per cycle, with such energy consumption expressed as the sum of the machine electrical energy consumption, the hot water energy consumption, the energy required for removal of the remaining moisture in the wash load, and the combined low-power mode energy consumption."

The Integrated Water Factor (IWF) indicates the total water consumption of the unit, with the lower the value the less water required: "The quotient of the total weighted per-cycle water consumption for all 67 wash cycles in gallons divided by the cubic foot (or liter) capacity of the clothes washer."[40]

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 14 years.[41]

DEEMED MEASURE COST

The incremental cost assumptions are provided below:[42]

Efficiency Level	Incremental Cost
ENERGY STAR®, CEE Tier 1	$32
ENERGY STAR® Most Efficient, CEE TIER 2	$393
CEE TIER 3	$454

LOADSHAPE

Miscellaneous RES

Algorithm

[39] See http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/39.
[40] Definitions provided in ENERGY STAR® v7.1 specification on the ENERGY STAR® website.
[41] Based on DOE Chapter 8 Life-Cycle Cost and Payback Period Analysis.
[42] Based on weighted average of top loading and front loading units (based on available product from the California Energy Commission (CEC) Appliance database (https://cacertappliances.energy.ca.gov/Pages/ApplianceSearch.aspx) and cost data from Life-Cycle Cost and Payback Period Excel-based analytical tool. See "2015 Clothes Washer Analysis.xls" for details.

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \left[\left(Capacity * \frac{1}{IMEFbase} * Ncycles\right) * \left(\%CWbase + (\%DHWbase * \%Electric_{DHW}) + (\%Dryerbase * \%Electric_{Dryer})\right)\right] - \left[\left(Capacity * \frac{1}{IMEFeff} * Ncycles\right) * \left(\%CWeff + (\%DHWeff * \%Electric_{DHW}) + (\%Dryereff * \%Electric_{Dryer})\right)\right]$$

Where:

Capacity = Clothes washer capacity (cubic feet)
= Actual - If capacity is unknown, assume 3.45 cubic feet [43]
IMEFbase = Integrated Modified Energy Factor of baseline unit
IMEFeff = Integrated Modified Energy Factor of efficient unit
= Actual. If unknown, assume average values provided below.
Ncycles = Number of Cycles per year
= 271[44]
%CW = Percentage of total energy consumption for Clothes Washer operation (different for baseline and efficient unit – see table below)

%DHW = Percentage of total energy consumption used for water heating (different for baseline and efficient unit – see table below)
%Dryer = Percentage of total energy consumption for dryer operation (different for baseline and efficient unit – see table below)
$\%Electric_{DHW}$ = Percentage of DHW savings assumed to be electric
$\%Electric_{Dryer}$ = Percentage of dryer savings assumed to be electric

Efficiency Level	IMEFbase		
	Top loading >2.5 Cu ft	Front Loading >2.5 Cu ft	Weighted Average[45]
Federal Standard	1.29	1.84	1.66

Efficiency Level	IMEFeff		
	Top loading >2.5 Cu ft	Front Loading >2.5 Cu ft	Weighted Average[46]
ENERGY STAR®, CEE Tier 1	2.06	2.38	2.26
ENERGY STAR® Most Efficient, CEE Tier 2	2.76	2.74	2.74
CEE Tier 3	2.92		2.92

	Percentage of Total Energy Consumption[47]		
	%CW	%DHW	%Dryer
Federal Standard	8%	31%	61%
ENERGY STAR®, CEE Tier 1	8%	23%	69%
ENERGY STAR® Most Efficient, CEE Tier 2	14%	10%	76%
CEE Tier 3	14%	10%	76%

[43] Based on the average clothes washer volume of all units that pass the new federal standard on the CEC database of clothes washer products (accessed on 08/28/2014). If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.
[44] Weighted average of 271 clothes washer cycles per year (based on 2009 Residential Energy Consumption Survey (RECS) national sample survey of housing appliances section, Midwest Census Region for state of Missouri): http://www.eia.gov/consumption/residential/data/2009/. See "2015 Clothes Washer Analysis.xls" for details.
If utilities have specific evaluation results providing a more appropriate assumption for singlefamily or multifamily homes in a particular market or geographical area, then that should be used.
[45] Weighted average IMEF of Federal Standard rating for Front Loading and Top Loading units. Weighting is based upon the relative top v front loading percentage of available non-ENERGY STAR® product in the CEC database (accessed 08/28/2014). The relative weightings are: 67% front and 33% top for Baseline; 62% front and 38% top for ENERGY STAR CEE Tier 1; 98% front and 2% top for ENERGY STAR Most Efficient, CEE Tier 2; and 100% front for CEE Tier 3. See more information in "2015 Clothes Washer Analysis.xlsx."
[46] Weighting is based upon the relative top vs. front loading percentage of available product in the CEC database (accessed 08/28/2014).
[47] The percentage of total energy consumption that is used for the machine, heating the hot water, or by the dryer is different depending on the efficiency of the unit. Values are based on a weighted average of top loading and front-loading units based on data from DOE Life-Cycle Cost and Payback Analysis. See "2015 Clothes Washer Analysis.xls" for details.

DHW fuel	$\%Electric_{DHW}$
Electric	100%
Natural Gas	0%
Unknown	43%[48]

Dryer fuel	$\%Electric_{Dryer}$
Electric	100%
Natural Gas	0%
Unknown	90%[49]

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:[50]

Front Loaders:

	ΔkWH			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	149.3	52.6	96.4	-0.2
ENERGY STAR® Most Efficient, CEE Tier 2	222.1	85.9	132.2	-4.0
CEE Tier 3	243.1	104.8	137.2	-1.1

Top Loaders:

	ΔkWH			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	149.3	97.0	77.0	24.8
ENERGY STAR® Most Efficient, CEE Tier 2	222.1	132.6	117.1	27.5
CEE Tier 3	243.1	374.4	230.5	42.0

Weighted Average:

	ΔkWH			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	149.3	70.6	88.0	9.4
ENERGY STAR® Most Efficient, CEE Tier 2	222.1	80.9	137.5	-3.7
CEE Tier 3	243.1	98.4	143.2	-1.5

If the DHW and dryer fuel is unknown, the prescriptive kWH savings based on defaults provided above should be:

	ΔkWH		
Efficiency Level	Front Loaders	Top Loaders	Weighted Average
ENERGY STAR®, CEE Tier 1	112.8	89.6	99.0
ENERGY STAR® Most Efficient, CEE Tier 2	161.5	136.6	134.3
CEE Tier 3	424.6	154.8	151.8

[48] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

[49] Default assumption for unknown is based on percentage of homes with clothes washers that use an electric dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

[50] Note that the baseline savings for all cases (front, top and weighted average) is based on the weighted average baseline IMEF (as opposed to assuming front baseline for front-efficient unit and top baseline for top- efficient unit). The reasoning is that the support of the program of more efficient units (which are predominately front loading) will result in some participants switching from planned purchase of a top loader to a front loader.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer peak coincidence factor for measure
= 0.0001148238

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

Front Loaders:

	ΔkW			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	0.022	0.008	0.015	0.000
ENERGY STAR® Most Efficient, CEE Tier 2	0.033	0.013	0.020	-0.001
CEE Tier 3	0.037	0.016	0.021	0.000

Top Loaders:

	ΔkW			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	0.022	0.015	0.012	0.004
ENERGY STAR® Most Efficient, CEE Tier 2	0.033	0.020	0.018	0.004
CEE Tier 3	0.037	0.056	0.035	0.006

Weighted Average:

	ΔkW			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	0.022	0.011	0.013	0.001
ENERGY STAR® Most Efficient, CEE Tier 2	0.033	0.012	0.021	-0.001
CEE Tier 3	0.037	0.015	0.022	0.000

If the DHW and dryer fuel is unknown, the prescriptive kW savings should be:

	ΔkW		
Efficiency Level	Front Loaders	Top Loaders	Weighted Average
ENERGY STAR®, CEE Tier 1	0.013	0.017	0.015
ENERGY STAR® Most Efficient, CEE Tier 2	0.021	0.024	0.020
CEE Tier 3	0.023	0.064	0.023

NATURAL GAS SAVINGS

$$\Delta Therms = \left[\left[\left(Capacity * \frac{1}{IMEFbase} * Ncycles\right) * \left(\left(\%DHWbase * \%Natural\ Gas_{DHW} * R_{eff}\right) + \left(\%Dryerbase * \%Gas_{Dryer}\right)\right)\right] - \left[\left(Capacity * \frac{1}{IMEFeff} * Ncycles\right) * \left(\left(\%DHW_{eff} * \%Gas_{DHW} * \%Natural\ Gas\_DHW * R\_eff\right) + \left(\%Dryereff * \%Gas_{Dryer}\right)\right)\right]\right] * Therm\_convert$$

Where:

$\%Gas_{DHW}$ = Percentage of DHW savings assumed to be Natural Gas

R_{eff} = Recovery efficiency factor
= 1.26[51]
$\%Gas_{Dryer}$ = Percentage of dryer savings assumed to be Natural Gas
Therm_convert = Conversion factor from kWh to therm
= 0.03412

Other factors as defined above.

DHW fuel	$\%Gas_{DHW}$
Electric	0%
Natural Gas	100%
Unknown	57%[52]

Dryer fuel	$\%Gas_{Dryer}$
Electric	0%
Natural Gas	100%
Unknown	10%[52]

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

Front Loaders:

	ΔTherms			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	0.0	2.2	2.5	4.7
ENERGY STAR® Most Efficient, CEE Tier 2	0.0	3.8	3.6	7.4
CEE Tier 3	0.0	8.1	11.3	19.4

Top Loaders:

	ΔTherms			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	0.0	4.2	1.8	6.0
ENERGY STAR® Most Efficient, CEE Tier 2	0.0	5.9	3.1	8.9
CEE Tier 3	0.0	5.9	3.6	9.6

Weighted Average:

	ΔTherms			
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer
ENERGY STAR®, CEE Tier 1	0.0	3.4	2.1	5.5
ENERGY STAR® Most Efficient, CEE Tier 2	0.0	6.1	2.9	9.0
CEE Tier 3	0.0	6.2	3.4	9.6

If the DHW and dryer fuel is unknown, the prescriptive therm savings should be:

ΔTherms

[51] To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency. (http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf). Therefore, a factor of 0.98/0.78 (1.26) is applied.
[52] Default assumption for unknown fuel is based EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then that should be used.

Efficiency Level	Front Loaders	Top Loaders	Weighted Average
ENERGY STAR®, CEE Tier 1	1.51	2.52	2.11
ENERGY STAR® Most Efficient, CEE Tier 2	2.52	3.60	3.71
CEE Tier 3	5.66	3.70	3.84

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta Water\ (gallons) = Capacity * (IWFbase - IWFeff) * Ncycles$$

Where:

IWFbase = Integrated Water Factor of baseline clothes washer
= 5.92[53]

IWFeff = Water Factor of efficient clothes washer
= Actual - If unknown assume average values provided below

Other factors as defined above.

Using the default assumptions provided above, the prescriptive water savings for each efficiency level are presented below:

Efficiency Level	IWF[54]			ΔWater (gallons per year)		
	Front Loaders	Top Loaders	Weighted Average	Front Loaders	Top Loaders	Weighted Average
Federal Standard	4.7	8.4	5.92	N/A		
ENERGY STAR®, CEE Tier 1	3.7	4.3	3.93	934	3,828	1,857
ENERGY STAR® Most Efficient, CEE Tier 2	3.2	3.5	3.21	1,400	4,575	2,532
CEE Tier 3	3.2		3.20	1,400	7,842	2,538

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[53] Weighted average IWF of Federal Standard rating for front loading and top loading units. Weighting is based upon the relative top vs. front loading percentage of available non-ENERGY STAR® products in the CEC database.

[54] IWF values are the weighted average of the new ENERGY STAR® specifications. Weighting is based upon the relative top vs. front loading percentage of available ENERGY STAR® and ENERGY STAR® Most Efficient products in the CEC database. See "2015 Clothes Washer Analysis.xls" for the calculation.

3.1.5 Dehumidifier

DESCRIPTION

A dehumidifier meeting the minimum qualifying efficiency standard established by the current ENERGY STAR® Version 4.0 (effective 2/1/2016) is purchased and installed in a residential setting in place of a unit that meets the minimum federal standard efficiency.

This measure was developed to be applicable to the following program types: TOS and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the new dehumidifier must meet the ENERGY STAR® standards as defined below:

Capacity (pints/day)	ENERGY STAR® Criteria (L/kWh)
<75	≥2.00
75 to ≤185	≥2.80

Qualifying units must be equipped with an adjustable humidistat control or must have a remote humidistat control to operate.

DEFINITION OF BASELINE EQUIPMENT

The baseline for this measure is defined as a new dehumidifier that meets the federal standard efficiency standards. The federal standard for dehumidifiers as of October 2012 is defined below:

Capacity (pints/day)	Federal Standard Criteria (L/kWh)
Up to 35	≥1.35
> 35 to ≤45	≥1.50
> 45 to ≤ 54	≥1.60
> 54 to ≤ 75	≥1.70
> 75 to ≤ 185	≥2.50

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 12 years.[55]

DEEMED MEASURE COST

The assumed incremental capital cost for this measure is $5.[56]

LOADSHAPE

Cooling RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (((Avg\ Capacity * 0.473) / 24) * Hours) * (1 / (L/kWh_{Base}) - 1 / (L/kWh_{Eff}))$$

Where:

Avg Capacity = Average capacity of the unit (pints/day)
= Actual, if unknown assume capacity in each capacity range as provided in table below, or if capacity range unknown assume average.

0.473 = Constant to convert Pints to Liters

24 = Constant to convert Liters/day to Liters/hour

[55] Lifetime determined by EPA research, 2012. ENERGY STAR® Qualified Room Air Cleaner Calculator. (ENERGY STAR® Appliance Calculator.xlsx).

[56] Incremental costs determined by EPA research on available models, July 2016. ENERGY STAR® Qualified Room Air Cleaner Calculator. (ENERGY STAR® Appliance Calculator.xlsx).

Hours = Run hours per year
= 1632[57]
L/kWh = Liters of water per kWh consumed, as provided in tables above

Annual kWh results for each capacity class are presented below:

Capacity Range (pints/day)	Capacity Used (pints/day)	Federal Standard Criteria (≥ L/kWh)	ENERGY STAR® Criteria (≥ L/kWh)	Annual kWh		
				Federal Standard	ENERGY STAR®	Savings
≤25	20	1.35	2.0	477	322	155
> 25 to ≤35	30	1.35	2.0	714	482	232
> 35 to ≤45	40	1.5	2.0	857	643	214
> 45 to ≤ 54	50	1.6	2.0	1005	804	201
> 54 to ≤ 75	65	1.7	2.0	1,229	1,045	184
> 75 to ≤ 185	130	2.5	2.8	1,672	1,493	179
Average[58]						204

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0009474181

Summer coincident peak demand results for each capacity class are presented below:

Capacity Range (pints/day)	Annual Summer peak kW Savings
≤25	0.095
> 25 to ≤35	0.142
> 35 to ≤45	0.131
> 45 to ≤ 54	0.123
> 54 to ≤ 75	0.113
> 75 to ≤ 185	0.110
Average	0.125

NATURAL GAS SAVINGS
N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

[57] Based on 24-hour operation over 68 days of the year. ENERGY STAR® Qualified Room Air Cleaner Calculator. (ENERGY STAR® Appliance Calculator.xlsx).

[58] The relative weighting of each product class is based on number of units on the ENERGY STAR® certified list. See “Dehumidifier Calcs.xls."

3.1.6 Dehumidifier Recycling

DESCRIPTION

This measure describes the savings resulting from the retirement of existing residential, inefficient dehumidifier units from service prior to end of their natural life. This measure assumes that a percentage of these units will be replaced with a baseline standard efficiency unit (note that if the unit is actually replaced by a new ENERGY STAR® qualifying unit, the savings increment between baseline and ENERGY STAR® will be recorded in the Efficient Products program).

This measure was developed to be applicable to the following program type: ERET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

N/A. This measure relates to the retiring of an existing inefficient unit.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is the existing inefficient dehumidifier unit.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 5 years.

DEEMED MEASURE COST

The incremental cost for this measure is $42.76.

LOADSHAPE

HVAC RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS[59]

Program Deemed Savings estimate:

Gross Electric Savings (kWh/unit)	Gross Demand Savings (kW/home)
139	.0648

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Gross customer annual kWh savings for the measure

CF = 0.0004660805

MEASURE CODE:

[59] Deemed value per 2018 MEMD database for a drop-off program.

3.1.7 Refrigerator

DESCRIPTION
A refrigerator meeting either ENERGY STAR®/CEE Tier 1 specifications or the higher efficiency specifications of CEE Tier 2 or CEE Tier 3 is installed instead of a new unit of baseline efficiency. The measure applies to TOS and early replacement programs.

This measure also includes a section accounting for the interactive effect of reduced waste heat on the heating and cooling loads.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
The high-efficiency level is a refrigerator meeting ENERGY STAR® specifications effective September 15th, 2014 (10% above federal standard), a refrigerator meeting CEE Tier 2 specifications (15% above federal standard), or CEE Tier 3 specifications (20% above federal standards).

DEFINITION OF BASELINE EQUIPMENT
Baseline efficiency is a new refrigerator meeting the minimum federal efficiency standard for refrigerators effective September 15th, 2014, for all programs except low-income direct install programs. For low-income programs, the baseline is the existing equipment.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
17 years[60]

DEEMED MEASURE COST
The full cost of a baseline unit is $742.[61]

The incremental cost to the ENERGY STAR® level is $11, to CEE Tier 2 level is $20, and to CEE Tier 3 is $59.[62]

LOADSHAPE
Refrigeration RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS
Savings by model may be pulled directly from ENERGY STAR® data. Alternatively, savings by product class may be calculated according to the algorithm below:

$$\Delta kWh_{Unit} = kWh_{\text{base}} - (kWh_{\text{new}} * (1 - \%Savings))$$

Where:

kWh_{base} = Baseline consumption,[63] assuming 22.5 ft^3 adjusted volume[64]
= Calculated using algorithms in table below, or using defaults provided based on 22.5 ft^3 adjusted volume [64]

%Savings = Specification of energy consumption below Federal Standard – see table below.

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[60] Mean from Figure 8.2.3, DOE, 2011-08-23 Technical Support Document for Energy Conservation Standards for Residential Refrigerators, Refrigerator-Freezers, and Freezers. http://www.regulations.gov/contentStreamer?objectId=0900006480f0c7df&disposition=attachment&contentType=pdf

[61] Configurations weighted according to table under Energy Savings. Values inflated 8.9% from 2009 dollars to 2015. Table 8.1.1, DOE, 2011-08-23 Technical Support Document for Energy Conservation Standards for Residential Refrigerators, Refrigerator-Freezers, and Freezers. http://www.regulations.gov/contentStreamer?objectId=0900006480f0c7df&disposition=attachment&contentType=pdf

[62] Configurations weighted according to table under Energy Savings. Values inflated 8.9% from 2009 dollars to 2015. Table 8.2.2, DOE, 2011-08-23 Technical Support Document for Energy Conservation Standards for Residential Refrigerators, Refrigerator-Freezers, and Freezers. http://www.regulations.gov/contentStreamer?objectId=0900006480f0c7df&disposition=attachment&contentType=pdf

[63] According to Federal Standard effective 9/15/14.

[64] DOE Building Energy Data Book, http://buildingsdatabook.eren.doe.gov/TableView.aspx?table=5.7.5.

Tier	%Savings
Energy Star® and CEE Tier 1	10%
Energy Star® Most Efficient and CEE Tier 2	15%
CEE Tier 3	20%

For low-income programs, the following table may be used to calculate baseline usage:

Age	*Bottom Freezer (16 cu ft)*	*Side-by-Side (14 cu ft)*	*Side-by-Side (15 cu ft)*	*Side-by-Side (16 cu ft)*	*Top Freezer (cu ft 14)*	*Top Freezer (15 cu ft)*	*Top Freezer (16 cu ft)*	*Top Freezer (17 cu ft)*	*Top Freezer (18 cu ft)*
2011-2015	483	592	592	592	374	374	374	412	412
2001 (after July-2010	724	747	747	747	556	556	556	613	613
1993-2001(before June)	962	1,139	1,139	1,139	861	861	861	962	962
1990-1992	1,519	1,617	1,617	1,617	1,272	1,272	1,272	1,432	1,432
1980-1989	1,992	2,119	2,119	2,119	1,668	1,668	1,668	1,877	1,877
Before 1980	2,523	2,684	2,684	2,684	2,112	2,112	2,112	2,377	2,377

<u>Additional Waste Heat Impacts</u>

For units in conditioned spaces in the home (if unknown, assume unit is in conditioned space).

$$\Delta kWh_{WasteHeat} = \Delta kWh * (WHFeHeatElectric + WHFeCool)$$

Where:

ΔkWh = kWh savings calculated from either method above

WHFeHeatElectric = Waste Heat Factor for Energy to account for electric heating increase from removing waste heat from refrigerator/freezer (if fossil fuel heating – see calculation of heating penalty in that section).

= - (HF / $\eta Heat_{Electric}$) * %ElecHeat

HF = Heating Factor or percentage of reduced waste heat that must now be heated

= 58% for unit in heated space or unknown [65]

= 0% for unit in unheated space

$\eta Heat_{Electric}$ = Efficiency in COP of Heating equipment

= Actual - If not available, use table below:[66]

%ElecHeat = Percentage of home with electric heat

System Type	Age of Equipment	HSPF Esitmate	ηHeat (COP Estimate)
Heat Pump	Before 2006	6.8	2.00
	2006-2014	7.7	2.26
	2015 on	8.2	2.40
Resistance	N/A	N/A	1.00
Unknown	N/A	N/A	1.28[67]

[65] Based on 212 days where HDD 65>0, divided by 365.25.

[66] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 and 2015 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[67] Calculation assumes 13% heat pump and 87% resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls." Average efficiency of heat pump is based on the assumption that 50% are units from before 2006 and 50% 2006-2014.

Heating Fuel	%ElecHeat
Electric	100%
Fossil Fuel	0%
Unknown	35%[68]

WHFeCool = Waste Heat Factor for Energy to account for cooling savings from removing waste heat from refrigerator/freezer.
= (CoolF / ηCool) * %Cool

CoolF = Cooling Factor or percentage of reduced waste heat that no longer needs to be cooled
= 40% for unit in cooled space or unknown[69]
= 0% for unit in uncooled space

ηCool = Efficiency in COP of Cooling equipment
= Actual - If not available, assume 2.8 COP[70]

%Cool = Percentage of home with cooling

Home	%Cool
Cooling	100%
No Cooling	0%
Unknown	91%[71]

Algorithms for the most common refrigerator configurations, kWh_{base}, $\Delta kWh_{WasteHeat}$ for unknown building characteristics and resulting deemed ΔkWh savings is provided below:

Product Class	Algorithm from Federal Standard	Baseline Usage kWh_{base}	Unit ΔkWh			$\Delta kWh_{WasteHeat}$			Total ΔkWh		
			ENERGY STAR® / CEE Tier 1	CEE Tier 2	CEE Tier 3	ENERGY STAR® / CEE Tier 1	CEE Tier 2	CEE Tier 3	ENERGY STAR® / CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	8.40AV + 385.4	574	57.4	86.1	114.8	-0.9	-1.4	-1.9	56.5	84.7	112.9
Side-by-Side w/ TTD (PC 7)	8.54AV + 432.8	625	62.5	93.75	125	-1.0	-1.5	-2.1	61.5	92.2	122.9
Bottom Freezer (PC 5)	8.85AV + 317.0	516	51.6	77.4	103.2	-0.8	-1.3	-1.7	50.8	76.1	101.5
Bottom Freezer w/ TTD (PC 5A)	9.25AV + 475.4	684	68.4	102.6	136.8	-1.1	-1.7	-2.2	67.3	100.9	134.6

If product class is unknown, the following table provides a market weighting that is applied to give a single deemed savings for each efficiency level:

Product Class	Market Weight[72]	Unit ΔkWh			$\Delta kWh_{WasteHeat}$			Total ΔkWh		
		Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3	Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3	Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	52%	59.2	88.8	118.4	-1.0	-1.5	-1.9	58.2	87.3	116.5
Side-by-Side w/ TTD (PC 7)	22%									
Bottom Freezer (PC 5)	13%									
Bottom Freezer w/ TTD (PC 5A)	13%									

[68] Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls."
[69] Based on 148 days where CDD 65>0, divided by 365.25.
[70] Starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER^2) + (1.12 * SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP.
[71] Based on 2009 Residential Energy Consumption Survey, see "HC7.9 Air Conditioning in Midwest Region.xls."
[72] Personal Communication from Melisa Fiffer, ENERGY STAR® Appliance Program Manager, EPA 10/26/14.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (\Delta kWh_{WasteHeatCooling}) * CF$$

Where:

$\Delta kWh_{WasteHeatCooling}$ = gross customer connected load kWh savings for the measure. Including any cooling system savings.

CF = Summer Peak Coincident Factor
= 0.0001285253[73]

Default values for each product class and unknown building characteristics are provided below:

Product Class	ΔkW		
	Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	0.0086	0.0130	0.0173
Side-by-Side w/ TTD (PC 7)	0.0094	0.0141	0.0188
Bottom Freezer (PC 5)	0.0078	0.0117	0.0155
Bottom Freezer w/ TTD (PC 5A)	0.0103	0.0155	0.0206

If product class is unknown, the following table provides a market weighting that is applied to give a single deemed savings for each efficiency level:

Product Class	Market Weight[74]	ΔkW		
		Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	52%	0.0089	0.0134	0.0178
Side-by-Side w/ TTD (PC 7)	22%			
Bottom Freezer (PC 5)	13%			
Bottom Freezer w/ TTD (PC 5A)	13%			

NATURAL GAS SAVINGS

Heating penalty for reduction in waste heat, only for units from conditioned space in gas heated home (if unknown, assume unit is from conditioned space).

$$\Delta Therms = \Delta kWh_{Unit} * WHFeHeatGas * 0.03412$$

Where:

ΔkWh_{Unit} = kWh savings calculated from either method above, not including the $\Delta kWh_{WasteHeat}$

WHFeHeatGas = Waste Heat Factor for Energy to account for gas heating increase from removing waste heat from refrigerator/freezer
= - (HF / $\eta Heat_{Gas}$) * %GasHeat

HF = Heating Factor or percentage of reduced waste heat that must now be heated
= 58% for unit in heated space or unknown[75]
= 0% for unit in unheated space

$\eta Heat_{Gas}$ = Efficiency of heating system
= 74%[76]

%GasHeat = Percentage of homes with gas heat

0.03412 = Converts kWh to therms

[73] Based on Ameren Missouri 2016 Loadshape for Residential Refrigeration End-Use.
[74] Personal Communication from Melisa Fiffer, ENERGY STAR® Appliance Program Manager, EPA 10/26/1.4.
[75] Based on 212 days where HDD 65>0, divided by 365.25.
[76] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 52% of Missouri homes - based on Energy Information Administration, 2009 Residential Energy Consumption Survey). Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.60*0.92) + (0.40*0.8)) * (1-0.15) = 0.74.

Heating Fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65%[77]

Default values for each product class and unknown building characteristics are provided below:

Product Class	ΔTherms		
	Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	-1.19	-1.78	-2.37
Side-by-Side w/ TTD (PC 7)	-1.29	-1.94	-2.58
Bottom Freezer (PC 5)	-1.07	-1.60	-2.13
Bottom Freezer w/ TTD (PC 5A)	-1.41	-2.12	-2.83

If product class is unknown, the following table provides a market weighting that is applied to give a single deemed savings for each efficiency level:

Product Class	Market Weight[78]	ΔTherms		
		Energy Star®/ CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	52%	-1.22	-1.84	-2.45
Side-by-Side w/ TTD (PC 7)	22%			
Bottom Freezer (PC 5)	13%			
Bottom Freezer w/ TTD (PC 5A)	13%			

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

[77] Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls."
[78] Personal Communication from Melisa Fiffer, ENERGY STAR® Appliance Program Manager, EPA 10/26/14.

3.1.8 Room Air Conditioner Recycling

DESCRIPTION

This measure describes the savings resulting from the retirement of existing residential, inefficient room air conditioner units from service prior to their natural end of life. This measure assumes that a percentage of these units will be replaced with a baseline standard efficiency unit (note that if it is actually replaced by a new ENERGY STAR® qualifying unit, the savings increment between baseline and ENERGY STAR® will be recorded in the Efficient Products program).

This measure was developed to be applicable to the following program type: ERET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

N/A. This measure relates to the retiring of an existing inefficient unit.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is the existing inefficient room air conditioning unit.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed remaining useful life of the existing room air conditioning unit being retired is 4 years.[79]

DEEMED MEASURE COST

The actual implementation cost for recycling the existing unit should be used.

LOADSHAPE

Cooling RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$DkWh = kWhexist - (\%replaced * kWhnewbase)$$

$$= \frac{Hours * BtuH}{EERexist * 1000} - (\%replaced * \frac{Hours * BtuH}{EERNewBase * 1000})$$

Where:

Hours	= Full Load Hours of room air conditioning unit
EERexist	= Efficiency of recycled unit
BtuH	= Average size of rebated unit. Use actual if available - if not, assume 8500[80]
	= Actual if recorded - If not, assume 9.0[81]
%replaced	= Percentage of units that are replaced
EERNewBase	= Efficiency of baseline unit
	= 10.9[82]

[79] One third of assumed measure life for room air conditioners.
[80] Based on maximum capacity average from the RLW Report; "Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008."
[81] The federal minimum for the most common type of unit (8000 – 13999 Btuh with side vents) from 1990-2000 was 9.0 EER, from 2000-2014 it was 9.8 EER, and is currently (2015) 10.9 CEER. Retirement programs will see a large array of ages being retired, and the true EER of many will have been significantly degraded. We have selected 9.0 as a reasonable estimate of the average retired unit. This is supported by material on the ENERGY STAR® website, which, if reverse-engineered, indicates that an EER of 9.16 is used for savings calculations for a 10-year old room air conditioner. Another statement indicates that units that are at least 10 years old use 20% more energy than a new ES unit, which equates to: 10.9EER/1.2 = 9.1 EER; http://www.energystar.gov/ia/products/recycle/documents/RoomAirConditionerTurn-InAndRecyclingPrograms.pdf.
[82] Minimum federal standard for capacity range and most popular class (without reverse cycle, with louvered sides, and 8,000 to 13,999 Btu/h). http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41.

Weather Basis (City based upon)	Hours[83]
St Louis, MO	860 for primary use and 556 for secondary use

Scenario	%replaced
Customer states unit will not be replaced	0%
Customer states unit will be replaced	100%
Unknown	76%[84]

Results using defaults provided above:

Weather Basis (City based upon)	ΔkWh		
	Unit not replaced	Unit replaced	Unknown
St Louis, MO	525.4	91.6	195.7

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = Summer Peak Coincidence Factor for measure
= 0.0009474181[85]

Results using defaults provided above:

Weather Basis (City based upon)	ΔkW		
	Unit not replaced	Unit replaced	Unknown
St Louis, MO	0.4978	0.0868	0.1854

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE

[83] Ameren Missouri PY2013 CoolSavers evaluation. http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf) to FLH for Central Cooling for the same locations (provided by AHRI: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) is 31%. This factor was applied to published CDD65 climate normals data to provide an assumption for FLH for Room AC.

[84] Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report." Report states that 63% were replaced with ENERGY STAR® units and 13% with non-ENERGY STAR®. However, this formula assumes all are non-ENERGY STAR® since the increment of savings between baseline units and ENERGY STAR® would be recorded by the Efficient Products program when the new unit is purchased.

[85] Based on Ameren Missouri 2016 loadshape for residential cooling end-use.

3.2 Electronics

3.2.1 Advanced Tier 1 Power Strips

DESCRIPTION

This measure applies to Tier 1 Advanced Power Strips (APS), which are multi-plug power strips with the ability to automatically disconnect specific connected loads depending upon the power draw of a master control load, also plugged into the strip. Power is disconnected from the switched (controlled) outlets when the master control load power draw is reduced below a certain adjustable threshold, thus turning off the appliances plugged into the switched outlets. By disconnecting, the standby load of the controlled devices, the overall load of a centralized group of equipment (i.e. entertainment centers and home office) can be reduced. Uncontrolled outlets are also provided that are not affected by the control device and are always providing power to any device plugged into it. This measure characterization provides savings for use of an APS in a home entertainment system, home office, or unknown setting.

This measure was developed to be applicable to the following program types: TOS, NC, DI, and KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a 4-8 plug Tier 1 master-controlled APS.

DEFINITION OF BASELINE EQUIPMENT

For TOS and NC applications, the baseline is a standard power strip that does not control connected loads.
For DI and KITS, the baseline is the existing equipment used in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the Tier 1 APS is 10 years.[86]

DEEMED MEASURE COST

For TOS and NC, the incremental cost of an APS over a standard power strip with surge protection is assumed to be $20.[87]
For DI and KITS, the actual full installation cost of an APS (including equipment and labor) should be used.

LOADSHAPE

Miscellaneous RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (kWh_{Office} * Weighting_{Office} + kWh_{Ent} * Weighting_{Ent}) * ISR$$

[86] "Advanced Power Strip Research Report," NYSERDA, August 2011.
[87] Incremental cost based on "Advanced Power Strip Research Report." Typical cost of an advanced power strip is $35, and average cost of a standard power strip is $15.

Where:

kWh_{Office} = Estimated energy savings from using an APS in a home office
= 31.0 kWh[88]

$Weighting_{Office}$ = Relative penetration of use in home office

Installation Location	$Weighting_{Office}$
Home Office	100%
Home Entertainment System	0%
Unknown[89]	TOS, NC, DI: 36% KITS: 48%

kWh_{Ent} = Estimated energy savings from using an APS in a home entertainment system
= 75.1 kWh[90]

$Weighting_{Ent}$ = Relative penetration of use with home entertainment systems

Installation Location	$Weighting_{Ent}$
Home Office	0%
Home Entertainment System	100%
Unknown[91]	TOS, NC, DI: 64% KITS: 52%

ISR = In service rate, dependent on program type

Program Type	ISR
TOS, NC, DI[92]	95%
KITS[93]	93.8%
Pay As You Save[94]	74.3%

Based on the default values above, default savings are provided in the table below:

Installation Location	Program Type	ΔkWh
Home Office	TOS, NC, DI	29.45
	KITS	29.08
Home Entertainment System	TOS, NC, DI	71.35
	KITS	70.44
Unknown	TOS, NC, DI	56.26
	KITS	50.59

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Electric energy savings, as calculated above.

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001148238[95]

[88] "Advanced Power Strip Research Report." Note that estimates are not based on pre/post metering but on analysis based on frequency and consumption of likely products in active, standby, and off modes. This measure should be reviewed frequently to ensure that assumptions continue to be appropriate.
[89] Relative weightings of home office and entertainment systems is based on "Ameren Missouri Efficient Product Impact and Process Evaluation: Program Year 2015," Cadmus, May 13, 2016. If the programs have their own evaluations of weightings, they should be used.
[90] "Advanced Power Strip Research Report."
[91] Relative weightings of home office and entertainment systems is based on "Ameren Missouri Efficient Product Impact and Process Evaluation: Program Year 2015," Cadmus, May 13, 2016. If the programs have their own evaluations of weightings, they should be used.
[92] Ameren Missouri Single Family Low Income Evaluation: PY2019, Table 10-10.
[93] Ameren Missouri Efficient Products Evaluation: PY2019, Table 6-9.
[94] Ameren Missouri Pay As You Save (PAYS®) Evaluation: PY2022 Participant Survey.
[95] Based on Ameren Missouri 2016 loadshape for residential miscellaneous end-use. This is deemed appropriate, because savings occur during hours when the controlled standby loads are turned off by the APS. This is estimated to be approximately 7,129, which representing the average of hours for controlled TV and computer from "Advanced Power Strip Research Report."

NATURAL GAS SAVINGS
N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

3.2.2 Tier 2 Advanced Power Strip – Residential Audio Visual

DESCRIPTION

This measure applies to the installation of a Tier 2 Advanced Power Strip for household audio visual environments (Tier 2 AV APS). Tier 2 AV APS are multi-plug power strips that remove power from audio visual equipment through intelligent control and monitoring strategies. Using advanced control strategies such as true RMS (Root Mean Square) power sensing, and/or external sensors,[96] both active power loads and standby power loads of controlled devices are managed by Tier 2 AV APS devices. Monitoring and controlling both active and standby power loads of controlled devices will reduce the overall load of a centralized group of electrical equipment (i.e. the home entertainment center). This intelligent sensing and control process has been demonstrated to deliver increased energy savings and demand reduction compared with Tier 1 Advanced Power Strips.

The Tier 2 AV APS market is a relatively new and developing one. With several new Tier 2 AV APS products coming to market, it is important that energy savings be clearly demonstrated through independent field trials. Field trial should effectively address the inherent variability in AV system usage patterns. Until there is enough independent evidence to demonstrate deemed savings for each of the various control strategies, it is recommended that products with independent field trial results be placed into performance bands and savings claimed accordingly.

This measure was developed to be applicable to the following program type: DI. If applied to other program types, the installation characteristics, including the number of AV devices under control and an appropriate in-service rate, should be verified through evaluation.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a Tier 2 AV APS in a residential AV (home entertainment) environment that includes control of at least 2 AV devices, one being the television.[97]

DEFINITION OF BASELINE EQUIPMENT

The assumed baseline equipment is the existing equipment used in the home (e.g., a standard power strip or wall socket) that does not control loads of connected AV equipment.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the Tier 2 AV APS is assumed to be 10 years.[98]

DEEMED MEASURE COST

The actual full installation cost of the Tier 2 AV APS (including equipment and labor) should be used. The estimated incremental cost is $30 based on online market research in 2019. Products installed through Direct Installation channels may also incur additional labor costs.

LOADSHAPE

Miscellaneous RES

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ERP * BaselineEnergy_{AV} * ISR$$

Where:

ERP = Energy reduction percentage of qualifying Tier 2 AV APS product Class; see table below:[99]

[96] Tier 2 AV APS identify when people are not engaged with their AV equipment and then remove power (e.g., a TV and its peripheral devices that are unintentionally left on when a person leaves the house or falls asleep while watching television).

[97] Given this requirement, an AV environment consisting of a TV and DVD player or a TV and home theater would be eligible for a Tier 2 AV APS installation.

[98] "Advanced Power Strip Research Report," NYSERDA, August 2011.

[99] Based on field test data for various APS products.

Product Class	Field Trial ERP Range	ERP Used
A	55 – 60%	55%
B	50 – 54%	50%
C	45 – 49%	45%
D	40 – 44%	40%
E	35 – 39%	35%
F	30 – 34%	30%
G	25 – 29%	25%
H	20 – 24%	20%
Average[100]	-	37.5%

$BaselineEnergy_{AV}$ = 432 kWh[101]

ISR = In Service Rate, the percentage of units rebated that are actually in service

Program/Channel	In Service Rate (ISR)
TOS, NC, DI[102]	95%
Efficient Kits[103]	93.8%
SF Low Income Kits[104]	93.8%

Based on the default values above, default savings are provided in the table below:

Program Type	ΔkWh
TOS, NC, DI	153.90
Efficient Kits	151.96
SF Low Income Kits	151.96

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Electric energy savings, calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001148238[105]

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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[100] Average of product classes B and G.
[101] "Energy Savings of Tier 2 Advanced Power Strips in Residential AV Systems," AESC, Inc., February 2016. Note this load represents the average *controlled* AV devices only and will likely be lower than total AV usage.
[102] Ameren Missouri Single Family Low Income Program Evaluation: PY2019, Table 10-10.
[103] Ameren Missouri Efficient Products Program Evaluation: PY2019, Table 6-9.
[104] Assume same as Efficient Kits.
[105] Based on Ameren Missouri 2016 loadshape for residential miscellaneous end-use. This is deemed appropriate, as savings occur during hours which the controlled standby loads are turned off by the APS, estimated to be approximately 7,129 representing the average of hours for controlled TV and computer from "Advanced Power Strip Research Report."

3.3 Hot Water

3.3.1 Low Flow Faucet Aerator

This measure relates to the installation of a low flow faucet aerator in a household kitchen or bath faucet fixture.

This measure may be used for units provided through efficiency kits. However, the in-service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI, and KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow faucet aerator for bathrooms rated at 1.5 gallons per minute (GPM) or less or for kitchens rated at 2.2 GPM or less. Savings are calculated on an average savings per faucet fixture basis.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard bathroom faucet aerator rated at 2.25 GPM or greater or a standard kitchen faucet aerator rated at 2.75 GPM or greater. Average measured flow rates are used in the algorithm and are lower, reflecting the penetration of previously installed low flow fixtures (and therefore the freerider rate for this measure should be 0), use of the faucet at less than full flow, debris buildup, and lower water system pressure than fixtures are rated at.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.[106]

DEEMED MEASURE COST

The incremental cost for this measure is $11.33[107] or program actual.

For faucet aerators provided in efficiency kits, the actual program delivery costs should be utilized. Absent of program data, use $3.00[108]

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Note these savings are *per* faucet retrofitted[109] (unless faucet type is unknown, then it is per household).

$$\Delta kWh = \%ElectricDHW * (GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * 365.25 * DF / FPH * EPG\_electric * ISR$$

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%
Natural Gas	0%
Unknown	42%[110]

[106] Measure lifetime is derived from the California DEER Effective Useful Life Table – 2014 Table Update. http://www.deeresources.com/files/DEER2013codeUpdate/download/DEER2014-EUL-table-update_2014-02-05.xlsx

[107] Direct-install price per showerhead assumes cost of showerhead (market research average of $3 and assess and install cost of $8.33) and also assumes 20min at $25 per hour, which is in line with the typical prevailing wage of a General Laborer, as per the Annual Wage Order No. 23 published by the Missouri Department of Labor.

[108] Illinois TRM.

[109] This algorithm calculates the amount of energy saved per aerator by determining the fraction of water consumption savings for the upgraded fixture.

[110] Ameren Missouri Energy Efficient Kits Impact and Process Evaluation: PY2019.

GPM_{base} = Average flow rate, in gallons per minute, of the baseline faucet "as-used." This includes the effect of existing low flow fixtures and therefore the freerider rate for this measure should be 0.
= 2.2[111] or custom based on metering studies[112] or if measured during DI:
= Measured full throttle flow * 0.83 throttling factor[113]

GPM_{low} = Average flow rate, in gallons per minute, of the low-flow faucet aerator "as-used"
= 1.5[114] or custom based on metering studies[115] or if measured during DI:
= Rated full throttle flow * 0.95 throttling factor[116]

L_{base} = Average baseline daily length faucet use per capita for faucet of interest in minutes
= if available custom based on metering studies, if not use:

Faucet Type	L_{base} (min/person/day)	
	Kitchen	Bathroom
Efficient Kits (School Kits, MF, ARP Kits)	4.5[117]	1.6[118]
Income Eligible; MFMR, Efficient Kits (SF LI Kits)[119]	3.7	3.7
If location unknown (total for household): Single-Family	7.8[120]	
If location unknown (total for household): Multi-Family	6.7[121]	

L_{low} = Average retrofit daily length faucet use per capita for faucet of interest in minutes
= if available custom based on metering studies, if not use:

Faucet Type	L_{low} (min/person/day)	
	Kitchen	Bathroom
Efficient Kits (School Kits, ARP Kits)	4.5[122]	1.6[122]
Efficient Kits (Multifamily, SFLI Kits); MFMR[123]	3.7	3.7
Income Eligible Common Area[124]	N/A	1.5
If location unknown (total for household): Single-Family	7.8[125]	
If location unknown (total for household): Multi-Family	6.7[126]	

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[111] Federal rated maximum flow rate for faucets (10 CFR 430.32 (p) (DOE 1998).
[112] Measurement should be based on actual average flow consumed over a period of time rather than a one-time spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior, which does not always use maximum flow.
[113] 2008, Schultdt, Marc, and Debra Tachibana, "Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes," 2008 ACEEE Summer Study on Energy Efficiency in Buildings, pp. 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf
[114] Program data, including PY2016 Program Data, per Community Saves 2016 EM&V report.
[115] Measurement should be based on actual average flow consumed over a period of time rather than a one-time spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior, which does not always use maximum flow.
[116] 2008, Schultdt, Marc, and Debra Tachibana, "Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes," 2008 ACEEE Summer Study on Energy Efficiency in Buildings, pp. 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf
[117] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum, dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multifamily homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.
[118] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multifamily homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.
[119] Cadmus PY2003 metering study. Cited in Ameren Missouri Low Income and Process Evaluation: program Year 2015. p.23
[120] One kitchen faucet plus 2.04 bathroom faucets. Based on findings from a 2012 Ameren Missouri potential study for single family homes.
[121] One kitchen faucet plus 1.4 bathroom faucets. Based on findings from Ameren Missouri PY2013 data for multifamily homes.
[122] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.
[123] Cadmus PY2003 metering study. Cited in Ameren Missouri Low Income and Process Evaluation: program Year 2015. p.23
[124] PY2016 Program Data, per Community Saves 2016 EM&V report.
[125]One kitchen faucet plus 1.4 bathroom faucets. Based on findings from an Ameren Missouri PY2013 data for multifamily homes.
[126] One kitchen faucet plus 1.4 bathroom faucets. Based on findings from an Ameren Missouri PY2013 data for multifamily homes.

Household = Average number of people per household

Program Delivery and Household Unit Type	Value
Single-Family	2.67[127]
School Kits	4.286[128]
Efficient Kits (MF)	1.777[129]
Multi-Family MR - Deemed	1.56[130]
Income Eligible, Efficient Kits (SFLI Kits)	1.564[131]
ARP Kits	2.65[132]
Custom	Actual Occupancy or Number of Bedrooms[133]

365.25 = Days in a year, on average.

DF = Drain Factor

Program Delivery	Drain Factor	
	Kitchen	Bath
Non SFLI Kits[134]	75%	90%
Income Eligible, MFMR; SFLI Kits[135]	100%	100%
Unknown	79.5%	N/A

FPH = Faucets Per Household

Program Delivery	FPH	
	Kitchen (KFPH)	Bathroom (BFPH)
Single-Family	1.19[136]	2.04[137]
School Kits	1.19[138]	2.28[139]
Efficient Kits (MF)	1.00[140]	1.337[141]
Multi-Family (MFMR)	1.00[142]	1.86[143]
Income Eligible, Efficient Kits (SFLI Kits)	1.00	1.86[144]
If location unknown (total for household): Single-Family	3.04	
If location unknown (total for household): Multi-Family	2.4	

EPG_electric = Energy per gallon of water used by faucet supplied by electric water heater
= (8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_electric * 3412)
8.33 = Specific weight of water (lbs/gallon)
1.0 = Heat Capacity of water (btu/lb-°F)
WaterTemp = Assumed temperature of mixed water

[127] Ameren Missouri Efficient Products Impact and Process Evaluation: Planning Year 2015, provided by Cadmus.
[128] Ameren Missouri Energy Efficient Kits Program Impact and Process Evaluation: PY2019.
[129] PY2018 Energy Efficiency Kits Property Manager Survey results (I1-I2)
[130] Ameren Missouri Community Savers Evaluation: PY2018.
[131] PY2006 program data (not reported in PY2016). Ameren Missouri Low Income and Process Evaluation: program Year 2015. p.23
[132] Ameren Missouri Appliance Recycling Program Evaluation: PY2019
[133] Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.
[134] Because faucet usages are at times dictated by volume (e.g., filling a cooking pot), only usage of the sort that would go straight down the drain will provide savings. VEIC is unaware of any metering study that has determined this specific factor and so recommends these values to be 75% for the kitchen and 90% for the bathroom. If the aerator location is unknown, an average of 79.5% should be used, which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom (0.7*0.75)+(0.3*0.9)=0.795.
[135] Ameren Missouri Community Savers Evaluation PY2018
[136] Ameren Missouri Energy Efficient Kits Evaluation: PY2018.
[137] Based on findings from a 2012 Ameren Missouri potential study for single family homes.
[138] Ameren Missouri Energy Efficient Kits Evaluation: PY2018.
[139] Ameren Missouri Energy Efficient Kits Program Impact and Process Evaluation: PY2018.
[140] Ameren Missouri EE Kits PY2018 Program Data
[141] Ameren Missouri Community Savers Evaluation: PY2018
[142] Ameren Missouri EE Kits PY2018 Program Data
[143] Ameren Missouri Community Savers Evaluation: PY2018
[144] Ameren Missouri Community Savers Evaluation: PY2018

= 86F for Bathroom (80F for Income Eligible and MFMR), 93F for Kitchen, 91F for Unknown[145]

SupplyTemp = Assumed temperature of water entering house

= 61.3F[146]

RE_electric = Recovery efficiency of electric water heater

= 98%[147]

3,412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of faucet aerators dependant on install method as listed in table below

Selection	In-Service Rate	
	Kitchen	Bathroom
Direct Install, Efficiency Kit—Low Income[148]	89%	89%
Efficiency Kit (School)—Single Family[149]	40%	48%
Efficiency Kit—Appliance Recycling[150]	20%	24%
Efficiency Kit (School)—Multi Family[151]	100%	100%
Income Eligible, Direct Install (Income Eligible and MFMR)[152]	95%	95%
Income Eligible, Non-Direct Install[153]	40%	48%
Income Eligible, Common Area	N/A	97.7%
Pay As You Save[154]	80.9%	80.9%

Summer Coincident Peak Demand Savings

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0.0000887318[155]

Natural Gas Savings

$\Delta Therms = \%GasDHW * ((GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * 365.25 * DF / FPH) * EPG\_gas * ISR$

Where:

%GasDHW = proportion of water heating supplied by Natural Gas heating

DHW fuel	%GasHW
Electric	0%
Natural Gas	100%
Unknown	48%[156]

EPG_gas = Energy per gallon of Hot water supplied by gas

= (8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_gas * 100,000)

RE_gas = Recovery efficiency of gas water heater

[145] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum, dated June 2013, directed to Michigan Evaluation Working Group. If the aerator location is unknown, an average of 91% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom (0.7*93)+(0.3*86)=0.91.

[146] Ameren Missouri 2012 Technical Resource Manual. Appendix A. pp. 43. Available online: https://www.efis.psc.mo.gov/mpsc/commoncomponents/viewdocument.asp?DocId=935658483.

[147] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx.

[148] Ameren Missouri Single Family Low Income Evaluation PY2019 (Table 10-10).

[149] Ameren Missouri Efficient Kits Impact and Process Evaluation: PY2019.

[150] Ameren Missouri Appliance Recycling Evaluation: PY2019.

[151] Ameren Missouri Efficient Products Impact and Process Evaluation: Program Year 2015.

[152] Ameren Missouri Community Savers Evaluation PY2018

[153] Ameren Missouri Efficient Kits Impact and Process Evaluation: PY2019.

[154] Ameren Missouri Pay As You Save (PAYS®) Evaluation: PY2022 Participant Survey

[155] Based on Ameren Missouri 2016 loadshape for residential water heating end-use.

[156] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then that should be used.

= 78% For SF homes[157]
= 67% For MF homes[158]
100,000 = Converts Btus to therms (btu/therm)
Other variables as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta gallons = ((GPM\_base * L\_base - GPM\_low * L\_low) * Household * 365.25 * DF / FPH) * ISR$$

Variables as defined above.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[157] DOE final rule discusses recovery efficiency with an average around 0.76 for gas- fired storage water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.

[158] Water heating in multifamily buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multifamily buildings.

3.3.2 Low Flow Showerhead

DESCRIPTION

This measure relates to the installation of a low flow showerhead in a single or multifamily household.

This measure may be used for units provided through efficiency kit's. However, the in-service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, RF, NC, DI, and KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow showerhead, typically rated at 2.0 gallons per minute (GPM) or less. Savings are calculated on a per showerhead fixture basis.

DEFINITION OF BASELINE EQUIPMENT

For DI programs, the baseline condition is assumed to be a standard showerhead rated at 2.5 GPM[159] or greater.
For RF and TOS programs, the baseline condition is assumed to be a representative average of existing showerhead flow rates of participating customers including a range of low flow showerheads, standard-flow showerheads, and high-flow showerheads.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.[160]

DEEMED MEASURE COST

The incremental cost for TOS, NC, or KITS is $7[161] for standard showerheads and $15.02 for handheld showerheads or program actual.

For low flow showerheads provided in RF or DI programs, the actual program delivery costs should be utilized; if unknown assume $15.33[162] for standard showerheads and $23.35 for handheld showerheads.

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Note these savings are per showerhead fixture

$$\Delta kWh = \%ElectricDHW * ((GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * SPCD * 365.25 / SPH) * EPG\_electric * ISR$$

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%
Natural Gas	0%
Unknown	42%[163]

[159] Maximum showerhead flow rate at 80 PSI is 2.5 GPM in accordance with federal standard 10 CFR Part 430.32(p). See docket filed at https://www.regulations.gov/document?D=EERE-2011-BT-TP-0061-0039
[160] Table C-6, "Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures," GDS Associates, June 2007. Evaluations indicate that consumer dissatisfaction may lead to reductions in persistence, particularly in Multifamily, http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf.
[161] Based on online pricing market research 2/6/2017.
[162] Direct-install price per showerhead assumes cost of showerhead (market research average of $7) and also assumes assess and install cost of $8.33 (20min at $25 per hour, which is in line with the typical prevailing wage of a General Laborer, as per the Annual Wage Order No. 23 published by the Missouri Department of Labor).
[163] Ameren Missouri Energy Efficient Kits Impact and Process Evaluation: PY2019.

GPM_{base} = Flow rate of the baseline showerhead

Program Delivery	GPM_base
Direct-install, SFLI Kits	2.2[164]
Retrofit, Efficiency Kits, NC or TOS	2.35[165]
MFMR	2.5[166]

GPM_{low} = As-used flow rate of the lowflow showerhead, which may, as a result of measurements of program evaulations deviate from rated flows, see table below:

Rated Flow
2.0 GPM
1.75 GPM
1.5 GPM
Custom or Actual[167]

L_{base} = Shower length in minutes with baseline showerhead
= 7.8 min[168] and 8.66 for Income Eligible, MFMR, SFIE Kits[169]

L_{low} = Shower length in minutes with low-flow showerhead
= 7.8 min[170] and 8.66 for Income Eligible, MFMR, SFIE Kits[171]

Household = Average number of people per household

Program Delivery	Household
Single-Family, Income Eligible (SFIE Kits)	2.67[172]
School Kits	4.29[173]
Efficient Kits (MF)	1.777[174]
Income Eligible Multi-Family	1.52[175]
Appliance Recycling Kits	2.65[176]
MFMR	2.07[177]
Custom	Actual Occupancy or Number of Bedrooms[178]

[164] Ameren Missouri Community Savers Evaluation: PY2018.
[165] Representative value from sources 1, 2, 4, 5, 6, and 7 (See Source Table at end of measure section) adjusted slightly upward to account for program participation, which is expected to target customers with existing higher flow devices rather than those with existing low flow devices.
[166] PY2019 Program Data
[167] Note that actual values may be either: a) program-specific minimum flow rate, or b) program-specific evaluation-based value of actual effective flow-rate due to increased duration or temperatures. The latter increases in likelihood as the rated flow drops and may become significant at or below rated flows of 1.5 GPM. The impact can be viewed as the inverse of the throttling described in the footnote for baseline flowrate.
[168] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum, dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multifamily homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.
[169] DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). "California SingleFamily Water Use Efficiency Study."
[170] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multifamily homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.
[171] DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). "California SingleFamily Water Use Efficiency Study."
[172] Ameren Missouri Efficient Products Impact and Process Evaluation: Planning Year 2015, provided by Cadmus.
[173] Ameren Missouri Energy Efficient Kits Evaluation: PY2019.
[174] PY2018 Energy Efficiency Kits Property Manager Survey results (I1-I2)
[175] Ameren Missouri Community Savers Evaluation: PY2018.
[176] Ameren Missouri Appliance Recycling Evaluation PY2019 (Appendix Table 55)
[177] Matches Community Savers EM&V
[178] Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

SPCD = Showers Per Capita Per Day
= 0.832[179] and 0.66 for Incomem Eligible, MFMR, SFIE Kits[180]
365.25 = Days per year, on average.
SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined

Program Delivery	SPH
Single-Family, Income Eligible (SFIE Kits)	2.05[181]
School Kits	2.14[182]
Efficient Kits (MF)	1.34[183]
Income Eligible Multi-Family	1.0[184]
MFMR	1.4[185]
Custom	Actual

EPG_electric = Energy per gallon of hot water supplied by electric
= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_electric * 3412)
= (8.33 * 1.0 * (101 – 60.83)) / (0.98 * 3412)
= 0.100 kWh/gal
8.33 = Specific weight of water (lbs/gallon)
1.0 = Heat capacity of water (btu/lb-°)
ShowerTemp = Assumed temperature of water
= 105.0 F[186]
SupplyTemp = Assumed temperature of water entering house
= 61.3 F[187]
RE_electric = Recovery efficiency of electric water heater
= 98%[188]
3,412 = Converts Btu to kWh (btu/kWh)
ISR = In service rate of showerhead
= Dependant on program delivery method as listed in table below:

Program Delivery	ISR
Direct Install[189]	100%
Efficiency Kit—School (Single Family)[190]	54%
Efficiency Kit—Multifamily[191]	100%
Efficiency Kit—Appliance Recycling[192]	24%
Income Eligible (Single Family Direct Install) [193]	94%
Income Eligible (Multifamily Direct Install), MFMR[194]	96.4%
Income Eligible (Non-Direct Install), SFLI Kits[195]	91.3%
Pay As You Save[196]	65%

179 Ameren Missouri Energy Efficient Kits Program Impact and Process Evaluation: PY2019
180 DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). "California SingleFamily Water Use Efficiency Study."
181 Ameren Missouri Efficient Products Impact and Process Evaluation: Planning Year 2015, provided by Cadmus.
182 Ameren Missouri Energy Efficient Kits Program Impact and Process Evaluation: PY2019.
183 Ameren Missouri PY2018 EE Kits Evaluation
184 Ameren Missouri Community Savers Evaluation: PY2017
185 Matches Community Savers EM&V
186 Ameren Missouri Efficient Kits Evaluation: PY2018.
187 Based on the DOE's Building America Standard DHW Event Schedule calculator. Average annual water main temperatures were determined for each defined weather zone in Missouri. The overall average of 60.83 is taken to represent the statewide average input water temperature.
188 Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx.
189 Ameren Missouri Community Savers Tenant Surveys and Site Visits PY2017
190 Ameren Missouri Efficient Kits Impact and Process Evaluation: PY2019, Table 7-10.
191 Ameren Missouri PY2018 EE Kits Evaluation.
192 Ameren Missouri Appliance Recycling Evaluation: PY2019, Table 9-10.
193 Ameren Missouri Single Family Low Income Evaluation PY2019 (Table 10-10)
194 Ameren Missouri Community Savers Evaluation PY2018 Tenant Surveys and Site Visits.
195 PY2007 Tenant surveys.
196 Ameren Missouri Pay As You Save (PAYS®) Evaluation: PY2022 Participant Survey.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318[197]

NATURAL GAS SAVINGS

$$\Delta Therms = \%GasDHW * ((GPM\_base * L\_base - GPM\_low * L\_low) * Household * SPCD * 365.25 / SPH) * EPG\_gas * ISR$$

Where:

%GasDHW = proportion of water heating supplied by natural gas heating

DHW fuel	%GasDHW
Electric	0%
Natural Gas	100%
Unknown	48%[198]

EPG_gas = Energy per gallon of Hot water supplied by gas
= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_gas * 100,000)
= 0.00429 therm/gal for SF homes
= 0.00499 therm/gal for MF homes

RE_gas = Recovery efficiency of gas water heater
= 78% For SF homes[199]
= 67% For MF homes[200]

100,000 = Converts Btus to therms (btu/Therm)

Other variables as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\boldsymbol{\Delta gallons = ((GPM\_base * L\_base - GPM\_low * L\_low) * Household * SPCD * 365.25 / SPH) * ISR}$$

Variables as defined above

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

---

[197] Based on Ameren Missouri 2016 loadshape for residential water heating end-use.

[198] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then that should be used.

[199] DOE final rule discusses recovery efficiency with an average around 0.76 for gas-fired storage water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas-fired condensing tankless water heaters. However, these numbers represent the range of new units, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.

[200] Water heating in multifamily buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multifamily buildings.

3.3.3 Water Heater Wrap

DESCRIPTION

This measure applies to a tank wrap or insulation "blanket" that is wrapped around the outside of an electric or gas domestic hot water (DHW) tank to reduce stand-by losses.

This measure was developed to be applicable to the following program types: DI, and RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is an electric or gas DHW tank with wrap installed that has an R-value that meets program requirements.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is an uninsulated electric or gas DHW tank.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 12 years.[201]

DEEMED MEASURE COST

The measure cost is the actual cost of material and installation. If actual costs are unknown, assume $58[202] for material and installation.

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Custom calculation below for electric DHW tanks, otherwise use default values from table that follows:

$$\Delta kWh = ((A_{Base}/R_{Base} - A_{EE}/R_{EE}) * \Delta T * Hours)/(\eta DHW_{Elec} * 3{,}412)$$

Where:

A_{Base} = Surface area (ft^2) of storage tank prior to adding tank wrap[203]
= Actual or if unknown, use default based on tank capacity (gal) from table below

R_{Base} = Thermal resistance coefficient (hr-°F-ft^2/BTU) of uninsulated tank
= Actual or if unknown, assume 14[204]

A_{EE} = Surface area (ft^2) of storage tank after addition of tank wrap[205]
= Actual or, if unknown, use default based on tank capacity (gal) from table below

R_{EE} = Thermal resistance coefficient ((hr-°F-ft2/BTU) of tank after addition of tank wrap (R-value of uninsulated tank + R-value of tank wrap)
= Actual or if unknown, assume 24

ΔT = Average temperature difference (°F) between tank water and outside air
= Actual or if unknown, assume 60°F[206]

[201] 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, "Cost Values and Summary Documentation," California Public Utilities Commission, January 2014. Average of values for electric DHW (13 years) and gas DHW (11 years).

[202] Average cost of R-10 tank wrap installation from the National Renewable Energy Laboratory's National Residential Efficiency Measures Database. http://www.nrel.gov/ap/retrofits/measures.cfm?gId=6&ctId=270.

[203] Area includes tank sides and top to account for typical wrap coverage.

[204] Baseline R-value based on information from Chapter 6 of *The Virginia Energy Savers Handbook*, Third Edition: The best heaters have 2 to 3 inches of urethane foam, providing R-values as high as R-20. Other less expensive models have fiberglass tank insulation with R-values ranging between R-7 and R-10.

[205] Area includes tank sides and top to account for typical wrap coverage.

[206] Assumes 125°F hot water tank temperature and average basement temperature of 65°F.

Hours = Hours per year
= 8,766
ηDHW_{Elec} = Recovery efficiency of electric hot water heater
= Actual or if unknown, assume 0.98[207]
3,412 = Conversion factor from Btu to kWh

The following table contains default savings for various tank capacities.

Capacity (gal)	A_{Base} (ft^2)[208]	A_{EE} (ft^2)[209]	ΔkWh	ΔkW
30	19.16	20.94	78.0	0.00890
40	23.18	25.31	94.6	0.01079
50	24.99	27.06	103.4	0.01180
80	31.84	34.14	134.0	0.01528

Summer Coincident Peak Demand Savings

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Electric energy savings, as calculated above.
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318[210]

The table above contains default kW savings for various tank capacities.

Natural Gas Savings

Custom calculation below for gas DHW tanks, otherwise use default values from table that follows:

$$\Delta Therms = ((A_{Base}/R_{Base} - A_{EE}/R_{EE}) * \Delta T * Hours)/(\eta DHW_{Gas} * 100{,}000)$$

Where:

ηDHW_{Gas} = Recovery efficiency of gas hot water heater
= 0.78[211]
100,000 = Conversion factor from Btu to therms
Other variables as defined above

The following table contains default savings for various tank capacities.

Capacity (gal)	A_{Base} (ft^2)[212]	A_{EE} (ft^2)[213]	ΔTherms	ΔPeakTherms
30	19.16	20.94	3.3	0.0092
40	23.18	25.31	4.1	0.0111
50	24.99	27.06	4.4	0.0121
80	31.84	34.14	5.7	0.0157

Water Impact Descriptions and Calculation

N/A

Deemed O&M Cost Adjustment Calculation

N/A

Measure Code:

---

[207] Electric water heater recovery efficiency from AHRI database: http://www.ahridirectory.org/ahridirectory/pages/home.aspx.
[208] Surface area assumptions from the June 2016 Pennsylvania TRM. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. Area includes tank sides and top to account for typical wrap coverage.
[209] Surface area assumptions from the June 2016 Pennsylvania TRM. A_{EE} was calculated by assuming that the water heater wrap is a 2" thick fiberglass material.
[210] 2016 Ameren Missouri Coincident Peak Demand Factor for Residential Water Heating. See reference "Ameren Missouri 2016 Appendix E - End Use Shapes and Coincident Factors.pdf."
[211] Review of AHRI directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.
[212] Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. Area includes tank sides and top to account for typical wrap coverage. Recommend updating with Missouri-specific data when available.
[213] A_{EE} was calculated by assuming that the water heater wrap is a 2" thick fiberglass material. Recommend updating with Missouri-specific data when available.

3.3.4 Heat Pump Water Heater

DESCRIPTION
This measure applies to the installation of a heat pump water heater (HPWH) in place of a standard electric water heater in a home. Savings are presented dependent on the heating system installed in the home due to the impact of the heat pump water heater on the heating and cooling loads.

This measure was developed to be applicable to the following program types: TOS, and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
To qualify for this measure, the installed equipment must be an ENERGY STAR® heat pump water heater with a storage volume ≤ 55 gallons.[214]

DEFINITION OF BASELINE EQUIPMENT
The baseline equipment is assumed to be a new, electric storage water heater meeting federal minimum efficiency standards[215] for units ≤55 gallons: 0.96 – (0.0003 * rated volume in gallons).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The expected measure life is assumed to be 13 years.[216]

DEEMED MEASURE COST
Actual costs should be used where available. The default value for incremental capital costs is $588.[217]

LOADSHAPE
Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \left[\left(\frac{(1/EF_{BASE} - 1/EF_{EE}) * GPD * Household * 365.25 * \gamma Water * (T_{Out} - T_{In}) * 1.0}{3{,}412}\right) + kWh_{cool} - kWh_{heat}\right] * ISR$$

Where:

EF_{BASE} = EF of standard electric water heater according to federal standards
= 0.96 – (0.0003 * rated volume in gallons)

= If rated volume is unknown, assume 0.945 for a 50-gallon water heater

EF_{EE} = EF of heat pump water heater
= Actual

GPD = Gallons per day of hot water use per person
= 17.6[218]

Household = Average number of people per household

[214] Since the federal standard effectively requires a heat pump water heater for units over 55 gallons, this measure is limited to units ≤ 55 gallons.
[215] Minimum federal standard as of 4/16/2015:
http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.
http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.
[216] 2010 Residential Heating Products Final Rule Technical Support Document, U.S. DOE, Table 8.7.2.
[217] Ameren Missouri MEEIA 2016-18 TRM – January 1, 2018.
[218] GPD based on 45.5 gallons of hot water per day per household and 2.59 people per household, from "Residential End Uses of Water Study 2013 Update," by Deoreo, B., and P. Mayer, for the Water Research Foundation, 2014.

Household Unit Type[219]	Household
Single-Family - Deemed	2.65[220]
Multi-Family - Deemed	2.07[221]
Custom	Actual Occupancy or Number of Bedrooms[222]

365.25 = Days per year
γWater = Specific weight of water
= 8.33 pounds per gallon
T_{OUT} = Tank temperature
= Actual, if unknown assume 125°F
T_{IN} = Incoming water temperature from well or municipal system
= 57.898°F[223]
1.0 = Heat capacity of water (1 Btu/lb*°F)
3,412 = Conversion factor from Btu to kWh
ISR = In Service Rate = 100%[224]
kWh_cool = Cooling savings from conversion of heat in home to water heat[225]

$$= \left[\frac{\left(\left(1 - \frac{1}{EF_{EE}}\right) * GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0 \right) * LF * WHF_C * LM}{COP_{COOL} * 3{,}412} \right] * \%Cool$$

Where:

LF = Location Factor
= 1.0 for HPWH installation in a conditioned space
= 0.0 for installation in an unconditioned space
WHF_C = Portion of reduced waste heat that results in cooling savings (if unknown, assume 53%) [226]
COP_{COOL} = COP of central air conditioner
= Actual, or if unknown, assume 2.8 COP[227]
LM = Latent multiplier to account for latent cooling demand [228]

Weather Basis (City based upon)	LM
St Louis, MO	1.33

[219] If household type is unknown, as may be the case for TOS measures, then single family deemed value shall be used.
[220] Ameren Missouri Efficient Products Evaluation: PY2018.
[221] Ameren Missouri Efficient Products Evaluation: PY2015
[222] Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.
[223] Using 40" deep soil temp as a proxy at Powell Gardens SCAN site. Average by month of available data from 3/28/02–10/11/14: 12-month average is 57.898. http://www.wcc.nrcs.usda.gov/nwcc/site?sitenum=2061.
[224] Ameren Missouri Efficient Products Evaluation: PY2019.
[225] This algorithm calculates the heat removed from the air by subtracting the heat pump water heater electric consumption from the total water heating energy delivered. This is then adjusted to account for location of the heat pump unit and the coincidence of the waste heat with cooling requirements, the efficiency of the central cooling, and latent cooling demands.
[226] Based on Ameren Missouri Efficient Products Evaluation PY2018.
[227] Starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * $SEER^2$) + (1.12 * SEER) (from Wassmer, M. (2003), "A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations," (Masters thesis), University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP.
[228] The Latent Multiplier is used to convert the sensible cooling savings calculated to a value representing sensible and latent cooling loads. The values are derived from the methodology outlined in "Infiltration Factor Calculation Methodology" by Bruce Harley, Senior Manager, Applied Building Science, CLEAResult 11/18/2015, and are based upon an 8760 analysis of sensible and total heat loads using hourly climate data. (Ameren Missouri Efficient Products Evaluation PY2018)

%Cool = Percentage of homes with central cooling

Home	%Cool
Cooling	100%
No Cooling	0%
Unknown	95%[229]

kWh_heat = Heating cost from conversion of heat in home to water heat (dependent on heating fuel)

$$= \left(\frac{\left(\left(1 - \frac{1}{EF_{EE}}\right) * GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0\right) * LF * WHF_H}{COP_{HEAT} * 3{,}412} \right) * \%ElectricHeat$$

Where:

WHF_H = Portion of reduced waste heat that results in increased heating load (if unknown, assume 43%) [230]

COP_{HEAT} = COP of electric heating system

= Actual, or if unknown, assume:[231]

System Type	Age of Equipment	Heating Seasonal Performance Factor (HSPF) Estimate	COP (Effective COP Estimate) (HSPF/3.412)* 0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 and after	8.2	2.04
Resistance	N/A	N/A	1

%ElectricHeat = Percentage of home with electric heat

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	35%[232]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = kWh * CF$$

Where:

kWh = Electric energy savings, as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0.0000887318[233]

[229] Ameren Missouri PY2019 Residential Baseline Study (Saturation of non-low income homes with central cooling).
[230] Based on Ameren Missouri Efficient Products Evaluation PY2018.
[231] These default system efficiencies are based on the applicable minimum federal standards. In 2006, the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.
[232] Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.
[233] Based on Ameren Missouri 2016 loadshape for residential water heating end-use.

NATURAL GAS SAVINGS

$$\Delta Therms = -\left(\frac{\left(\left(1-\frac{1}{EF_{EE}}\right)*GPD*Household*365.25*\gamma Water*(T_{OUT}-T_{IN})*1.0\right)*LF*43\%}{\eta Heat*100{,}000}\right)*\%GasHeat$$

Where:

ΔTherms = Heating cost from conversion of heat in home to water heat for homes with Natural Gas heat[234]

100,000 = Conversion factor from Btu to therms

ηHeat = Efficiency of heating system

= 71%[235]

%GasHeat = Percentage of homes with gas heat

Heating Fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65%[236]

Other factors as defined above

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[234] This is the additional energy consumption required to replace the heat removed from the home during the heating season by the heat pump water heater. The variable kWh_heating (electric resistance) is that additional heating energy for a home with electric resistance heat (COP 1.0). This formula converts the additional heating kWh for an electric resistance home to the MMBtu required in a natural gas heated home, applying the relative efficiencies.

[235] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference "HC6.9 Space Heating in Midwest Region.xls." In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

[236] Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls."

3.3.5 Hot Water Pipe Insulation

DESCRIPTION
This measure applies to the addition of insulation to uninsulated domestic hot water (DHW) pipes. The measure assumes the pipe wrap is installed on the first length of both the hot and cold pipe up to the first elbow. This is the most cost-effective section to insulate since the water pipes act as an extension of the hot water tank up to the first elbow, which acts as a heat trap. Insulating this section helps to reduce standby losses.

This measure was developed to be applicable to the following program types: DI, and RF

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
The efficient condition is a domestic hot or cold water pipe with pipe wrap installed that has an R value that meets program requirements.

DEFINITION OF BASELINE EQUIPMENT
The baseline condition is an uninsulated, domestic hot or cold water pipe.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The expected measure life is assumed to be 12 years.[237]

DEEMED MEASURE COST
The measure cost is the actual cost of material and installation. If the actual cost is unknown, assume a default cost of $7.10 [238] per linear foot, including material and installation. For a kit program, assume a default cost of $2.87.[239]

LOADSHAPE
Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS
Custom calculation below for electric systems, otherwise assume 24.7 kWh per 6 linear feet of ¾ in, R-4 insulation or 35.4 kWh per 6 linear feet of 1 in, R-6 insulation:

$$\Delta kWh = \%ElectricDHW * ((C_{Base}/R_{Base} - C_{EE}/R_{EE}) * L * \Delta T * Hours)/(\eta DHW_{Elec} * 3{,}412) * ISR$$

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%
Natural Gas	0%
Unknown	42%[240]

C_{Base} = Circumference (ft) of uninsulated pipe
= Diameter (in) * π/12
= Actual or if unknown, assume 0.196 ft for a pipe with a 0.75 inch diameter

R_{Base} = Thermal resistance coefficient (hr-°F-ft^2)/Btu) of uninsulated pipe

[237] 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, "Cost Values and Summary Documentation," California Public Utilities Commission, January 2014. Average of values for electric DHW (13 years) and gas DHW (11 years).
[238] Average cost of R-5 pipe wrap installation from the National Renewable Energy Laboratory's National Residential Efficiency Measures Database. http://www.nrel.gov/ap/retrofits/measures.cfm?gId=6&ctId=323
[239] Cost based on RS Means 2018 data
[240] Ameren Missouri Energy Efficient Kits Impact and Process Evaluation: PY2019.

= 1.0[241]

C_{EE} = Circumference (ft) of insulated pipe
= Diameter (in) * π/12
= Actual or if unknown, assume 0.524 ft for a 0.46 in diameter pipe insulated with 3/4 in, R-4 wrap ((0.75 + 1/2 + 1/2) * π/12)

R_{EE} = Thermal resistance coefficient (hr-°F-ft^2)/Btu) of insulated pipe
= 1.0 + R value of insulation
= Actual or if unknown, assume 5.0 for R-4 wrap or 7.0 for R-6 wrap

L = Length of pipe from water heating source covered by pipe wrap (ft)
= Actual or if unknown, assume 6 ft

ΔT = Average temperature difference (°F) between supplied water and outside air
= Actual or if unknown, assume 60°F[242]

Hours = Hours per year
= 8,766

ηDHW_{Elec} = Recovery efficiency of electric hot water heater
= Actual or if unknown, assume 0.98[243]

3,412 = Conversion factor from Btu to kWh

ISR = Installation rate (varies by program)

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Electric energy savings, as calculated above.

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318

NATURAL GAS SAVINGS

Custom calculation below for gas DHW systems, otherwise assume 1.1 therms per 6 linear feet of ¾ in, R-4 insulation or 1.5 therms per 6 linear feet of 1 in, R-6 insulation:

$$\Delta Therms = ((C_{Base}/R_{Base} - C_{EE}/R_{EE}) * L * \Delta T * Hours)/(\eta DHW_{Gas} * 100{,}000)$$

Where:

ηDHW_{Gas} = Recovery efficiency of gas hot water heater
= 0.78[244]

100,000 = Conversion factor from Btu to therms

Other variables as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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[241] "Measures and Assumptions for Demand Side Management (DSM) Planning; Appendix C Substantiation Sheets," Navigant, April 2009.
[242] Assumes 125°F water leaving the hot water tank and average basement temperature of 65°F.
[243] Electric water heater recovery efficiency from AHRI database: http://www.ahridirectory.org/ahridirectory/pages/home.aspx.
[244] Review of AHRI directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.

3.3.6 Thermostatic Restrictor Shower Valve

DESCRIPTION

The measure is the installation of a thermostatic restrictor shower valve in a single or multifamily household. This is a valve attached to a residential showerhead which restricts hot water flow through the showerhead once the water reaches a set point (generally 95F or lower).

This measure was developed to be applicable to the following program types: RF, NC, and DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a thermostatic restrictor shower valve installed on a residential showerhead.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is the residential showerhead without the restrictor valve installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years. [245]

DEEMED MEASURE COST

The incremental cost of the measure should be the actual program cost (including labor if applicable) or $30[246] plus $20 labor[247] if not available.

LOADSHAPE

Water Heating RES

COINCIDENCE FACTOR

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \%ElectricDHW * ((GPM\_base\_S * L\_showerdevice) * Household * SPCD * 365.25 / SPH) * EPG\_electric * ISR$$

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%
Natural Gas	0%
Unknown	16%[248]

[245] Assumptions based on NY TRM, Pacific Gas and Electric Company Work Paper PGECODHW113 and measure life of lowflow showerhead.
[246] Based on actual cost of the SS-1002CP-SB Ladybug Water-Saving Shower-Head adapter from Evolve showerheads.
[247] Estimate for contractor installation time.
[248] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then that should be used.

GPM_base_S = Flow rate of the base case showerhead, or actual if available

Program	GPM
Direct-install, device only	1.5[249]
New Construction or direct install of device and low flow showerhead	Rated or actual flow of program-installed showerhead
Retrofit or TOS	2.35[250]

L_showerdevice = Hot water waste time avoided due to thermostatic restrictor valve
= 0.89 minutes[251]
Household = Average number of people per household

Household Unit Type[252]	Household
Single-Family - Deemed	2.67[253]
Multi-Family - Deemed	2.07[254]
Custom	Actual Occupancy or Number of Bedrooms[255]

SPCD = Showers Per Capita Per Day
= 0.66[256]
365.25 = Days per year, on average.
SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined

Household Type	SPH
Single-Family	2.05[257]
Multi-Family	1.4[258]
Custom	Actual

EPG_electric = Energy per gallon of hot water supplied by electric
= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_electric * 3,412)
= (8.33 * 1.0 * (105 – 61.3)) / (0.98 * 3,412)
= 0.109 kWh/gal
8.33 = Specific weight of water (lbs/gallon)
1.0 = Heat capacity of water (btu/lb-°)
ShowerTemp = Assumed temperature of water
= 105F[259]
SupplyTemp = Assumed temperature of water entering house
= 61.3F[260]
RE_electric = Recovery efficiency of electric water heater
= 98%[261]

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[249] Illinois Statewide Technical Reference Manual for Energy Efficiency Version 5.0. pp. 184. 2016. http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Version_5/Final/IL-TRM_Version_5.0_dated_February-11-2016_Final_Compiled_Volumes_1-4.pdf. Assumes low flow showerhead is included in direct installation.
[250] Representative value from sources 1, 2, 4, 5, 6 and 7 (See Source Table at end of measure section) adjusted slightly upward to account for program participation which is expected to target customers with existing higher flow devices rather than those with existing low flow devices.
[251] Average of the following sources: ShowerStart LLC survey; "Identifying, Quantifying and Reducing Behavioral Waste in the Shower: Exploring the Savings Potential of ShowerStart" City of San Diego Water Department survey; "Water Conservation Program: ShowerStart Pilot Project White Paper," and PG&E Work Paper PGECODHW113.
[252] If household type is unknown, as may be the case for TOS measures, then single family deemed value should be used.
[253] Missouri TRM 2017 - Low Flow Showerheads 3.3.2.
[254] Missouri TRM 2017 - Low Flow Showerheads 3.3.2.
[255] Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.
[256] DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). "California Single Family Water Use Efficiency Study."
[257] Missouri TRM 2017 - Low Flow Showerheads 3.3.2.
[258] Missouri TRM 2017 - Low Flow Showerheads 3.3.2.
[259] Illinois Statewide Technical Reference Manual for Energy Efficiency Version 5.0. 2016. pp 103. Available Online: http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Version_5/Final/IL-TRM_Version_5.0_dated_February-11-2016_Final_Compiled_Volumes_1-4.pdf.
[260] Ameren Missouri 2012 Technical Resource Manual. Appendix A. pp. 43. https://www.efis.psc.mo.gov/mpsc/commoncomponents/viewdocument.asp?DocId=935658483.
[261] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx.

3412 = Converts Btu to kWh (btu/kWh)
ISR = In service rate of showerhead
= Dependent on program delivery method as listed in table below

Selection	ISR
Direct Install - Single Family	0.91
Direct Install – Multi Family	0.91[262]
Efficiency Kits	To be determined through evaluation

EXAMPLE

For example, a direct installed valve in a single-family home with electric DHW:
ΔkWh = 1.0 * (2.67 * 0.89 * 1.5 * 0.66 * 365.25 / 2.05) * 0.108 * 0.91
= 42 kWh

Summer Coincident Peak Demand Savings

$$\Delta kW = \Delta kWh / Hours * CF$$

Where:
ΔkWh = calculated value above
Hours = Annual electric DHW recovery hours for wasted showerhead use prevented by device
= ((GPM_base_S * L_showerdevice) * Household * SPCD * 365.25) * 0.712[263] / GPH
GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.
= 27.51
= 34.4 for SF direct install; 28.3 for MF direct install
= 30.3 for SF Retrofit and TOS; 24.8 for MF Retrofit and TOS
Water Heating RES

EXAMPLE

For example, a direct installed thermostatic restrictor device in a single family home with electric DHW where the number of showers is not known.
ΔkW = 85.3/34.4 * 0.0022
= 0.0055 kW

Natural Gas Savings

$$\Delta Therms = \%FossilDHW * ((GPM\_base\_S * L\_showerdevice) * Household * SPCD * 365.25 / SPH) * EPG\_gas * ISR$$

Where:
%FossilDHW = proportion of water heating supplied by Natural Gas heating

DHW fuel	%Fossil_DHW
Electric	0%
Natural Gas	100%
Unknown	84%[264]

EPG_gas = Energy per gallon of Hot water supplied by gas
= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_gas * 100,000)

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262 Based on Ameren Missouri Community Savers Evaluation.
263 71.2% is the proportion of hot 120F water mixed with 54.1F supply water to give 101F shower water.
264 Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then that should be used.

= 0.00501 therm/gal for SF homes
= 0.00583 therm/gal for MF homes
RE_gas = Recovery efficiency of gas water heater
= 78% For SF homes[265]
= 67% For MF homes[266]
100,000 = Converts Btus to therms (btu/therm)
Other variables as defined above.

EXAMPLE

For example, a direct installed thermostatic restrictor device in a gas fired DHW single family home where the number of showers is not known:

ΔTherms = 1.0 * ((2.67 * 0.89) * 2.56 * 0.6 * 365.25 / 1.79) * 0.00501 * 0.98
= 3.7 therms

Water Impact Descriptions and Calculation

$$\Delta gallons = ((GPM\_base\_S * L\_showerdevice) * Household * SPCD * 365.25 / SPH) * ISR$$

Variables as defined above

EXAMPLE

For example, a direct installed thermostatic restrictor device in a single family home where the number of showers is not known:

Δgallons = ((2.67 * 0.89) * 2.56 * 0.6 * 365.25 / 1.79) * 0.98
= 730 gallons

Deemed O&M Cost Adjustment Calculation

N/A

[265] DOE final rule discusses recovery efficiency with an average around 0.76 for gas-fired storage water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.

[266] Water heating in multifamily buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multifamily buildings.

Sources

Source ID	Reference
1	2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011.
2	2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000.
3	1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999.
4	2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003.
5	2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011.
6	2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011.
7	2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings.
8	2011, Lutz, Jim. "Water and Energy Wasted During Residential Shower Events: Findings from a Pilot Field Study of Hot Water Distribution Systems," Energy Analysis Department Lawrence Berkeley National Laboratory, September 2011.
9	2008, Water Conservation Program: ShowerStart Pilot Project White Paper, City of San Diego, CA.
10	2012, Pacific Gas and Electric Company, Work Paper PGECODHW113, Low Flow Showerhead and Thermostatic Shower Restriction Valve, Revision # 4, August 2012.
11	2008, "Simply & Cost Effectively Reducing Shower Based Warm-Up Waste: Increasing Convenience & Conservation by Attaching ShowerStart to Existing Showerheads," ShowerStart LLC.
12	2014, New York State Record of Revision to the TRM, Case 07-M-0548, June 19, 2014.

Measure Code:

3.4 HVAC

3.4.1 Advanced Thermostat

Description

This measure characterizes the household energy savings from the installation of a new thermostat(s) for reduced heating and cooling consumption through a configurable schedule of temperature setpoints (like a programmable thermostat) *and* automatic variations to that schedule to better match HVAC system runtimes to meet occupant comfort needs. These schedules may be defaults, established through user interaction, and be changed manually at the device or remotely through a web or mobile app. Automatic variations to that schedule could be driven by local sensors and software algorithms and/or through connectivity to an internet software service. Data triggers to automatic schedule changes might include, for example: occupancy/activity detection, arrival & departure of conditioned spaces, optimization based on historical or population-specific trends, or weather data and forecasts.[267] This class of products and services are relatively new, diverse, and rapidly changing. Generally, the savings expected for this measure aren't yet established at the level of individual features, but rather at the system level and how it performs overall. Like programmable thermostats, it is not suitable to assume that heating and cooling savings follow a similar pattern of usage and savings opportunity, so this measure treats these savings independently. This is a very active area of ongoing study to better map features to savings value and establish standards of performance measurement based on field data so that a standard of efficiency can be developed.[268] That work is not yet complete but does inform the treatment of some aspects of this characterization and recommendations. Energy savings are applicable at the household level; installation of multiple advanced thermostats per home does not accrue additional savings.

This measure was developed to be applicable to the following program types: TOS, NC, RF, and DI.

If applied to other program types, the measure savings should be verified.

Definition of Efficient Equipment

This measure involves replacement of a manual-only or programmable thermostat with one that has the default-enabled capability or the automatic capability to establish a schedule of temperature setpoints according to driving device inputs above and beyond basic time and temperature data of conventional programmable thermostats. As summarized in the description, this category of products and services is broad and rapidly advancing with regard to thermostat capability, usability, and sophistication. At a minimum, a qualifying thermostat must be capable of two-way communication[269] and exceed the typical performance of manual and conventional programmable thermostats through the automatic or default capabilities described above.

Definition of Baseline Equipment

The baseline is either the actual thermostat type (manual or programmable), if known,[270] or an assumed mix of both types based upon information available from evaluations or surveys that represent the population of program participants. This mix may vary by program, but as a default, 44% programmable and 56% manual thermostats may be assumed.[271]

Deemed Lifetime of Efficient Equipment

The expected measure life for advanced thermostats is assumed to be similar to that of a programmable thermostat, 10 years,[272] based upon equipment life only.[273]

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[267] For example, the capabilities of products and added services that use ultrasound, infrared, or geofencing sensor systems, automatically develop individual models of a home's thermal properties through user interaction. The thermostats optimize system operation based on equipment type and performance traits, such as using n weather forecasts, to demonstrate the type of automatic schedule change functionality that apply to this measure characterization.

[268] The ENERGY STAR® program discontinued its support for basic programmable thermostats effective 12/31/09 and is presently developing a new specification for "Residential Climate Controls."

[269] This measure recognizes that field data may be available, through the thermostat's two-way communication capability, to more accurately establish efficiency criteria and make savings calculations. It is recommended that program implementations incorporate this data into their planning and operation activities to improve understanding of the measure to manage risks and enhance savings results.

[270] If the actual thermostat is programmable and is found to be used in override mode or otherwise is effectively being operated like a manual thermostat, then the baseline may be considered to be a manual thermostat.

[271] Value for blend of baseline thermostats comes from an Illinois potential study conducted by ComEd in 2013; Opinion Dynamics Corporation, "ComEd Residential Saturation/End Use, Market Penetration & Behavioral Study," Appendix 3: Detailed Mail Survey Results, April 2013, p. 34.

[272] Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007.

[273] Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a number of savings studies that lasted a single year or less, the longer-term impacts should be assessed.

DEEMED MEASURE COST
For DI and other programs for which installation services are provided, the actual material, labor, and other costs should be used. For retail, Bring Your Own Thermostat (BYOT) programs,[274] or other program types, actual costs are still preferable.[275] If actual costs are unknown, then the average incremental cost for the new installation measure is assumed to be $125.[276]

LOADSHAPE
Cooling RES
Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS
Electrical savings are a function of both heating and cooling energy usage reductions. For heating, this is a function of the percent of electric heat (heat pumps) and fan savings in the case of a natural gas furnace.

$$\Delta kWh = \Delta kWh_{heating} + \Delta kWh_{cooling}$$

$$\Delta kWh_{heating} = \%ElectricHeat * HeatingConsumption_{Electric} * HF * HeatingReduction * Eff\_ISR + (\Delta Therms * Fe * 29.3)$$

$$\Delta kWh_{cool} = \%AC * ((EFLHcool * CapacityCool * 1/SEER)/1000) * CoolingReduction * Eff\_ISR$$

Where:

%ElectricHeat = Percentage of heating savings assumed to be electric

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	33%[277]

$HeatingConsumption_{Electric}$ = Estimate of annual household heating consumption for electrically heated single-family homes.[278]

Weather Basis (Ameren Missouri Average)	Elec_Heating_ Consumption (kWh)[279]		
	Electric Resistance	Electric Heat Pump	Unknown Electric
SF or MF	14,202	8,355	11,456
MFc (comprehensive envelope)	4,832	2,843	3,898

274 In contrast to program designs that utilize program-affiliated contractors or other trade ally partners that support customer participation through thermostat distribution, installation, and other services, BYOT programs enroll customers after the time of purchase through online rebate and program integration sign-ups.
275 Actual costs include any one-time software integration, annual software maintenance, and/or individual device energy feature fees.
276 Market prices vary considerably in this category, generally increasing with thermostat capability and sophistication. The core suite of functions required by this measure's eligibility criteria can be found on units readily available in the market. Prices are in the range of $200 and $250, excluding the availability of any wholesale or volume discounts. The assumed incremental cost is based on the middle of the range ($175) minus a cost of $50 for the baseline equipment blend of manual. Add-on energy service costs, which may include one-time setup and/or annual per device costs, are not included in this assumption.
277 Ameren Missouri Efficient Products Evaluation: PY2020.
278 Ameren Missouri Efficient Products Evaluation PY2018 workpapers. For Comprehensive Envelope (CompE) Measures, the ratio of MF effective full load hours (1496) to the Opinion Dynamic recommendation for Comprehensive Envelope full load hours (509) was used to scale heating consumption values.
279 Ibid.

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65%[280]
Actual	Custom[281]

HeatingReduction = Assumed percentage reduction in total household heating energy consumption due to advanced thermostat

Existing Thermostat Type	Heating_Reduction[282]
Manual	8.8%
Programmable	5.6%
Blended Average	6.67%

Eff_ISR = Effective In-Service Rate, the percentage of thermostats installed and configured effectively for 2-way communication
= If programs are evaluated during program deployment then custom ISR assumptions should be applied. If in service rate is captured within the savings percentage, ISR should be 100%. If using default savings, use 100%.[283]

ΔTherms = Therm savings if natural gas heating system
= See calculation in natural gas section below

F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%[284]

29.3 = kWh per therm

%AC = Fraction of customers with thermostat-controlled air-conditioning

Thermostat control of air conditioning?	%AC
Yes	100%
No	0%
Unknown	Actual population data, or 91%[285]

$EFLH_{cool}$ = Equivalent full load hours of air conditioning:

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$ (Hours)
SF or MF	869[286]
MFc (comprehensive envelope)	632[287]

CapacityCool = Capacity of air cooling system (Btu/hr) (Note: One ton is equal to 12,000 Btu/hr.)
= Actual installed - If actual size unknown, assume 36,000 Btu/h

SEER = the cooling equipment's Seasonal Energy Efficiency Ratio rating (kBtu/kWh)

280 Multifamily household heating consumption relative to singlefamily households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to multifamily homes with electric resistance, based on professional judgment that average household size, and heat loads of multifamily households are smaller than singlefamily homes.

281 Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

282 These values represent adjusted baseline savings values for different existing thermostats, as presented in Navigant's IL TRM Workpaper on Impact Analysis from Preliminary Gas savings findings (page 28). The unknown assumption is calculated by multiplying the savings for manual and programmable thermostats by their respective share of baseline. Further evaluation and regular review of this key assumption is encouraged. Ameren Missouri Efficient Products Evaluation PY2017.

283 As a function of the method for determining savings impact of these devices, in-service rate effects are already incorporated into the savings value for heating reduction above.

284 F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBTU/yr) and Eae (kWh/yr). An average of a 300- record sample (non-random) out of 1495 was 3.14%. This is appropriately ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

285 91% of homes have central cooling in Missouri (based on 2009 Residential Energy Consumption Survey, see "RECS 2009 Air Conditioning_hc7.9.xls").

286 Based on full load hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren Missouri territory, which suggests an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

287 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

= Use actual SEER rating where it is possible to measure or reasonably estimate. If unknown assume 13.[288]

1/1000 = kBtu per Btu

CoolingReduction = Assumed percentage reduction in total household cooling energy consumption due to installation of advanced thermostat

= If programs are evaluated during program deployment then custom savings assumptions should be applied. Otherwise use:

= 8.0%[289]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW \quad = \Delta kWh_{cooling} * CF$$

Where:

$kWh_{cooling}$ = Electric energy savings for cooling, calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0.0009474181[290]

NATURAL GAS ENERGY SAVINGS

$$\Delta Therms = \%FossilHeat * HeatingConusmption_{Gas} * HF * HeatingReduction * Eff\_ISR$$

Where:

%FossilHeat = Percentage of heating savings assumed to be Natural Gas

Heating fuel	%FossilHeat
Electric	0%
Natural Gas	100%
Unknown	67%[291]

$HeatingConsumption_{Gas}$

= Estimate of annual household heating consumption for gas heated single-family homes.[292]

Weather Basis (City based upon)	Gas_Heating_ Consumption (Therms)
St Louis, MO	682

Other variables as provided above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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[288] Based on minimum federal standard: http://www1.eere.energy.gov/buildings/appliance_standards/residential/residential_cac_hp.html.

[289] This assumption is based upon the review of many evaluations from other regions in the United States. Cooling savings are more variable than heating due to significantly more variability in control methods and potential population and product capability.

[290] 2016 Ameren Missouri Coincident Peak Demand Factor for Residential Cooling. See reference "Ameren Missouri 2016 Appendix E - End Use Shapes and Coincident Factors.pdf."

[291] Ameren Missouri Efficient Products Evaluation: PY2020.

[292] Values in table are based on average household heating load (834 therms) for Chicago based on Illinois furnace metering study ('Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, Navigant, August 1 2013) and adjusted for Missouri climate region values using the relative climate- normal HDD data with a base temp ratio of 60°F. This load value is then divided by standard assumption of existing unit efficiency of 83.5% (estimate based on 29% of furnaces purchased in Missouri were condensing in 2000 (based on data from GAMA, provided to Department of Energy) (see 'Thermostat_FLH and Heat Load Calcs.xls'). The resulting values are generally supported by data provided by Laclede Gas, which showed an average pre-furnace replacement consumption of 1009 therms for St Louis, and a post-replacement consumption of 909. Assuming a typical hot water consumption at 225 therms (using defaults from http://energy.gov/eere/femp/energy-cost-calculator-electric-and-gas-water-heaters-0#output), this indicates a heating load of 684-784 therms. Ameren Missouri Efficient Products Evaluation PY2017

3.4.2 Air Source Heat Pump Including Dual Fuel Heat Pumps

DESCRIPTION

An air source heat pump (ASHP) provides heating and/or cooling by moving heat between indoor and outdoor air. A cold climate air source heat pump (ccASHP) operates the same as a traditional ASHP, but is able to meet a home's full heating load at lower outdoor temperatures approaching 0°F. A dual fuel heat pump (DFHP) pairs an air source heat pump with a gas furnace such that the air source heat pump provides heating in mild weather, and as temperature drops the heat pump shuts off and the furnace provides heating. This measure may also apply to replacing a Central Air Conditioner with non-electric heating with an Air Source Heat Pump. In this case, only cooling savings (ER1, ER2, ROF) may be claimed using the ASHP cooling algorithm. This measure applies to central ducted systems and single zone split-systems with ductless indoor units that are capable of meeting a home's full cooling and heating demand.

This measure characterizes:

1. TOS, NC: The installation of a new residential sized (<= 65,000 Btu/hr) air source heat pump that is more efficient than required by federal standards. This could relate to the replacement of an existing ASHP at the end of its useful life or the installation of a new ASHP in a new home.
2. EREP: The early removal of functioning electric heating and cooling systems from service, prior to its natural end of life, and replacement with a new high efficiency ASHP unit. To qualify as Early Replacement, the existing unit must be operational when replaced. If the SEER of the existing unit is known and the Baseline SEER is the actual SEER value of the unit replaced and if unknown use assumptions in the variable list below ($SEER_{exist}$ and $HSPF_{exist}$). If the operational status of the existing unit is unknown, use TOS assumptions.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new residential-sized (<= 65,000 Btu/hr) air source heat pump with specifications to be determined by the program.

The heating capacity of the efficient heat pump should be within 90% to 120% of the capacity of the existing equipment, unless the trade ally can provide documentation confirming the existing system is oversized. It is recommended to collect the existing and new unit capacities to confirm that the heat pump has sufficient capacity to minimize use of backup electric resistance heating.

Using a dual fuel heat pump, which uses a gas furnace for heating at lower outside air temperatures, or a cold climate rated heat pump are two options to ensure minimal use of backup electric resistance heating.

DEFINITION OF BASELINE EQUIPMENT

A new residential-sized (<= 65,000 Btu/hr) air source heat pump meeting federal standards.

New federal standards affecting heat pumps became effective January 1, 2023; however, these new federal standards will be adopted by the program, beginning 1/1/2024. Under the new standards, the baseline for the TOS measure is the federal standard efficiency level; 15 SEER (14.3 SEER2) and 8.6 HSPF (7.5 HSPF2), when replacing an existing air source heat pump; and 14 SEER (13.4 SEER2) and 3.41 HSPF when replacing a central air conditioner and electric resistance heating. Non-electric heating replaced with an air source heat pump can only claim cooling savings. Under the new federal standards, the M1 testing protocol was revised, resulting in new SEER and HSPF performance metrics, now called SEER2 and HSPF2. When quantifying energy savings, the same metric should be used for the existing, baseline, and new equipment. The following conversion formulas can be used to convert between efficiency metrics:

SEER2 = SEER x 0.96
HSPF2 = HSPF x 0.87

Through December 31, 2023, the baseline for the TOS measure is the previous federal standard efficiency level; 14 SEER and 8.2 HSPF, when replacing an existing air source heat pump; and 14 SEER and 3.41 HSPF when replacing a central air conditioner and electric resistance heating. Non-electric heating replaced with an air source heat pump can only claim cooling savings.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The expected measure life is assumed to be 18 years.[293]
Remaining life of existing ASHP/CAC equipment is assumed to be 6 years[294] and 18 years for electric resistance.

DEEMED MEASURE COST
Dual Fuel Heat Pump:

Efficiency (EER)	Cost (including labor) per measure
DFHP - SEER 19 MF heat pump base	$2,936.60
DFHP - SEER 20 MF heat pump base	$3,176.60
DFHP - SEER 21 MF heat pump base	$3,626.60

Air Source Heat Pump:

TOS/ROF: The incremental capital cost for this measure is dependent on the efficiency and capacity of the new unit.:

Efficiency (SEER)	ROF Incremental Cost ($)	Source
SEER 15	$303.00	IL TRM V8.0
SEER 16	$438.00	IL TRM V8.0
SEER 17	$724.00	IL TRM V8.0
SEER 18	$962.92	Derived using IL TRM ($/unit) and the % change in Mid Atlantic TRM V9 ($/ton)
SEER 19	$1,203.65	
SEER 20	$1,444.38	
SEER 21	$1,689.92	

Early Replacement (ER): The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume the following (note these costs are per ton of unit capacity):

Efficiency (SEER)	*ER Incremental Cost for 3 ton unit ($)	Source
SEER 15	$1,019.81	IL TRM V8.0
SEER 16	$1,154.81	IL TRM V8.0
SEER 17	$1,440.81	IL TRM V8.0
SEER 18	$1,679.73	Derived using IL TRM ($/unit) and the percent change in Mid-Atlantic TRM V9 ($/ton)
SEER 19	$1,920.46	
SEER 20	$2,161.19	
SEER 21	$2,406.74	
*Hypothetical values calculated based on a 3 ton system. Actual values based on system size and SEER combinations.		

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be $1,525[295] per ton of capacity. This cost should be discounted to present value using the utilities' real discount rate.

LOADSHAPE
Cooling RES
Heating RES

[293] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf.
[294] Assumed to be one third of effective useful life.
[295] Ibid. $1381 per ton (IL TRM V8.0) inflated using rate of 2.0%

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

TOS:

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER_{base} - 1/SEER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{base} - 1/HSFP_{ee})) / 1,000)] * ISR$$

Cooling only for Central Air Conditioning and Non-Electric Heating Backup

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER_{base} - 1/SEER_{ee})) / 1000) * ISR$$

EREP:[296]

ΔkWh for remaining life of existing unit (1st 6 years for replacing an ASHP, 18 years for replacing electric resistance):

$$= [((EFLH_{cool} * Capacity_{cool} * (1/SEER_{exist} - 1/SEER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{exist} - 1/HSFP_{ee})) / 1,000)] * ISR$$

Cooling only for Central Air Conditioning and Non-Electric Heating Backup

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER_{exist} - 1/SEER_{ee})) / 1000) * ISR$$

ΔkWh for remaining measure life (next 12 years if replacing an ASHP):

$$= [((EFLH_{cool} * Capacity_{cool} * (1/SEER_{base} - 1/SEER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{base} - 1/HSFP_{ee})) / 1,000)] * ISR$$

Cooling only for Central Air Conditioning and Non-Electric Heating Backup

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER_{base} - 1/SEER_{ee})) / 1000) * ISR$$

Where:

$EFLH_{cool}$ = Equivalent full load hours of air conditioning:[297]

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$ (Hours)
SF or MF	869
MFc (comprehensive envelope)	632[298]

$Capacity_{cool}$ = Cooling Capacity of Air Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

$SEER_{exist}$ = Seasonal Energy Efficiency Ratio of existing cooling system (kBtu/kWh)

= Use actual SEER rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.[299] If age is unknown, use 12 years.

= $SEER * (1-1.44\%)^{Age}$

If unknown, use defaults provided below:

[296] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a first year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input, which would be the either the new base to efficient savings or the (existing to efficient savings.

[297] Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren Missouri' service territory, suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

[298] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

[299] Based on IL TRM V8.0, which bases justification for degradation factors on page 21 of 'AIC HVAC Metering Study Memo FINAL 2_28_2018'. Default of 12 years based on the remaining measure life of the equipment.

Existing Cooling System	$SEER_{exist}$[300]
Air Source Heat Pump	7.2
Central AC	6.8
No central cooling[301]	Let '$1/SEER_{exist}$' = 0

$SEER_{base}$ = Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/kWh)[302]
= 14[303] through 12/31/2023 and 15 (14.3 SEER2) beginning 1/1/2024[304] when replacing an ASHP
= 14 (SEER 13.4) when replacing a CAC

$SEER_{ee}$ = Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/kWh)
= Actual

$EFLH_{heat}$ = Equivalent full load hours of heating:[305]

Weather Basis (Ameren Missouri Average)	$EFLH_{heat}$ (Hours)
SF or MF	1496 for ASHP, 1119 for DFHP, and 1769[306] for ccAHSP
MFc (comprehensive envelope)	510[307] for ASHP and DFHP, and 603 for ccASHP

$Capacity_{heat}$ = Heating Capacity of Air Source Heat Pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)

$HSPF_{exist}$ = Heating Seasonal Performance Factor of existing heating system (kBtu/kWh)
= Use actual HSPF rating where it is possible to measure or reasonably estimate. If not available use:

Existing Heating System	$HSPF_{exist}$
Air Source Heat Pump	5.44[308]
Electric Resistance	3.41[309]

$HSPF_{base}$ = Heating Seasonal Performance Factor of baseline Air Source Heat Pump (kBtu/kWh)[310]
=8.33[311] through 12/31/2023 and 8.6 (7.5 HSPF2) beginning 1/1/2024[312]

$HSFP_{ee}$ = Heating Seasonal Performance Factor of efficient Air Source Heat Pump (kBtu/kWh)
= Actual

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[300] ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8.
[301] If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.
[302] SEER to SEER2 conversion factor: SEER2 = SEER x 96%. Conversion factor for SEER to SEER2 is used when converting an existing system that is rated in SEER to SEER2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in the same metrics (e.g., both SEER or SEER2) before applying formulas.
[303] Based on minimum federal standard effective 1/1/2015: http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.
[304] Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5)
[305] Ameren Missouri HVAC Evaluation PY2017
[306] Evaluation – Opinion Dynamics review PY2022. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) for St. Louis and technological differences between traditional and cold climate heat pumps, which are capable of meeting whole home heating requirements at lower temperatures than traditional heat pumps, resulting in increased effective full load operating hours.
[307] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
[308] This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596 and applying to the average nameplate SEER rating of all early replacement qualifying equipment in Ameren PY2003-PY2004. This estimation methodology appears to provide a result within 10% of actual HSPF.
[309] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.
[310] HSPF to HSPF2 conversion factor: HSPF2 = HSPF x 87%. Conversion factor for HSPF to HSPF2 is used when converting an existing system that is rated in HSPF to HSPF2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in the same metrics (e.g., both HSPF or HSPF2) before applying formulas.
[311] Ameren Missouri HVAC Evaluation: PY2017.
[312] Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5)

ISR = In Service Rate = 100%[313]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of sale:

$$\Delta kW = \Delta kWh_{cooling} * CF$$

CF = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[313] Ameren Missouri HVAC Evaluation: PY2020.

3.4.3 Duct Sealing and Duct Repair

DESCRIPTION

This measure describes evaluating the savings associated with performing duct sealing to the distribution system of homes with central cooling and/or a ducted heating system. While sealing ducts in conditioned space can help with control and comfort, energy savings are largely limited to sealing ducts in unconditioned space where the heat loss is to outside the thermal envelope. Therefore, for this measure to be applicable at least 30% of ducts should be within unconditioned space (e.g., attic with floor insulation, vented crawlspace, unheated garages; basements should be considered conditioned space).

Three methodologies for estimating the savings associate from sealing the ducts are provided.

1. **Modified Blower Door Subtraction** – this technique is described in detail on p. 44 of the Energy Conservatory Blower Door Manual; http://dev.energyconservatory.com/wp-content/uploads/2014/07/Blower-Door-model-3-and-4.pdf.
   It involves performing a whole house depressurization test and repeating the test with the ducts excluded.
2. **Duct Blaster Testing** - as described in RESNET Test 803.7: http://www.resnet.us/standards/DRAFT_Chapter_8_July_22.pdf.
   This involves using a blower door to pressurize the house to 25 Pascals and pressurizing the duct system using a duct blaster to reach equilibrium with the inside. The air required to reach equilibrium provides a duct leakage estimate.
3. **Deemed Savings per Linear Foot** – this method provides a deemed conservative estimate of savings and should only be used where performance testing described above is not possible.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is sealed duct work throughout the unconditioned space in the home.

DEFINITION OF BASELINE EQUIPMENT

The existing baseline condition is leaky duct work with at least 30% of the ducts within the unconditioned space in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of this measure is 20 years.[314]

DEEMED MEASURE COST

The actual duct sealing measure cost should be used.

LOADSHAPE

HVAC RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Methodology 1: Modified Blower Door Subtraction

a. Determine Duct Leakage rate before and after performing duct sealing:

$$Duct\ Leakage\ (CFM50_{DL}) = (CFM50_{Whole\ House} - CFM50_{Envelope\ Only}) * SCF$$

Where:

$CFM50_{Whole\ House}$ = Standard Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differentials

$CFM50_{Envelope\ Only}$ = Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differentials with all supply and return registers sealed

[314] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

SCF = Subtraction Correction Factor to account for underestimation of duct leakage due to connections between the duct system and the home. Determined by measuring pressure with respect to the building in the sealed duct system, with the building pressurized to 50 Pascals with respect to the outside. Use the following look up table provided by energy conservatory to determine the appropriate subtraction correction factor:

House to Duct Pressure	Subtraction Correction Factor
50	1.00
49	1.09
48	1.14
47	1.19
46	1.24
45	1.29
44	1.34
43	1.39
42	1.44
41	1.49
40	1.54
39	1.60
38	1.65
37	1.71
36	1.78
35	1.84
34	1.91
33	1.98
32	2.06
31	2.14

House to Duct Pressure	Subtraction Correction Factor
30	2.23
29	2.32
28	2.42
27	2.52
26	2.64
25	2.76
24	2.89
23	3.03
22	3.18
21	3.35
20	3.54
19	3.74
18	3.97
17	4.23
16	4.51
15	4.83
14	5.20
13	5.63
12	6.12
11	6.71

b. Calculate duct leakage reduction, convert to $CFM25_{DL,}$[315] and factor in Supply and Return Loss Factors:

$$Duct\ Leakage\ Reduction\ (\Delta CFM25_{DL}) = (Pre\ CFM50_{DL} - Post\ CFM50_{DL}) * 0.64 * (SLF + RLF)$$

Where:

0.64 = Converts $CFM50_{DL}$ to $CFM25_{DL}$[316]

SLF = Supply Loss Factor[317]
= % leaks sealed located in Supply ducts * 1
Default = 0.5[318]

RLF = Return Loss Factor[319]
= % leaks sealed located in Return ducts * 0.5
Default = 0.25[320]

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[315] 25 Pascals is the standard assumption for typical pressures experienced in the duct system under normal operating conditions.
[316] To convert CFM50 to CFM25, multiply by 0.64 (inverse of the "Can't Reach Fifty" factor for CFM25; see Energy Conservatory Blower Door Manual).
[317] Assumes that for each percent of supply air loss there is one percent annual energy penalty. This assumes supply leaks are direct losses to the outside and are not recaptured back to the house. This could be adjusted downward to reflect regain of usable energy to the house from duct leaks. For example, during the winter some of the energy lost from supply leaks in a crawlspace will probably be regained back to the house (sometimes 1/2 or more may be regained). More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from Energy Conservatory Blower Door Manual.
[318] Assumes 50% of leaks are in supply ducts.
[319] Assumes that for each percent of return air loss there is a half percent annual energy penalty. Note that this assumes that return leaks contribute less to energy losses than do supply leaks. This value could be adjusted upward if there was reason to suspect that the return leaks contribute significantly more energy loss than "average" (e.g., pulling return air from a super-heated attic), or can be adjusted downward to represent significantly less energy loss (e.g., pulling return air from a moderate temperature crawl space). More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from Energy Conservatory Blower Door Manual.
[320] Assumes 50% of leaks are in return ducts.

c. Calculate electric savings

$$\Delta kWh = \Delta kWhCooling + \Delta kWhHeating$$

$$\Delta kWhCooling = \frac{\frac{\Delta CFM25_{DL}}{(CapacityCool/12{,}000 * 400)} * EFLHcool * CapacityCool}{1{,}000 * SEER}$$

$$\Delta kWhHeating_{Electric} = \frac{\frac{\Delta CFM25_{DL}}{(CapacityHeat/12{,}000 * 400)} * EFLHheat * CapacityHeat}{COP * 3{,}412}$$

$$\Delta kWhHeating_{Gas} = (\Delta Therms * Fe * 29.3)$$

Where:

$\Delta CFM25_{DL}$ = Duct leakage reduction in CFM2 as calculated above

CapacityCool = Capacity of Air Cooling system (Btu/hr)
= Actual

12,000 = Converts Btu/H capacity to tons

400 = Conversion of Capacity to CFM (400CFM / ton) [321]

EFLHcool = Equivalent Full Load Cooling Hours:[322]

Weather Basis (Ameren Missouri Average)	EFLHcool (Hours)
SF or MF	869
MFc (comprehensive envelope)	632[323]

1,000 = Converts Btu to kBtu

SEER = Seasonal Energy Efficiency Ratio of Air Conditioning equipment (kBtu/kWh)
= Actual - If not available, use:[324]

Equipment Type	Age of Equipment	SEER Estimate
Central AC	Before 2006	10
	After 2006	13
Heat Pump	Before 2006	10
	2006-2014	13
	2015 on	14

CapacityHeat = Heating output capacity (Btu/hr) of electric heat
= Actual

EFLHheat = Equivalent Full Load Heating Hours: [325]

Weather Basis (Ameren Missouri Average)	EFLHheat (Hours)
SF or MF	1496
MFc (comprehensive envelope)	510

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321 This conversion is an industry rule of thumb. E.g., see http://www.hvacsalesandsupply.com/Linked%20Documents/Tech%20Tips/61-Why%20400%20CFM%20per%20ton.pdf.

322 Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren territory suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

323 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

324 These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

325 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

COP = Efficiency in COP of Heating equipment
= Actual - If not available, use:[326]

System Type	Age of Equipment	HSPF Estimate	COP (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1

3412 = Converts Btu to kWh
ΔTherms = Therm savings as calculated in Natural Gas Savings
F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%[327]
29.3 = kWh per therm

Methodology 2: Duct Blaster Testing

$$\Delta kWh = \Delta kWhCooling + \Delta kWhHeating$$

$$\Delta kWhCooling = \frac{\frac{Pre\_CFM25 - Post\_CFM25}{CapacityCool/12{,}000 * 400} * EFLHcool * CapacityCool}{1{,}000 * SEER}$$

$$\Delta kWhHeating_{Electric} = \frac{\frac{Pre\_CFM25 - Post\_CFM25}{CapacityCool/12{,}000 * 400} * EFLHheat * CapacityHeat}{COP * 3{,}412}$$

$$\Delta kWhHeating_{Gas} = (\Delta Therms * Fe * 29.3)$$

Where:

Pre_CFM25 = Duct leakage in CFM25 as measured by duct blaster test before sealing
Post_CFM25 = Duct leakage in CFM25 as measured by duct blaster test after sealing
All other variables as provided above

Methodology 3: Deemed Savings[328]

$$\Delta kWh = \Delta kWh_{Cooling} + \Delta kWh_{HeatingElectric} + \Delta kWh_{HeatingGas}$$

$$\Delta kWhcooling = CoolSavingsPerUnit * Duct_{Length}$$

$$\Delta kWh_{HeatingElectric} = HeatSavingsPerUnit * Duct_{Length}$$

$$\Delta kWh_{HeatingGas} = (\Delta Therms * Fe * 29.3)$$

Where:

CoolSavingsPerUnit = Annual cooling savings per linear foot of duct

[326] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

[327] F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e.

[328] Savings per unit are based upon analysis performed by Cadmus for the 2011 Iowa Joint Assessment of Potential. It was based on 10% savings in system efficiency. This would represent savings from homes with significant duct work outside of the thermal envelope. With no performance testing or verification, a deemed savings value should be very conservative and therefore the values provided in this section represent half of the savings – or 5% improvement. These values are provided as a conservative deemed estimate for Missouri, while encouraging the use of performance testing and verification for determination of more accurate savings estimates.

Building Type	HVAC System	CoolSavingsPerUnit (kWh/ft)
Multifamily	Cool Central	0.70
Single-family	Cool Central	0.81
Manufactured	Cool Central	0.95
Multifamily	Heat Pump—Cooling	0.70
Single-family	Heat Pump—Cooling	0.81
Manufactured	Heat Pump—Cooling	0.95

$Duct_{Length}$ = Linear foot of duct
= Actual

HeatSavingsPerUnit = Annual heating savings per linear foot of duct

Building Type	HVAC System	HeatSavingsPerUnit (kWh/ft)
Manufactured	Heat Pump—Heating	5.06
Multifamily	Heat Pump - Heating	3.41
Single-family	Heat Pump— Heating	4.11

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * C$$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805[329]

NATURAL GAS SAVINGS

For homes with Natural Gas Heating:

Methodology 1: Modified Blower Door Subtraction

$$\Delta Therm = \frac{\frac{\Delta CFM25_{DL}}{CapacityHeat * 0.0136} * EFLHheat * CapacityHeat * \frac{\eta Equipment}{\eta System}}{100{,}000}$$

Where:

$\Delta CFM25_{DL}$ = Duct leakage reduction in CFM25
= As calculated in Methodology 1 under electric savings

CapacityHeat = Heating input capacity (Btu/hr)
= Actual

0.0125 = Conversion of Capacity to CFM (0.0125CFM / Btu/hr)[330]

ηEquipment = Heating Equipment Efficiency
= Actual[331] - If not available, use 83.5%[332]

[329] 2016 Ameren Missouri Coincident Peak Demand Factor for Residential Cooling. See reference "Ameren Missouri 2016 Appendix E - End Use Shapes and Coincident Factors.pdf."

[330] Based on natural draft furnaces requiring 100 CFM per 10,000 Btu, induced draft furnaces requiring 130CFM per 10,000Btu, and condensing furnaces requiring 150 CFM per 10,000 Btu (rule of thumb from http://contractingbusiness.com/enewsletters/cb_imp_43580/). Data provided by GAMA during the federal rulemaking process for furnace efficiency standards, suggested that in 2000, 29% of furnaces purchased in Missouri were condensing units. Therefore, a weighted average required airflow rate is calculated assuming a 50:50 split of natural v induced draft non-condensing furnaces, as 125 per 10,000Btu or 0.0125/Btu.

[331] The actual Heating Equipment Efficiency can be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. If there is more than one heating system, the weighted (by consumption) average efficiency should be used.
If the heating system or distribution is being upgraded within a package of measures together with the insulation upgrade, the new average heating system efficiency should be used.

[332] In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment; see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the state. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: (0.29*0.92) + (0.71*0.8) = 0.835.

ηSystem = Pre duct sealing Heating System Efficiency (Equipment Efficiency * Pre Distribution Efficiency)[333]
= Actual - If not available use 71.0%[334]
100,000 = Converts Btu to therms

Methodology 2: Duct Blaster Testing

$$\Delta Therms = \frac{\frac{Pre\_CFM25 - Post\_CFM25}{CapacityHeat * 0.0136} * EFLHgasheat * CapacityHeat * \frac{\eta Equipment}{\eta System}}{100{,}000}$$

Where:

All variables as provided above

Methodology 3: Deemed Savings[335]

$$\Delta Therms = HeatSavingsPerUnit * Duct_{Length}$$

Where:

HeatSavingsPerUnit = Annual heating savings per linear foot of duct

Building Type	HVAC System	HeatSavingsPerUnit (Therms/ft)
Multifamily	Heat Central Furnace	0.19
Single-family	Heat Central Furnace	0.21
Manufactured	Heat Central Furnace	0.26

$Duct_{Length}$ = Linear foot of duct
= Actual

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[333] The distribution efficiency can be estimated via a visual inspection and by referring to a look-up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf) - or by performing duct blaster testing.

[334] Estimated as follows: 0.835 * (1-0.15) = 0.710.

[335] Savings per unit are based upon analysis performed by Cadmus for the 2011 Joint Assessment of Potential. It was based on 10% savings in system efficiency. This would represent savings from homes with significant duct work outside of the thermal envelope. With no performance testing or verification, a deemed savings value should be very conservative and therefore the values provided in this section represent half of the savings – or 5% improvement. These values are provided as a conservative deemed estimate for Missouri, while encouraging the use of performance testing and verification for determination of more accurate savings estimates.

3.4.4 Mini/Multi-Split Air Source Heat Pump and Air Conditioners

DESCRIPTION

This measure is designed to calculate electric savings from retrofitting existing electric HVAC systems with ductless and/or ducted mini/multi-split heat pumps (*MMSHPs*) or mini/multi-split air conditioners. *MMSHPs* save energy in heating mode because they provide heat more efficiently than electric resistance heat and central ASHP systems. Additionally, *MMSHPs* use less fan energy to move heat and don't incur heat loss through a lengthy duct distribution system while operating at very low static pressure. Often *MMSHPs* are installed in addition to (do not replace) existing heating or cooling equipment because the existing heating or cooling equipment is inadequate to efficiently heat or cool the space. Both ductless and ducted indoor units can be installed as a mixed mini/multi-split heat pump or air conditioner under this measure. Duct runs for a ducted mini/multi-split indoor unit should be installed within the conditioned envelope, be well-sealed and insulated ducts, and maintain low static pressure per manufacturer specifications for the installation configuration to maximize energy savings.

For cooling, the proposed savings calculations are aligned with those of typical replacement systems. *MMSHPs* save energy in cooling mode because they provide cooling capacity more efficiently than other types of unitary cooling equipment. A *MMSHPs* installed in a home with an existing central ASHP or CAC system will save energy by offsetting some of the cooling energy of the ASHP or CAC. In order for this measure to apply, the control strategy for the heat pump or air conditioner is assumed to be chosen to maximize savings per installer recommendation.[336]

This measure was developed to be applicable to the following program type: NC, ROF, and ER.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the new equipment must be a high-efficiency, variable-capacity (typically "inverter-driven" DC motor) ductless and/or ducted mini/multi-split heat pump or air conditioning system that exceeds the program minimum efficiency requirements.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, baseline equipment must include a permanent electric resistance heating source or a ducted air-source heat pump or ducted air conditioner. For residences with central air conditioner/non-electric heating, cooling savings will only apply. For multifamily buildings, each residence must have existing individual heating equipment. Multifamily residences with central heating do not qualify for this characterization. Existing cooling equipment is assumed to be standard efficiency. Note that in order to claim cooling savings, there must be an existing air conditioning system (e.g. central air conditioning, Window ACs, or air source heat pump).

New federal standards affecting heat pumps became effective January 1, 2023. These new federal standards will be adopted by programs on January 1, 2024. Under the new standards, the baseline for a ROF measure is the federal standard efficiency; 15 SEER (14.3 SEER2) and 8.6 HSPF (7.5 HSPF2) when replacing a ducted air-source heat pump; 14 SEER (13.4 SEER2) and 3.41 HSPF when replacing a central air conditioner and electric resistance heating; 14 SEER (13.4 SEER2) when replacing central air conditioner with non-electric heating or no heating.

Under the new federal standards, the M1 testing protocol was revised, resulting in new SEER and HSPF performance metrics, now called SEER2 and HSPF2. When quantifying energy savings, the same metric should be used for the existing, baseline, and new equipment. The following conversion formulas can be used to convert between efficiency metrics:

SEER2 = SEER x 0.96
HSPF2 = HSPF x 0.87

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

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[336] The whole purpose of installing ductless heat pumps is to conserve energy, so the installer can be assumed to be capable of recommending an appropriate control strategy. For most applications, the heating setpoint for the ductless heat pump should be at least 2F higher than any remaining existing system and the cooling setpoint should be at least 2F cooler than the existing system (this should apply to all periods of a programmable schedule, if applicable). This helps ensure that the ductless heat pump will be used to meet as much of the load as possible before the existing system operates to meet the remaining load. Ideally, the new ductless heat pump controls should be set to the current comfort settings, while the existing system setpoints should be adjusted down (heating) and up (cooling) to capture savings.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years.[337]

DEEMED MEASURE COST

The incremental cost for this measure is provided below:

Measure	Incremental Cost ($/ 1.5 ton)	Source
Mini/Multi-Split AC - ER1 SF	$1,231.16	IL-TRM v8.0
Mini/Multi-Split AC - Replace on fail SF	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP - Replace on fail SF NC	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP - Replace on fail SF ROF	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP Replace Electric Resistance ER1 SF	$2,504.17	IL-TRM v8.0
Mini/Multi-Split ASHP Replace Electric Resistance ROF	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP ER1 SF	$648.60	IL-TRM v8.0
Mini/Multi-Split AC - ER1 MF	$1,231.16	IL-TRM v8.0
Mini/Multi-Split AC - Replace on fail MF	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP - Replace on fail MF NC	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP - Replace on fail MF ROF	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP Replace Electric Resistance ER1 MF	$2,504.17	IL-TRM v8.0
Mini/Multi-Split ASHP Replace Electric Resistance ROF MF	$336.00	IL-TRM v8.0
Mini/Multi-Split ASHP ER1 MF	$648.60	IL-TRM v8.0

LOADSHAPE

Cooling RES

Heating RES

Algorithms

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings

$$\Delta kWh = \Delta kWh_{heating} + \Delta kWh_{cooling}$$

$$\Delta kWh_{heating} = ((Capacity_{heat} * EFLH_{heat} * (1/HSPF_{exist} - 1/HSPF_{ee})) / 1000) * HF * ISR$$

$$\Delta kWh_{cooling} = ((Capacity_{cool} * EFLH_{cool} * (1/SEER_{exist} - 1/SEER_{ee})) / 1000) * HF * ISR$$

$$\Delta kW = \Delta kWh_{cooling} * CF \qquad \Delta kW = \Delta kWh_{heating} * CF$$

Electric savings – cooling only in presence of non-electric heating or MMAC (Mini/Multi-Split AC)

[337] Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.

$$\Delta kWh_{cooling} = ((Capacity_{cool} * EFLH_{cool} * (1/SEER_{exist} - 1/SEER_{ee})) / 1000) * HF * ISR$$

$$\Delta kW = \Delta kWh_{cooling} * CF$$

Where:

$Capacity_{heat}$ = Heating capacity of the ductless heat pump unit in Btu/hr
= Actual

$EFLH_{heat}$ = Equivalent Full Load Hours for heating. See table below:

Weather Basis (Ameren Missouri Average)	$EFLH_{heat}$[338]
SF or MF	1,034
MFc (comprehensive envelope)	393

$HSPF_{exist}$ = HSPF rating of existing equipment (kBtu/kWh)

Existing Equipment Type	$HSPF_{exist}$[339]
Electric resistance heating	3.412
Air Source Heat Pump	6.58

$HSPF_{base}$ = HSPF rating of baseline equipment (kBtu/kWh)
= 8.2[340] through 12/31/2023 and 8.6 (7.5 HSPF2) beginning 1/1/2024[341] when replacing an ASHP
= 3.412 when replacing electric resistance heating

$HSPF_{ee}$ = HSPF rating of new equipment (kBtu/kWh)
= Actual installed

$Capacity_{cool}$ = the cooling capacity of the ductless heat pump unit in Btu/hr.[342]
= Actual installed

$SEER_{exist}$ = SEER rating of existing equipment (kBtu/kWh)

= Use actual SEER rating where possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.[343] If age is unknown, use 12 years.
= $SEER * (1-1.44\%)^{Age}$

If unknown, see table below

Existing Cooling System	$SEER_{exist}$[344]
Air Source Heat Pump	7.2

[338] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
[339] Ameren Missouri Heating and Cooling Evaluation PY2018
[340] Based on minimum federal standard effective 1/1/2015: http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.
[341] Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5)
[342] 1 Ton = 12 kBtu/hr.
[343] Based on IL TRM V8.0, which bases justification for degradation factors on page 21 of 'AIC HVAC Metering Study Memo FINAL 2_28_2018'. Default of 12 years based on the remaining measure life of the equipment.
[344] ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8, 78.9% of 8.0 SEER RAC nameplate gives an operational SEER of 6.3.

Central AC	6.8
Room AC	6.3[345]
No existing cooling[346]	Let '1/SEER_exist' = 0

$SEER_{base}$ = Seasonal Energy Efficiency Ratio of baseline equipment (kBtu/kWh)[347]
= 14[348] through 12/31/2023 and 15 (14.3 SEER2) beginning 1/1/2024[349] when replacing an ASHP
= 14 (13.4 SEER2) when replacing a CAC

$SEER_{ee}$ = SEER rating of new equipment (kBtu/kWh)
= Actual installed[350]

$EFLH_{cool}$ = Equivalent Full Load Hours for cooling. See table below

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$
SF or MF	635
MFc (comprehensive envelope)	417

ISR = In Service Rate = 100%[351]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh_{cooling} * CF$$

Where:

CF = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

345 Estimated by converting the EER assumption using the conversion equation; EER_base = (-0.02 * SEER_base^2) + (1.12 * SEER). From Wassmer, M. (2003), "A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations," (Masters thesis) University of Colorado at Boulder. Adjusted to account for degradation per above footnote.

346 If there is no existing cooling in place but the incentive encourages installation of a new DMSHP with cooling, the added cooling load should be subtracted from any heating benefit.

347 SEER to SEER2 conversion factor: SEER2 = SEER x 96%. Conversion factor for SEER to SEER2 is used when converting an existing system that is rated in SEER to SEER2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in the same metrics (e.g., both SEER or SEER2) before applying formulas.

348 Based on minimum federal standard effective 1/1/2015: http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.

349 Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5)

350 Note that if only an EER rating is available, use the following conversion equation; EER_base = (-0.02 * SEER_base^2) + (1.12 * SEER). From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

351 Ameren Missouri HVAC Evaluation: PY2020.

3.4.5 Standard Programmable Thermostat

DESCRIPTION
This measure characterizes the household energy savings from the installation of a new standard programmable thermostat for reduced heating and cooling energy consumption through temperature set-back during unoccupied or reduced demand times.

Energy savings are applicable at the household level; installation of multiple programmable thermostats per home does not accrue additional savings.

If the home has a heat pump, a programmable thermostat specifically designed for heat pumps should be used to minimize the use of backup electric resistance heat systems.

This measure was developed to be applicable to the following program types: RF, and DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
The criteria for this measure are established by replacement of a manual-only temperature control with one that has the capability to adjust temperature setpoints according to a schedule without manual intervention.

DEFINITION OF BASELINE EQUIPMENT
For new thermostats the baseline is a non-programmable thermostat requiring manual intervention to change temperature set point.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The expected equipment life of a programmable thermostat is assumed to be 10 years.[352]

DEEMED MEASURE COST
Actual material and labor costs should be used if the implementation method allows. If unknown (e.g., through a retail program), the capital cost for the new installation is assumed to be $70.[353]

LOADSHAPE
Cooling RES
Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS
For central air conditioners and air source heat pumps:

$$\Delta kWh_{cool} = EFLH_{cool} * Capacity_{Cooling} * \left(\frac{1}{SEER}\right) * SBdegrees * SF * EF/1000$$

For air source heat pumps there are additional heating savings:

$$\Delta kWh_{heat} = EFLH_{heat} * Capacity_{Heating} * \left(\frac{1}{HSPF}\right) * SBdegrees * SF * EF/1000$$

Where:

$EFLH_{cool}$ = Equivalent full load hours of air conditioning[354]:

[352] Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007. Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a large scale but only 2-year study of the energy impacts of programmable thermostats, the longer-term impacts should be assessed.
[353] Market prices vary significantly in this category, generally increasing with thermostat capability and sophistication. The basic functions required by this measure's eligibility criteria are available on units readily available in the market for $30. Labor is assumed to be one hour at $40 per hour.
[354] Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren Missouri' service territory, suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$ (Hours)
SF or MF	869
MFc (comprehensive envelope)	632

$Capacity_{Cooling}$ = Cooling capacity of system in BTU/hr (1 ton = 12,000 BTU/hr)
= Use Actuals based upon units served

SEER = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)
If unknown, use defaults provided below:

Cooling System	SEER
Air Source Heat Pump	10[355]
Central AC	10[356]

HSPF = Heating Season Performance Factor of heating system (kBtu/kWh)
If unknown, use defaults provided below:

Existing Heating System	$HSPF_{exist}$
Air Source Heat Pump	7.00[357]
Electric Resistance	3.41[358]

$EFLH_{heat}$ = Equivalent full load hours of heating:[359]

Weather Basis (Ameren Missouri Average)	$EFLH_{heat}$ (Hours)
SF or MF	1496
MFc (comprehensive envelope)	510

$Capacity_{Heating}$ = Heating capacity of system in BTU/hr (1 ton = 12,000 BTU/hr)
= Use Actuals based upon units served
SBdegrees = weighted sum of setback degrees to comfort temperature
= SBdegrees Heating = 1.8[360]
= SBdegrees Cooling = 1.91[361]
SF = Savings factors from ENERGY STAR® calculator
= 3% / degree heat, 6% / degree cool
EF = Efficiency ratio from Cadmus metering study
= 13% heat[362]
= 100% cool[363]

[355] IL-TRM (V5) - based on minimum federal standards between 1992 and 2006 – Ameren Missouri Community Saver Program Evaluation PY2018.
[356] IL-TRM - based on minimum federal standards between 1992 and 2006 – Ameren Missouri Community Saver Program Evaluation PY2018.
[357] IL-TRM (Based on minimum federal standards between 1992 and 2006) – Ameren Missouri Community Saver Program Evaluation PY2018.
[358] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.
[359] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
[360] Ameren Missouri Community Saver Program Evaluation PY2018 Site Visit Thermostat SB Data.
[361] Ameren Missouri Community Saver Program Evaluation PY2018
Site Visit Thermostat SB Data.
[362] Ameren Missouri Community Saver Program Evaluation PY2014 Cadmus metering study (PY2014 pg. 31).
[363] Ameren Missouri Community Saver Program Evaluation PY2017.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh_{cooling} * CF$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0009474181

NATURAL GAS ENERGY SAVINGS

$$\Delta Therms = \%FossilHeat * HeatingConusmption_{Gas} * HF * Heating_{Reduction} * Eff_{ISR} * PF$$

Where:

%FossilHeat = Percentage of heating savings assumed to be Natural Gas

Heating fuel	%FossilHeat
Electric	0%
Natural Gas	100%
Unknown	65%[364]

$HeatingConsumption_{Gas}$ = Estimate of annual household heating consumption for gas heated single-family homes.[365]

Weather Basis (City based upon)	Gas_Heating_ Consumption (Therms)
St Louis, MO	680

Other variables as provided above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[364] Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

[365] Values in table are based on average household heating load (834 therms) for Chicago based on Illinois furnace metering study ('Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, Navigant, August 1, 2013) and adjusted for Missouri weather basis values using the relative climate normals HDD data with a base temp ratio of 60°F. This load value is then divided by standard assumption of existing unit efficiency of 83.5% (estimate based on 29% of furnaces purchased in Missouri were condensing in 2000 (based on data from GAMA, provided to Department of Energy) (see 'Thermostat_FLH and Heat Load Calcs.xls'). The resulting values are generally supported by data provided by Laclede Gas, which showed an average pre-furnace replacement consumption of 1009 therms for St Louis, and a post-replacement consumption of 909. Assuming a typical hot water consumption at 225 therms (using defaults from http://energy.gov/eere/femp/energy-cost-calculator-electric-and-gas-water-heaters-0#output), this indicates a heating load of 684-784 therms.

3.4.6 HVAC Tune-Up (Central Air Conditioning or Air Source Heat Pump)

DESCRIPTION

This measure involves the measurement of refrigerant charge levels and airflow over the central air conditioning or heat pump unit coil, correction of any problems found, and post-treatment re-measurement. Tune-up activities include a general tune-up, refrigerant charge, indoor coil cleaning, and outdoor coil cleaning. These tune-up actions may be performed individually or as a packaged service with more than one tune-up activity.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A tuned and commissioned residential central air conditioning unit or air source heat pump.

DEFINITION OF BASELINE EQUIPMENT

An existing residential central air conditioning unit or air source heat pump that has required tuning to restore optimal performance.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 2 years.[366]

DEEMED MEASURE COST

As a RF measure, actual costs should be used. If unavailable, the measure cost should be assumed to be $175.[367] The table below identifies more specific costs for varying services.

Tune- up Service for HP or AC	Incremental Cost ($)	
General Tune-Up (no charge or coil clean)	$70.00	
Tune-up / refrigerant charge	$81.00	$175.00
Tune-up / Indoor Coil (Evaporator) Cleaning	$63.00	
Tune-up / Outdoor Coil (Condenser) Cleaning	$31.00	
Tune-Up / Packaged Service	$185[368]	

LOADSHAPE

Cooling RES
Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh_{Central\ AC} = ((EFLH_{cool} * Capacity_{cool} * (1/SEER_{test-in} - 1/SEER_{test-out})) / 1{,}000)$$

$$\Delta kWh_{ASHP} = ((EFLH_{cool} * Capacity_{cool} * (1/SEER_{test-in} - 1/SEER_{test-out})) / 1{,}000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{test-in} - 1/HSFP_{test-out})) / 1{,}000)$$

Where:

$EFLH_{cool}$ = Equivalent full load hours of air conditioning
= dependent on location:[369]

[366] Sourced from DEER Database Technology and Measure Cost Data.
[367] Based on personal communication with HVAC efficiency program consultant Buck Taylor of Roltay Inc., 6/21/10, who estimated the cost of tune up at $125 to $225, depending on the market and the implementation details.
[368] Estimated average packaged tune-up cost based on implementer data from 2015-2016.
[369] Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren territory suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

$Capacity_{cool}$ = Cooling Capacity of Air Source Heat Pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)

$SEER_{test-in}$ = Seasonal Energy Efficiency Ratio of existing cooling system before tuning (kBtu/kWh)
= In most instances, test-in EER will be determined and noted prior to tuning. SEER rating can be estimated by using the following relationship:[370] $EER = (-0.02 * SEER^2) + (1.12 * SEER)$
When unknown,[371] assume SEER = 11.9

$SEER_{test-out}$ = Seasonal Energy Efficiency Ratio of existing cooling system after tuning (kBtu/kWh)
= In most instances, test-out EER will be determined and noted after tuning. SEER rating can be estimated by using the following relationship:[372] $EER = (-0.02 * SEER^2) + (1.12 * SEER)$

$EFLH_{heat}$ = Equivalent full load hours of heating:

$Capacity_{heat}$ = Heating Capacity of Air Source Heat Pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)

$HSPF_{test-in}$ = Heating Seasonal Performance Factor of existing ASHP before tuning (kBtu/kWh)
= Use actual HSPF rating where it is possible to measure or reasonably estimate. If not available, assumeHSPF = 6.3.[373]

$HSPF_{test-out}$ =Heating System Performance Factor of existing ASHP after tuning (kBtu/kWh)
= Use actual HSPF rating where it is possible to measure or reasonably estimate.

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$ (Hours)	$EFLH_{heat}$ (Hours)
SF or MF	869[374]	1496[375]
MFc (comprehensive envelope)	632[376]	510[377]

When SEER test-in and test-out values are unknown, tune-ups are assumed to improve efficiency as follows:

Measure	% Improvement	$SEER_{test-out}$ (based on default 11.9 test-in value)
Refrigerant charge adjustment	22.0%	15.3
Condenser Cleaning Only	7.9%	12.8
Indoor coil cleaning	3.8%	12.4
General tune-up	5.6%	12.6
Packaged Service	13.6%[378]	13.8

When HSPF test-out values are unknown, use the following default test-out values based on the tune-up service(s) performed:

Measure	$HSPF_{test-out}$ (based on default 6.3 test-in value)

370 Based on Wassmer, M. (2003)," A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations," (Masters thesis) University of Colorado at Boulder. Note this is appropriate for single speed units only.
371 Using aforementioned relationship and test-in efficiency of 10.5 EER, as listed in "Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015."
372 Based on Wassmer, M. (2003), "A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations," (Masters thesis), University of Colorado at Boulder. Note: this is appropriate for single speed units only.
373 Based on evaluation results outlined in "Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015."
374 Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren Missouri' service territory, suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).
375 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
376 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
377 Ibid.
378 Average percentage improvement across 74 packaged service tune-up measures in the Ameren Missouri PY2019 Low Income Multifamily program.

Refrigerant charge adjustment	6.72
Condenser Cleaning Only	6.42
Indoor coil cleaning	6.36
General tune-up	6.38
Packaged Service	7.29[379]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh_{cooling} * CF$$

Where:

CF = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[379] Average percentage improvement across 74 packaged service tune-up measures in the Ameren Missouri PY2019 Low Income Multifamily program.

3.4.7 Blower Motor

DESCRIPTION

A new furnace with a brushless permanent magnet (BPM) blower motor is installed instead of a new furnace with a lower efficiency motor. This measure characterizes only the electric savings associated with the fan and could be coupled with gas savings associated with a more efficient furnace. Savings decrease sharply with static pressure so duct improvements, and clean, low pressure drop filters can maximize savings. Savings improve when the blower is used for cooling as well and when it is used for continuous ventilation, but only if the non-BPM motor would have been used for continuous ventilation too. If the resident runs the BPM blower continuously because it is a more efficient motor and would not run a non-BPM motor that way, savings are near zero and possibly negative. This characterization uses a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin, which accounted for the effects of this behavioral impact.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A furnace with a brushless permanent magnet (BPM) blower motor, also known by the trademark ECM, BLDC, and other names.

DEFINITION OF BASELINE EQUIPMENT

A furnace with a non-BPM blower motor. As part of the Code of Federal Regulations, energy conservation standards for covered residential furnace fans became effective on July 3, 2019 (10 CFR 430.32(y)). This code requirement effectively makes ECMs part of the baseline for New Construction (NC), Replace-on-Fail (ROF), Time-of-Replacement (TOS), and Early Replacement (EREP) scenarios.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years.[380]

DEEMED MEASURE COST

The capital cost for this measure is assumed to be:

Incremental Cost ($)	
$74.33[381]	Time of Sale
$475[382]	Early Replacement

LOADSHAPE

HVAC RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh_{Heating\ Mode} = (1 - \%\ with\ New\ ASHP) \times \left(400\frac{kWh}{year} \times \frac{Heating\ EFLH}{Wisconsin\ Heating\ EFLH}\right) * HF * ISR$$

$$\Delta kWh_{Cooling\ Mode} = (1 - \%\ with\ New\ Central\ Cooling) \times \left(70\frac{kWh}{year} \times \frac{Cooling\ EFLH}{Wisconsin\ CoolingEFLH}\right) * HF * ISR$$

$$\Delta kWh_{Auto\ Circulation} = \left(25\frac{kWh}{year} \times \frac{CoolingEFLH}{Wisconsin\ CoolingEFLH} + 2960\frac{kWh}{year} \times RT\% - 30\frac{kWh}{year}\right) * HF * ISR$$

$$\Delta kWh_{Continous\ Circulation} = \left(25\frac{kWh}{year} \times \frac{CoolingEFLH}{Wisconsin\ CoolingEFLH} + 2960\frac{kWh}{year} \times RT\% - 30\frac{kWh}{year}\right) * HF * ISR$$

Where:

[380] Consistent with assumed life of a new gas furnace. Table 8.3.3 The technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf.

[381] Adapted from Tables 8.2.3 and 8.2.13 in http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/hvac_ch_08_lcc_2011-06-24.pdf.

[382] Minnesota TRM, https://www.energy.gov/sites/prod/files/2014/02/f7/case_study_variablespeed_furnacemotor.pdf.

Parameter	Value
Wisconsin Cooling Savings kWh/year	70.00
Cooling Savings All Systems	25.00
Wisconsin Cooling EFLH	542.50
Wisconsin Heating Savings kWh/year	400.00
Wisconsin Heating EFLH	2,545.25
Wisconsin Circulation Savings kWh/year	2,960.00
RT=Percent additional run time factor	8.81%
Standby losses	30
Saint Louis Heating EFLH	2,009.00
Saint Louis Cooling EFLH	1,215.00
% with New Central Cooling	82%[383]
% with New ASHP	10%[384]
ISR	100%[385]
HF	100%[386]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805

NATURAL GAS SAVINGS

Δtherms[387] = - Heating Savings * 0.03412 / AFUE

Where:

0.03412 = Converts kWh to therms
AFUE = Efficiency of the Furnace
= Actual. If unknown assume 95%[388] if in new furnace or 64.4 AFUE%[389] if in existing furnace

Using defaults:

For new Furnace = - (430 * 0.03412) / 0.95
= - 15.4 therms
For existing Furnace = - (430 * 0.03412) / 0.644
= - 22.8 therms

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

383 Ameren Missouri HVAC Program Evaluation PY2019.
384 Ibid.
385 Ameren Missouri HVAC Program Evaluation PY2020.
386 Household Factor (HF) is assumed to be 100%. 65% multifamily value is not applicable for this measure, as savings should be based upon pressure drop in the system.
387 The blower fan is in the heating duct so all, or very nearly all, of its waste heat is delivered to the conditioned space. Negative value since this measure will increase the heating load due to reduced waste heat.
388 Minimum efficiency rating from ENERGY STAR® Furnace Specification v4.0, effective February 1, 2013.
389 Average nameplate efficiencies of all early replacement qualifying equipment in Ameren IL PY2003-PY2004.

3.4.8 Central Air Conditioner

DESCRIPTION
This measure characterizes:

1. TOS: The installation of a new residential sized (<= 65,000 Btu/hr) central air conditioning ducted split system meeting ENERGY STAR® efficiency standards presented below. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.
2. EREP: Early Replacement determination will be defined by program requirements. All other conditions will be considered TOS. The baseline SEER of the existing central air conditioning unit replaced: If the SEER of the existing unit is known and, the baseline SEER is the actual SEER value of the unit replaced. If the SEER of the existing unit is unknown, use assumptions in variable list below (SEER_exist).

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
In order for this characterization to apply, the efficient equipment is assumed to be a ducted split central air conditioning unit meeting the minimum ENERGY STAR® efficiency level standards; 15 SEER and 12 EER.

DEFINITION OF BASELINE EQUIPMENT
New federal standards affecting testing protocols for central air conditioners became effective January 1, 2023, but did not affect the standard efficiency of central air conditioners. The baseline for the TOS measure is based on the current federal standard efficiency level: 14 SEER (13.4 SEER2) and 11 EER (10.6 EER2).

Under the new federal standards, the M1 testing protocol was revised, resulting in a new SEER performance metric called SEER2. When quantifying energy savings, the same metric should be used for the existing, baseline, and new equipment. The following conversion formula can be used to convert between efficiency metrics:

SEER2 = SEER x 0.96

THE BASELINE FOR THE EARLY REPLACEMENT MEASURE IS THE EFFICIENCY OF THE EXISTING EQUIPMENT FOR THE ASSUMED REMAINING USEFUL LIFE OF THE UNIT AND THE NEW BASELINE AS DEFINED ABOVE[390] FOR THE REMAINDER OF THE MEASURE LIFE. DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The expected measure life is assumed to be 18 years.[391]

Remaining life of existing equipment is assumed to be 6 years.[392]

DEEMED MEASURE COST
TOS: The incremental capital cost for this measure is dependent on efficiency. Assumed incremental costs are provided below:

Early replacement: The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume the following:

[390] Baseline SEER and EER should be updated when new minimum federal standards become effective.
[391] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf.
The "lifespan" of a central air conditioner is about 15 to 20 years (US DOE: http://www.energysavers.gov/your_home/space_heating_cooling/index.cfm/mytopic=12440).
[392] Assumed to be one third of effective useful life.

Efficiency Level	ROF Cost ($)	*Early Replacement Cost [393]	Source
SEER 14	$0.00	$447.06	IL-TRM v8.0
SEER 15	$108	$555.06	IL-TRM v8.0
SEER 16	$221	$668.06	IL-TRM v8.0
SEER 17	$620.00	$1,067.06	IL-TRM v8.0
SEER 18	$826.67	$1,273.73	Derived using IL-TRM ($/unit) and the percentage change in Mid-Atlantic TRM V9 (NEEP)($/ton)
SEER 19	$1,033.33	$1,480.39	
SEER 20	$1,240.00	$1,687.06	
SEER 21	$1,446.67	$1,893.73	
Average	$686.96	$1,134.02	
*Hypothetical values calculated based on a 3 ton system. Actual values based on system size and SEER combinations.			

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be $3,217.[394] This cost is based on a 3 ton unit and should be discounted to present value using the utilities' discount rate.

LOADSHAPE
Cooling RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS
Time of sale:

$$\Delta kWh = ((FLH_{cool} * Capacity * (1/SEER_{base} - 1/SEER_{ee}))/1{,}000) * HF * ISR$$

Early replacement:[395]

ΔkWh for remaining life of existing unit (1st 6 years):

$$= ((FLH_{cool} * Capacity * (1/SEER_{exist} - 1/SEER_{ee}))/1{,}000) * HF * ISR$$

ΔkWh for remaining measure life (next 12 years):

$$= ((FLH_{cool} * Capacity * (1/SEER_{base} - 1/SEER_{ee}))/1{,}000) * HF * ISR$$

Where:

FLH_{cool} = Full load cooling hours:[396]

Weather Basis (Ameren Missouri Average)	EFLHcool (Hours)
SF or MF	869
MFc (comprehensive envelope)	632[397]

[393] These values are calculated in the deemed tables based on the unit size and SEER combination.
[394] Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR® central AC calculator, $2,857, and applying inflation rate of 2.0% (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls). While baselines are likely to shift in the future, there is currently no good indication of what the cost of a new baseline unit will be in 6 years. In the absence of this information, assuming a constant federal baseline cost is within the range of error for this prescriptive measure.
[395] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).
[396] Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren territory suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).
[397] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

Capacity = Size of new equipment in Btu/hr (note 1 ton = 12,000Btu/hr)
= Actual installed, or if actual size unknown 33,600Btu/hr for single-family buildings[398]

$SEER_{base}$ = Seasonal Energy Efficiency Ratio of baseline unit (kBtu/kWh)[399]
= 14 or 13.4 SEER2[400]

$SEER_{exist}$ = Seasonal Energy Efficiency Ratio of existing unit (kBtu/kWh)
= Use actual SEER rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.[401] If age is unknown, use 12 years.
= $SEER * (1-1.44\%)^{Age}$
If unknown, assume 10.0.[402]

$SEER_{ee}$ = Seasonal Energy Efficiency Ratio of ENERGY STAR® unit (kBtu/kWh)
= Actual installed or 14.5 if unknown

HF = For Multifamily units, use a factor of 65% to convert residential single family to multifamily capacity. If actual capacity is used apply 100%.

ISR = In service rate
= 100%[403]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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398 Actual unit size required for multifamily building, no size assumption provided because the unit size and resulting savings can vary greatly depending on the number of units.

399 SEER to SEER2 conversion factor: SEER2 = SEER x 96%. Conversion factor for SEER to SEER2 is used when converting an existing system that is rated in SEER to SEER2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in the same metrics (e.g., both SEER or both SEER2) before applying formulas.

400 Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5)

401 Based on IL TRM V8.0, which bases justification for degradation factors on page 21 of 'AIC HVAC Metering Study Memo FINAL 2_28_2018'. Default of 12 years based on the remaining measure life of the equipment.

402 Estimate based on Department of Energy standard between 1992 and 2006. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then that should be used.

403 Ameren Missouri HVAC Evaluation: PY2020.

3.4.9 Filter Cleaning or Replacement and Dirty Filter Alarms

DESCRIPTION

An air filter on a central forced air heating system is replaced prior to the end of its useful life with a new filter, resulting in a lower pressure drop across the filter. As filters age, the pressure drop across them increases as filtered medium accumulates. Replacing filters before they reach the point of becoming ineffective can save energy by reducing the pressure drop required by filtration, subsequently reducing the load on the blower motor.

This measure was developed to be applicable to the following program type: RET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new filter offering a lower pressure drop across the filter medium compared to the existing filter.

DEFINITION OF BASELINE EQUIPMENT

A filter that is nearing the end of its effective useful life, defined by having a pressure drop twice that of its original state.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 1 year[404] for a filter replacement and 14 years for a dirty filter alarm.

DEEMED MEASURE COST

Actual material and labor cost should be used if known, since there is a wide range of filter types and costs. If unknown,[405] the cost of a fiberglass filter is assumed to be $7.33 and the cost of a pleated filter is assumed to be $15.66. If unknown, the cost of a dirty filter alarm is assumed to be $5.

LOADSHAPE

HVAC RES

Algorithm

CALCULATION OF SAVINGS

Electric energy savings are calculated by estimating the difference in power requirements to move air through the existing and new filter and multiplying by the anticipated operating hours of the blower during the heating season.

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = kWh_{heating} + kWh_{cooling}$$

$$kWh_{heating} = \%Heating * kW_{motor} * EFLH_{heat} * EI * Utility\ Adjustment * ISR$$

$$kWh_{cooling} = \%AC * kW_{motor} * EFLH_{cool} * EI * Utility\ Adjustment * ISR$$

[404] Many manufacturers suggest replacing filters more often than an annual basis, however this measure assumes that a filter will generally last one full heating season before it needs replacement.

[405] Assumes an average price of $1.08 for fiberglass and $9.41 for pleated, plus $6.25 in labor (based on 15 minutes, including portion of travel time, and $25 per hour, which is in line with the typical prevailing wage of a General Laborer, as per Annual Wage Order No. 23 documents published by the Missouri Department of Labor). Average filter costs sourced from “Air Filter Testing, Listing, and Labeling,” Docket #12-AAER-2E prepared for the California Energy Commission, July 23, 2013.

Where:

Factor	Term	School Value
%Heating	Fraction of participants with electric heating	95.65%[406]
%AC	Fraction of participants with central cooling	95.65%[407]
kW_{motor}	Average motor full load electric demand (kW) - Kits	0.5
	Average motor full load electric demand (kW) – MFLI	0.43
$EFLH_{heat}$	Equivalent Full Load Hours (EFLH) Heating (hours/year) – SF or MF	1496
	Equivalent Full Load Hours (EFLH) Heating (hours/year) - MFc (comprehensive envelope)	510[408]
$EFLH_{cool}$	Equivalent Full Load Hours (EFLH) Cooling (hours/year) - SF or MF	869
	Equivalent Full Load Hours (EFLH) Cooling (hours/year) - MFc (comprehensive envelope)	632[409]
EI	Efficiency Improvement (%)	15%
Utility Adjustment	% Homes in Service Territory	72%[410]
ISR	In Service Rate - Kits	44%[411]
	In Service Rate – Appliance Recycling Program	9%[412]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = 0.0004660805

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[406] Ameren Missouri Energy Efficient Kits Evaluation: PY2018.
[407] Ibid.
[408] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
[409] Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.
[410] Ameren Missouri Energy Efficient Kits Evaluation: PY2019.
[411] Ibid.
[412] Ameren Missouri Appliance Recycling Evaluation: PY2019.

3.4.10 Packaged Terminal Air Conditioner (PTAC) and Packaged Terminal Heat Pump (PTHP)

DESCRIPTION

A PTAC is a packaged terminal air conditioner that cools and provides heat through an electric resistance heater (heat strip). A PTHP is a packaged terminal heat pump. A PTHP uses its compressor year-round to heat or cool. In warm weather, it efficiently captures heat from inside a space and pumps it outside for cooling. In cool weather, it captures heat from outdoor air and pumps it into a space, adding heat from electric heat strips as necessary to provide heat.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

This measure characterizes:

1. TOS: the purchase and installation of a new efficient PTAC or PTHP.
2. EREP: the early removal of an existing PTAC or PTHP from service, prior to its natural end of life, and replacement with a new efficient PTAC or PTHP unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life. The measure is only valid for non-fuel switching installations – for example replacing a cooling only PTAC with a PTHP can currently not use the TRM.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be PTACs or PTHPs that exceed baseline efficiencies.

DEFINITION OF BASELINE EQUIPMENT

TOS: the baseline condition is defined by the Code of Federal Regulations at 10 CFR 431.97(c), section §431.97.

EREP: the baseline is the existing PTAC or PTHP for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[413]

Remaining life of existing equipment is assumed to be 5 years.[414]

DEEMED MEASURE COST

TOS: The incremental capital cost for this equipment is estimated to be $84/ton.[415]

EREP: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used; if unknown, assume $1,047 per ton.[416]

The assumed deferred cost (after 5 years) of replacing existing equipment with new baseline unit is assumed to be $1,039 per ton.[417] This cost should be discounted to present value using the utilities' discount rate.

LOADSHAPE

Cooling RES
Heating RES

[413] Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.
[414] Standard assumption of one third of effective useful life.
[415] DEER 2008. This assumes that baseline shift from IECC 2012 to IECC 2015 carries the same incremental costs. Values should be verified during evaluation.
[416] Based on DCEO – IL PHA Efficient Living Program data.
[417] Based on subtracting TOS incremental cost from the DCEO data and incorporating inflation rate of 1.91%.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings for PTACs and PTHPs should be calculated using the following algorithms

Time of sale:

$$\Delta kWh = ((EFLH_{cool} * Capacity_{cool} * (1/SEER_{base} - 1/SEER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{base} - 1/HSFP_{ee})) / 1000)$$

Early replacement:[418]

ΔkWh for remaining life of existing unit:

$$= ((EFLH_{cool} * Capacity_{cool} * (1/SEER_{exist} - 1/SEER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{exist} - 1/HSFP_{ee})) / 1000)$$

ΔkWh for remaining measure life:

$$= ((EFLH_{cool} * Capacity_{cool} * (1/SEER_{base} - 1/SEER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{base} - 1/HSFP_{ee})) / 1000)$$

Where:

$Capacity_{heat}$ = Heating capacity of the unit in Btu/hr
= Actual

$EFLH_{heat}$ = Equivalent Full Load Hours for heating.
= Custom input if program or regional evaluation results are available, otherwise, per the following table:

Weather Basis (City based upon)	$EFLH_{heat}$[419]
St Louis	1,040

$HSPF_{ee}$ = HSPF rating of new equipment (kbtu/kwh)
= Actual installed

$HSPF_{base}$ =Heating System Performance Factor of baseline unit (kBtu/kWh)

Equipment Type	$HSPF_{base}$ (manufacture date prior to 1/1/2017)	$HSPF_{base}$ (manufacture date on or after 1/1/2017)
PTHP (Heating mode) Standard Sized	3.7 – (0.052 x $Capacity_{cool}$/1000) x 3.41	
PTHP (Heating mode) Non-Standard Size	2.9 – (0.026 x $Capacity_{cool}$/1000) x 3.41	

$HSPF_{exist}$ = Actual HSPF rating of existing equipment (kbtu/kwh). If unknown, assume:

Existing Equipment Type	$HSPF_{exist}$
Electric resistance heating (PTAC)	3.412[420]

[418] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a first year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[419] Base values reported in *All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems*, Cadmus, October 2015, Ameren. Illinois were adjusted to fit Missouri climate zones by a comparison of relative annual heating and cooling degree hours (base 65). See 3.4.8 EFLH 06022016.xlsx for derivation. FLH values are based on metering of multifamily units that were used as the primary heating source to the whole home, and in buildings that had received weatherization improvements. A DMSHP installed in a single-family home may be used more sporadically, especially if the DMSHP serves only a room, and buildings that have not been weatherized may require longer hours. Additional evaluation is recommended to refine the EFLH assumptions for the general population.

[420] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

Existing Equipment Type	$HSPF_{exist}$
PTHP	5.44[421]

$Capacity_{cool}$ = the cooling capacity of the ductless heat pump unit in Btu/hr.[422]
= Actual installed

$SEER_{ee}$ = SEER rating of new equipment (kbtu/kwh)
= Actual installed[423]

$SEER_{base}$ = Seasonal Energy Efficiency Ratio of baseline unit (kBtu/kWh). When using the formulas in the table below, convert the baseline EER to SEER using the EER conversion formula.[424]

Equipment Type	EER_{base} (manufacture date prior to 1/1/2017)	EER_{base} (manufacture date on or after 1/1/2017)
PTAC (Cooling mode) Standard Sized	13.8 – (0.3 x $Capacity_{cool}$/1000)	14.0 – (0.300 x $Capacity_{cool}$/1000)
PTAC (Cooling mode) Non-Standard Size	10.9 – (0.213 x $Capacity_{cool}$/1000)	
PTHP (Cooling mode) Standard Sized	14.0 – (0.300 x $Capacity_{cool}$/1000)	
PTHP (Cooling mode) Non-Standard Size	10.8 – (0.213 x $Capacity_{cool}$/1000)	

$SEER_{exist}$ = Actual SEER rating of existing equipment (kbtu/kwh). If unknown, assume:

Existing Cooling System	$SEER_{exist}$[425]
PTHP	7.2
PTAC	6.8

$EFLH_{cool}$ = Equivalent Full Load Hours for cooling.
= Custom input if program or regional evaluation results are available, otherwise, per the following table.[426]

Weather Basis (City based upon)	$EFLH_{cool}$
St Louis	617

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of sale:

$$\Delta kW = \Delta kWh_{cooling} * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = 0.0009474181

421 This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596 and applying to the average nameplate SEER rating of all early replacement qualifying equipment in Ameren PY2003-PY2004. This estimation methodology appears to provide a result within 10% of actual HSPF.

422 1 Ton = 12 kBtu/hr.

423 Note that if only an EER rating is available, use the following conversion equation; EER_base = (1.12 - √(1.2544 - 0.08 * EER)) / 0.. From Wassmer, M. (2003), "A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations," (Masters thesis), University of Colorado at Boulder.

424 Ibid.

425 ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8.

426 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

NATURAL GAS ENERGY SAVINGS
N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

3.4.11 Room Air Conditioner

DESCRIPTION

This measure relates to the purchase and installation of a room air conditioning unit that meets the ENERGY STAR® minimum qualifying efficiency specifications, in place of a baseline unit meeting minimum federal standard efficiency ratings presented below:[427]

Product Class (Btu/H)	Federal Standard CEERbase, with louvered sides, without reverse cycle[428]	Federal Standard CEERbase, without louvered sides, without reverse cycle	ENERGY STAR® CEERee, with louvered sides	ENERGY STAR® CEERee, without louvered sides
< 6,000	12.1	11.0	11.5	10.5
6,000 to 7,999			11.4	10.1
8,000 to 10,999	12.0	10.6		10.0
11,000 to 13,999		10.5	11.2	9.7
14,000 to 19,999	11.8	10.5	9.8	9.8
20,000-27,999	10.3	10.2		
>=28,000	9.9	10.3	9.5	

Casement	Federal Standard CEERbase	ENERGY STAR® CEERee
Casement-only	10.5	10.0
Casement-slider	11.4	10.8

Reverse Cycle - Product Class (Btu/H)	Federal Standard CEERbase, with louvered sides	Federal Standard CEERbase, without louvered sides[429]	ENERGY STAR® CEERee, with louvered sides[430]	ENERGY STAR® CEERee, without louvered sides
< 14,000	N/A	10.2	N/A	9.7
>= 14,000	N/A	9.6	N/A	9.1
< 20,000	10.8	N/A	10.3	N/A
>= 20,000	10.2	N/A	9.7	N/A

This measure was developed to be applicable to the following program type: TOS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the new room air conditioning unit must meet the ENERGY STAR® efficiency standards presented above.

DEFINITION OF BASELINE EQUIPMENT

The baseline assumption is a new room air conditioning unit that meets the current minimum federal efficiency standards presented above.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 9 years.[431]

DEEMED MEASURE COST

The incremental cost for this measure is assumed to be $20 for an ENERGY STAR® unit.[432]

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[427] Side louvers that extend from a room air conditioner model in order to position the unit in a window. A model without louvered sides is placed in a built-in wall sleeve and are commonly referred to as "through-the-wall" or "built-in" models.
Casement-only refers to a room air conditioner designed for mounting in a casement window of a specific size.
Casement-slider refers to a room air conditioner with an encased assembly designed for mounting in a sliding or casement window of a specific size. Reverse cycle refers to the heating function found in certain room air conditioner models. https://www.energystar.gov/products/heating_cooling/air_conditioning_room/key_product_criteria

[428] Federal standard air conditioner baselines. https://ees.lbl.gov/product/room-air-conditioners.

[429] Federal standard air conditioner baselines. https://ees.lbl.gov/product/room-air-conditioners.

[430] EnergyStar® version 4.0 Room Air Conditioner Program Requirements. https://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Version%204.0%20Room%20Air% 20Conditioners%20Program%20Requirements.pdf.

[431] ENERGY STAR® Room Air Conditioner Savings Calculator: http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=AC.

[432] Cost from RS Means 2018.

LOADSHAPE
Cooling RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \frac{\left(FLH_{RoomAC} * Btu/H * \left(\frac{1}{CEER_{base}} - \frac{1}{CEER_{ee}}\right)\right)}{1{,}000}$$

Where:

FLH_{RoomAC} = Full Load Hours of room air conditioning unit:

Weather Basis (City based upon)	Hours[433]
St Louis, MO	860 for primary use and 556 for secondary use

Btu/H = Size of unit
= Actual. If unknown assume 8500 Btu/hr [434]

$CEER_{base}$ = Efficiency of baseline unit
= As provided in tables above

$CEER_{ee}$ = Efficiency of ENERGY STAR® unit
= Actual. If unknown assume minimum qualifying standard as provided in tables above

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer Peak Coincidence Factor for measure
= 0.0009474181[435]

NATURAL GAS SAVINGS
N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

433 Primary is based upon Ameren Missouri PY2013 CoolSavers Evaluation data, Secondary is based upon Ameren Missouri Efficient Products PY2016 Evaluation.
434 Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008.
435 Based on Ameren Missouri 2016 loadshape for residential cooling end-use.

3.4.12 Ground Source Heat Pump

DESCRIPTION
A heat pump provides heating or cooling by moving heat between indoor and the ground.

This measure characterizes:

1. TOS: The installation of a new residential sized ground source heat pump. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.
2. EREP: The early removal of functioning electric heating and cooling systems from service, prior to its natural end of life, and replacement with a new high efficiency ground source heat pump unit. To qualify as early replacement, the existing unit must be operational when replaced. If the SEER of the existing unit is known and the baseline SEER is the actual SEER value of the unit replaced and if unknown use assumptions in the variable list below ($SEER_{exist}$ and $HSPF_{exist}$). If the operational status of the existing unit is unknown, use TOS assumptions.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT
A new residential sized ground source heat pump with specifications to be determined by program.

DEFINITION OF BASELINE EQUIPMENT
The baseline for the TOS measure is federal standard efficiency level as of: 3.3 COP and 14.1 EER when replacing an existing ground source heat pump, 14 SEER and 8.2HSPF when replacing an existing air source heat pump, and 13 SEER and 3.41 HSPF when replacing a central air conditioner and electric resistance heating.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT
The expected measure life is assumed to be 18 years.

For early replacement, the remaining life of existing equipment is assumed to be 6 years for GSHP, ASHP and CAC and 18 years for electric resistance.

DEEMED MEASURE COST
TOS: The incremental capital cost for this measure is dependent on the efficiency and capacity of the new unit.[436]

Efficiency (EER)	Cost (including labor) per measure
GSHP - EER 23 - replace electric furnace / CAC	$4,717
GSHP EER 23 Replace at Fail GSHP	$3,200

EREP: The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume the following (note these costs are per ton of unit capacity):[437]

Efficiency (EER)	Cost (including labor) per measure
GSHP - EER 23 - replace electric furnace / CAC Early Replacement	$5,250
GSHP EER 23	$4,859

LOADSHAPE
Cooling RES
Heating RES

[436] Cost based upon Ameren Missouri MEEIA 2016-18 TRM effective January 1, 2018.
[437] Ibid.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

TOS:

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/EER_{base} - 1/EER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{base} - 1/HSPF_{ee})) / 1000)] * ISR$$

EREP:[438]

ΔkWh for remaining life of existing unit (1st 6 years for replacing an ASHP or GSHP, 18 years for replacing electric resistance):

$$= [((EFLH_{cool} * Capacity_{cool} * (1/EER_{exist} - 1/EER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{exist} - 1/HSPF_{ee})) / 1000)] * ISR$$

ΔkWh for remaining measure life (next 12 years if replacing an ASHP or GSHP):

$$= [((EFLH_{cool} * Capacity_{cool} * (1/EER_{base} - 1/EER_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF_{base} - 1/HSPF_{ee})) / 1000)] * ISR$$

Where:

$EFLH_{cool}$ = Equivalent full load hours of air conditioning:[439]

Weather Basis (City based upon)	$EFLH_{cool}$ (Hours)
St Louis, MO	869

$Capacity_{cool}$ = Cooling capacity of air source heat pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)

EER_{exist} = Seasonal Energy Efficiency Ratio of existing cooling system (kBtu/kWh)
= Use actual SEER rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.[440] If age is unknown, use 12 years.
= EER * $(1-1.44\%)^{Age}$

Existing Cooling System	$SEER_{exist}$[441]
Air Source Heat Pump	7.2
Central AC	6.54
No central cooling[442]	Let '$1/SEER_{exist}$' = 0

EER_{base} = Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/kWh)
= 14[443]

EER_{ee} = Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/kWh)
= Actual

$EFLH_{heat}$ = Equivalent full load hours of heating

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438 The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a first year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

439 Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren territory suggesting an appropriate EFLH of 869.The other climate region values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

440 Based on IL TRM V8.0, which bases justification for degradation factors on page 21 of 'AIC HVAC Metering Study Memo FINAL 2_28_2018'. Default of 12 years based on the remaining measure life of the equipment.

441 Ameren Missouri HVAC Program Evaluation PY2018 - Operating would have the manufacturers recommendations of 10-12 EER and 2.4-2.8 COP. Use of 12 EER and 2.8 COP. is conservative.

442 If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.

443 Based on minimum federal standard effective 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.

= Dependent on location:[444]

Weather Basis (City based upon)	$EFLH_{heat}$ (Hours)
St Louis, MO	1496

$Capacity_{heat}$ = Heating Capacity of Air Source Heat Pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)

$HSPF_{exist}$ = Heating System Performance Factor of existing heating system (kBtu/kWh)
= Use actual HSPF rating where it is possible to measure or reasonably estimate. If not available use:

Existing Heating System	$HSPF_{exist}$
Air Source Heat Pump	5.44[445]
Electric Resistance	3.41[446]

$HSPF_{base}$ = Heating System Performance Factor of baseline Air Source Heat Pump (kBtu/kWh)
= 8.2[447]

$HSFP_{ee}$ = Heating System Performance Factor of efficient Air Source Heat Pump (kBtu/kWh)

ISR = In Service Rate = 100%[448]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

TOS:

$$\Delta kW = \Delta kWh_{cooling} * CF$$

Where:

ΔkWh = Energy Savings as calculated above
CF = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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[444] Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls). The other weather basis values are calculated using the relative climate normals HDD data with a base temp ratio of 60°F.

[445] This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596, and applying to the average nameplate SEER rating of all early replacement qualifying equipment in Ameren PY2003-PY2004. This estimation methodology appears to provide a result within 10% of actual HSPF.

[446] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

[447] Based on minimum federal standard effective 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.

[448] Ameren Missouri HVAC Evaluation: PY2020.

3.5 Lighting

3.5.1 LED Screw Based Omnidirectional Bulb

DESCRIPTION

This measure provides savings assumptions for LED screw-based omnidirectional (e.g., A-Type) lamps installed in a known location (i.e., residential and in-unit interior or exterior) or, if the implementation strategy does not allow for the installation location to be known (e.g., an upstream retail program or efficiency kit), an unknown residential location. For upstream programs, utilities should develop an assumption of the Residential v Commercial split and apply the relevant assumptions to each portion.

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) requires all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than standard incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012, followed by restrictions on 75W lamps in 2013 and 60W and 40W lamps in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations required that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in effect making the baseline equivalent to a current day CFL. In 2019, the Department of Energy issued a final determination and clarified that a) the EISA backstop provision had not been triggered and therefore b) the efficiency standard would not change in 2020. In May 2022, DOE reversed this decision by issuing a final rule that expanded the General Service Lamp (GSL) and General Service Incandescent Lamp (GSIL) definitions and reinstated the 45 lumen per watt backstop provision with phased enforcement between January 2023 and July 2023.[449]

This measure was developed to be applicable to the following program types: TOS, NC, and RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this measure to apply, new lamps must be ENERGY STAR® labeled based upon the ENERGY STAR® specification v2.0 which became effective on 1/2/2017 (https://www.energystar.gov/sites/default/files/Luminaires%20V2%200%20Final.pdf).

Qualification could also be based on the Design Light Consortium's qualified product list.[450]

DEFINITION OF BASELINE EQUIPMENT

Starting August 1, 2023, the EISA backstop provision became effective, limiting the sale, manufacture, and import of non-compliant lamps. Therefore, the baseline condition for this measure is a reflection of the 2022 DOE final rule reinstating the 45 lumen per watt backstop provisions for all GSL and GSILs between 310 and 3,300 lumens. All other lamps, i.e., those below 310 lumens and above 3,300 lumens, the baseline condition is a reflection of products available in the market and standards agreed upon in practice.

Prior to August 1, 2023, the baseline condition for this measure is assumed to be an EISA-qualified halogen or CFL lamp.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is 19 years for residential applications and 6 years for non-residential applications.[451]

DEEMED MEASURE COST

The deemed measures cost for a LED screw based omnidirectional bulb is $1.45 per bulb.[452]

LOADSHAPE

Lighting RES
Lighting BUS

Algorithm

[449] DOE 87 FR 27439
[450] https://www.designlights.org/QPL.
[451] Measure life is estimated based on the ratio of average equipment specifications for lifetime hours to the estimated annual operating hours. EULs of 19 years for residential and 6 years for non-residential are based on average rated lifetime for 2021 program measures (through 8/3/2021) divided by 995 hours for residential settings and by 3,351 for non-residential settings.
[452] Based on IL TRM V11.0, Section 5.5.8.

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \Delta kWh_{RES} + \Delta kWh_{NRES}$$

$$\Delta kWh_{RES} = (Watt_{Base} - Watt_{EE}) * \%RES * ISR * (1 - LKG) * Hours_{RES} * WHF_{RES} / 1{,}000$$

$$\Delta kWh_{NRES} = (Watt_{Base} - Watt_{EE}) * (1 - \%RES) * ISR * (1 - LKG) * Hours_{NRES} * WHF_{NRES} / 1{,}000$$

Where:

$Watts_{Base}$ = Based on lumens of LED bulb installed. If lumens of LED bulb are unknown, refer to table below.
$Watts_{EE}$ = Actual wattage of LED purchased / installed - If unknown, use default provided below:[453]

Lower Lumen Range	Upper Lumen Range	EISA-qualified Halogen or CFL (Apply through 7/31/2023)			EISA Backstop (apply starting 8/1/2023)		
		$Watts_{Base}$	$Watts_{EE}$ LED	Delta Watts	$Watts_{Base}$	$Watts_{EE}$ LED	Delta Watts
250	309	25	4.0	21	25	4.0	21
310	749	29	6.7	22.3	11.8	6.7	5.1
750	1,049	43	10.1	32.9	20	10.1	9.9
1,050	1,489	53	12.8	40.2	28.2	12.8	15.4
1,490	2,600	72	17.4	54.6	45.4	17.4	28
2,601	3,300	150	43.1	106.9	65.6	43.1	22.5
3,301	3,999	200	53.8	146.2	200	53.8	146.2
4,000	6,000	300	76.9	223.1	300	76.9	223.1

%RES = percentage of bulbs sold to residential customers
= 100% for Online Store and 96% for Upstream Lighting, or 96% if unknown[454]

LKG = leakage rate (program bulbs installed outside Ameren Missouri's service area)

Program	Channel or Subgroup	Leakage	Utility Adjustment (1-Leakage)
Retail (Time of Sale)[455]	Overall Average	3.98%	96.02%
	Online Store	0%	100%
	Upstream	4%	96%
Efficiency Kit (School)[456]	-	28%	72%
Efficiency Kit (MF)[457]	-	0%	100%
Appliance Recycling[458]	-	0%	100%
Low Income[459]	-	0%	100%
MFMR[460]	-	0%	100%

453 $Watts_{EE}$ defaults are based upon the average available ENERGY STAR® product, accessed 06/18/2015. For any lumen range where there is no ENERGY STAR® product currently available, $Watts_{EE}$ is based upon the ENERGY STAR® minimum luminous efficacy (55Lm/W for lamps with rated wattages less than 15W and 65 Lm/W for lamps with rated wattages ≥ 15 watts) for the mid-point of the lumen range. See calculation at "cerified-light-bulbs-2015-06-18.xlsx." These assumptions should be reviewed regularly to ensure they represent the available product.
454 Ameren Missouri Lighting Evaluation: PY2022. 96% is the weighted average for bulbs sold through the Online Store and Upstream Programs based on evaluation results and the distribution of bulbs in the PY2022 program.
455 Ameren Missouri Lighting Evaluation: PY2019. 3.98% is the weighted average for bulbs sold through the Online Store and Upstream Programs based on evaluation results and the distribution of bulbs in the PY2019 program.
456 Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2019 (Table 7-9)
457 Assumed based on program design.
458 Ameren Missouri Appliance Recycling Evaluation PY2019 (Appendix Table 56)
459 Assumed based on program design.
460 Ibid.

ISR = In Service Rate, the percentage of units rebated that are actually in service

Program	Channel or Subgroup	Discounted In Service Rate (ISR)
Retail (Time of Sale)[461]	Overall Program Average	88.44%
	Online Store - Standard	80.00%
	Online Store - Reflector	80.00%
	Online Store - Specialty	84.00%
	Upstream - Standard	88.00%
	Upstream - Reflector	90.00%
	Upstream - Specialty	93.00%
Direct Install (MFLI) [462]	-	98.2%
Efficiency Kit (School)[463]	-	92%
Efficiency Kit (MF)[464]	-	100%
Appliance Recycling[465]	-	88%
Low Income Kits	-	90%
Pay As You Save[466]	-	87%

$Hours_{RES}$ = Average hours of use per year for bulbs in residential homes. Use custom value or table below.
$Hours_{NRES}$ = Average hours of use per year for bulbs in non-residential buildings. Use custom value or table below.

Program	HOU Res	HOU NRes
Residential	995.18[467]	3,351[468]
Efficient Kits	995.18	N/A
Income Eligible RES	674.18[469]	7,321[469]
MFMR	693.50[470]	3,351[471]

$WHFe_{RES}$ = Waste Heat Factor for energy to account for the impact from reducing waste heat from efficient lighting on electric cooling and heating loads in residential homes.
= 0.99 if unknown[472]

$WHFe_{NRES}$ = Waste Heat Factor for energy to account for the impact from reducing waste heat from efficient lighting on electric cooling and heating loads in non-residential spaces.
= If unknown assume 1.1 or 0.97 for Income Eligible.[473]

$WHFe_{Heat}$ = Waste Heat Factor for energy to account for electric heating increase from reducing waste heat from efficient lighting (if fossil fuel heating, see calculation of heating penalty in that section).
= 1 - ((HF / ηHeat) * %ElecHeat).
= If unknown assume 0.88[474]

[461] Ameren Missouri Lighting Evaluation: PY2019. 88.61% is the weighted average for bulbs sold through the Online Store and Upstream Programs based on evaluation results and the distribution of bulbs in the PY2019 program.
[462] Ameren Missouri Community Savers Evaluation: PY2018.
[463] Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2019 (Table 7-9).
[464] Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2018.
[465] Ameren Missouri Appliance Recycling Evaluation PY2019 (Table 9-9; cumulative value)
[466] Ameren Missouri Pay As You Save (PAYS®) Evaluation: PY2022 Participant Survey
[467] Ameren Missouri Lighting Evaluation PY2018.
[468] Ameren Missouri TRM, Volume 2, C&I Lighting Hours of Use and Waste Heat Factors by Building type; 3,351 is the average C&I value.
[469] Ameren Missouri Community Savers Evaluation PY2018 workpapers- Weighted Avg. HOU from ADM workpapers.
[470] ADM 2017 Community Savers EM&V
[471] Ameren Missouri TRM, Volume 2, C&I Lighting Hours of Use and Waste Heat Factors by Building type; 3,351 is the average C&I value.
[472] Ameren Missouri PY2014 Evaluation.
[473] Ameren Missouri Community Savers Evaluation PY2018 workpapers. Weighted Avg. calculated from ADM workpapers.
[474] Calculated using defaults: 1-((0.53/1.57) * 0.35) = 0.88.

Where:

HF = Heating Factor or percentage of light savings that must now be heated
= 53%[475] for interior or unknown location
= 0% for exterior or unheated location

$\eta Heat_{Electric}$ = Efficiency in COP of Heating equipment
= Actual - If not available, use:[476]

System Type	Age of Equipment	HSPF Estimate	ηHeat (COP Estimate)
Heat Pump	Before 2006	6.8	2.00
	2006-2014	7.7	2.26
	2015 and after	8.2	2.40
Resistance	N/A	N/A	1.00
Unknown	N/A	N/A	1.57[477]

%ElecHeat = Percentage of heating savings assumed to be electric

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	35%[478]

$WHFe_{Cool}$ = Waste Heat Factor for energy to account for cooling savings from reducing waste heat from efficient lighting

Bulb Location	$WHFe_{Cool}$
Building with cooling	1.12[479]
Building without cooling or exterior	1.0
Unknown	1.11[480]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001492529 for residential bulbs and 0.0001899635 for nonresidential bulbs

[475] This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). These results were judged to be equally applicable to Missouri.

[476] These default system efficiencies are based on the applicable minimum federal standards. In 2006 and 2015, the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[477] Calculation assumes 50% heat pump and 50% resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls." Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% 2006-2014.

[478] Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

[479] The value is estimated at 1.12 (calculated as 1 + (0.34 / 2.8)), and it is based on cooling loads decreasing by 34% of the lighting savings (average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington)). The estimate also assumes typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm ($-0.02 * SEER^2$) + (1.12 * SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). Results of the Iowa study are assumed to be applicable to Missouri.

[480] The value is estimated at 1.11 (calculated as 1 + (0.91*(0.34 / 2.8)), which is based on assumption that 91% of homes have central cooling (based on 2009 Residential Energy Consumption Survey, see "HC7.9 Air Conditioning in Midwest Region.xls").

NATURAL GAS SAVINGS

Heating Penalty for Natural Gas heated homes:[481]

$$\Delta Therms = -\frac{\frac{Watts_{Base} - Watts_{EE}}{1{,}000} * ISR * Hours * HF * 0.03412}{\eta \text{Heat}} * \%\text{GasHeat}$$

Where:

HF = Heating Factor or percentage of light savings that must now be heated

= 53%[482] for interior or unknown location

= 0% for exterior or unheated location

0.03412 =Converts kWh to therms

$\eta Heat_{Gas}$ = Efficiency of heating system

= 71%[483]

%GasHeat = Percentage of heating savings assumed to be Natural Gas

Heating fuel	%GasHeat
Electric	0%
Natural Gas	100%
Unknown	65%[484]

MEASURE CODE:

[481] Negative value because this is an increase in heating consumption due to the efficient lighting.

[482] This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). Results of the Iowa study are judged to be equally applicable to Missouri.

[483] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference "HC6.9 Space Heating in Midwest Region.xls." In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the state. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

[484] Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

3.5.2 LED Specialty Lamp

DESCRIPTION

This measure provides savings assumptions for LED directional, decorative, and globe lamps when the LED is installed in a known location (i.e., residential and in-unit interior or exterior) or, if the implementation strategy does not allow for the installation location to be known (e.g., an upstream retail program or efficiency kit), an unknown residential location. For upstream programs, utilities should develop an assumption of the Residential v Nonresidential split and apply the relevant assumptions to each portion.

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) requires all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than standard incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012, followed by restrictions on 75W lamps in 2013 and 60W and 40W lamps in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations required that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in effect making the baseline equivalent to a current day CFL. In 2019, the Department of Energy issued a final determination and clarified that a) the EISA backstop provision had not been triggered and therefore b) the efficiency standard would not change in 2020. In May 2022, DOE reversed this decision by issuing a final rule that expanded the General Service Lamp (GSL) and General Service Incandescent Lamp (GSIL) definitions and reinstated the 45 lumen per watt backstop provision with phased enforcement between January 2023 and July 2023.[485]

This measure was developed to be applicable to the following program types: TOS, NC, and RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, new lamps must be ENERGY STAR® labeled based upon the ENERGY STAR® specification v2.0 which became effective on 1/2/2017 https://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V2_0%20Revised%20AUG-2016.pdf). Qualification could also be based on the Design Light Consortium's qualified product list.[486]

DEFINITION OF BASELINE EQUIPMENT

Starting August 1, 2023, the EISA backstop provision became effective, limiting the sale, manufacture, and import of non-compliant lamps. Therefore, the baseline condition for this measure is a reflection of the 2022 DOE final rule reinstating the 45 lumen per watt backstop provisions for all GSL and GSILs between 310 and 3,300 lumens. All other lamps, i.e., those below 310 lumens and above 3,300 lumens, the baseline condition is a reflection of products available in the market and standards agreed upon in practice.

Prior to August 1, 2023, the baseline condition for this measure is assumed to be an EISA-qualified halogen lamp.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is 19 years for residential applications and 6 years for non-residential applications.[487]

DEEMED MEASURE COST

The deemed measures cost for a specialty LED is $1.66 per lamp.[488] In the case of direct install programs or lighting included in efficient kits, the actual cost of the measure should be used.

LOADSHAPE

Lighting RES
Lighting BUS

Algorithm

[485] DOE 87 FR 27439.
[486] https://www.designlights.org/QPL.
[487] Measure life is estimated based on the ratio of average equipment specifications for lifetime hours to the estimated annual operating hours. EULs of 19 years for residential and 6 years for non-residential are based on average rated lifetime for 2021 program measures (through 8/3/2021) divided by 995 hours for residential settings and by 3,351 for non-residential settings.
[488] Based on IL TRM V11.0, Section 5.5.6.

Calculation of Savings

Electric Energy Savings

$$\Delta kWh_{RES} = (Watt_{Base} - Watt_{EE}) * \%RES * ISR * (1 - LKG) * Hours_{RES} * WHF_{RES} /1{,}000$$

$$\Delta kWh_{NRES} = (Watt_{Base} - Watt_{EE}) * (1 - \%RES) * ISR * (1 - LKG) * Hours_{NRES} * Days * WHF_{NRES}/1{,}000$$

Where:

$Watts_{Base}$ = Based on bulb type and lumens of LED bulb installed. See tables below.

$Watts_{EE}$ = Actual wattage of LED purchased / installed - If unknown, use default provided below:[489]

Baseline and Efficienct Wattages through July 31, 2023, assuming a EISA-compliant halogen baseline

Bulb Type	Lower Lumen Range	Upper Lumen Range	$Watts_{Base}$	$Watts_{EE}$	Delta Watts
Directional	250	349	25	5.6	19.4
	350	399	35	6.3	28.7
	400	599	40	7.5	32.5
	600	749	60	9.7	50.3
	750	999	75	12.7	62.3
	1000	1250	100	16.2	83.8
Decorative	70	89	10	1.8	8.2
	90	149	15	2.7	12.3
	150	299	25	3.2	21.8
	300	499	40	4.7	35.3
	500	699	60	6.9	53.1
Globe	250	349	25	4.1	20.9
	350	499	40	5.9	34.1
	500	574	60	7.6	52.4
	575	649	75	13.6	61.4
	650	1099	100	17.5	82.5
	1100	1300	150	13.0	137.0

Baseline and Efficienct Wattages after August 1, 2023, under the EISA backstop provision of 45 lumens/watt

Bulb Type	Lower Lumen Range	Upper Lumen Range	$Watts_{Base}$	$Watts_{EE}$	Delta Watts
Directional	250	309	25	5.6	19.4
	310	399	7.9	6.3	1.6
	400	599	11.1	7.5	3.6
	600	749	15	9.7	5.3
	750	999	19.4	12.7	6.7
	1000	1250	25	16.2	8.8
Decorative	70	89	10	1.8	8.2
	90	149	15	2.7	12.3
	150	309	25	3.2	21.8
	310	499	40	4.7	35.3

[489] $Watts_{EE}$ defaults are based upon the average available ENERGY STAR® product, accessed 06/18/2015. For any lumen range where there is no ENERGY STAR® product currently available, $Watts_{EE}$ is based upon the ENERGY STAR® minimum luminous efficacy (directional; 40Lm/W for lamps with rated wattages less than 20Wand 50 Lm/W for lamps with rated wattages ≥ 20 watts. decorative and globe; 45Lm/W for lamps with rated wattages less than 15W, 50lm/W for lamps ≥15 and <25W, 60 Lm/W for lamps with rated wattages ≥ 25 watts) for the mid-point of the lumen range. See calculation at “cerified-light-bulbs-2015-06-18.xlsx.” These assumptions should be reviewed regularly to ensure they represent the available product.

Bulb Type	Lower Lumen Range	Upper Lumen Range	$Watts_{Base}$	$Watts_{EE}$	Delta Watts
	500	699	60	6.9	53.1
Globe	250	309	25	4.1	20.9
	310	499	9.0	5.9	3.1
	500	574	11.9	7.6	4.3
	575	649	13.6	13.6	0.0
	650	1099	19.4	17.5	1.9
	1100	1300	26.7	13.0	13.7

%RES = percentage of bulbs sold to residential customers
= 100% for Online Store and 96% for Upstream Lighting or 96% if unknown[490]

LKG = leakage rate (program bulbs installed outside Ameren Missouri's service area)
= 0% for Online Store and 4% for Upstream Lighting or 3.98% if unknown [490]

ISR = In Service Rate, the percentage of units rebated that are actually in service – see table below

$Hours_{RES}$ = Average hours of use per year
= Custom, or if unknown assume 728[491] for interior or 1,314 for exterior, or 776 if location is not known.

$Hours_{NRES}$ = 3,351

$WHFe_{Heat}$ = Waste Heat Factor for energy to account for electric heating increase from reducing waste heat from efficient lighting (if fossil fuel heating – see calculation of heating penalty in that section).
= 1 - ((HF / ηHeat) * %ElecHeat)
If unknown assume 0.88[492]

HF = Heating Factor or percentage of light savings that must now be heated
= 53%[493] for interior or unknown location
= 0% for exterior or unheated location

$\eta Heat_{Electric}$ = Efficiency in COP of Heating equipment
= Actual - If not available, use values in table below[494]

%ElecHeat = Percentage of heating savings assumed to be electric

$WHFe_{Cool}$ = Waste Heat Factor for energy to account for cooling savings from reducing waste heat from efficient lighting

490 Ameren Missouri Lighting Evaluation: PY2022. 96% is the weighted average for bulbs sold through the Online Store and Upstream Programs based on evaluation results and the distribution of bulbs in the PY2022 program.

491 Ameren Missouri Lighting Impact and Process Evaluation: Program Year 2015. Average daily HOU for efficient bulbs is listed as 3.6 for outside bulbs and a weighted (by inventory) average of 1.99 for inside spaces. Unknown location is weighted average (by inventory) of all bulbs. See 'MO Lamp Hours.xls' for calculations.

492 Calculated using defaults: 1-((0.53/1.57) * 0.35) = 0.88.

493 This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). Results of the Iowa study were judged to be equally applicable to Missouri.

494 These default system efficiencies are based on the applicable minimum federal standards. In 2006 and 2015 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

Program	Channel or Subgroup	Discounted In Service Rate (ISR)
Retail (Time of Sale)[495]	Overall Program Average	88.44%
	Online Store - Reflector	80.00%
	Online Store - Specialty	84.00%
	Upstream - Reflector	90.00%
	Upstream - Specialty	93.00%
Direct Install (MFLI)[496]		98.2%
Efficiency Kit (School)[497]		90%
Efficiency Kit (Multi-Family)[497]		100%
Pay As You Save[498]		87%

System Type	Age of Equipment	HSPF Estimate	ηHeat (COP Estimate)
Heat Pump	Before 2006	6.8	2.00
	2006-2014	7.7	2.26
	2015 and after	8.2	2.40
Resistance	N/A	N/A	1.00
Unknown	N/A	N/A	1.57 [499]

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	35%[500]

Bulb Location	$WHFe_{Cool}$
Building with cooling	1.12[501]
Building without cooling or exterior	1.0
Unknown	1.11[502]

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001492529 for Lighting RES (Residential)
= 0.0001899635 for Lighting BUS (Business)

[495] Ameren Missouri Lighting Evaluation: PY2019. 3.98% is the weighted average for bulbs sold through the Online Store and Upstream Programs based on evaluation results and the distribution of bulbs in the PY2019 program.
[496] Ameren Missouri Community Savers Program Evaluation: PY2018.
[497] Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2018.
[498] Ameren Missouri Pay As You Save (PAYS®) Evaluation: PY2022 Participant Survey
[499] Calculation assumes 50% heat pump and 50% resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Regionals." Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% 2006-2014.
[500] Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.
[501] The value is estimated at 1.12 (calculated as 1 + (0.34 / 2.8)), is based on cooling loads decreasing by 34% of the lighting savings (average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington)). The estimate also assumes typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * $SEER^2$) + (1.12 * SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). Results of the Iowa study were assumed to be applicable to Missouri.
[502] The value is estimated at 1.11 (calculated as 1 + (0.91*(0.34 / 2.8)). Based on assumption that 91% of homes have central cooling (based on 2009 Residential Energy Consumption Survey, see “HC7.9 Air Conditioning in Midwest Region.xls”).

NATURAL GAS SAVINGS

Heating Penalty for Natural Gas heated home:s[503]

$$\Delta Therms = -\frac{\frac{Watts_{Base} - Watts_{EE}}{1{,}000} * ISR * Hours * HF * 0.03412}{\eta Heat} * \%GasHeat$$

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 53%[504] for interior or unknown location

= 0% for exterior or unheated location

0.03412 =Converts kWh to therms

$\eta Heat_{Gas}$ = Efficiency of heating system

=71%[505]

%GasHeat = Percentage of homes with gas heat

Heating fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65%[506]

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

MEASURE CODE:

[503] Negative value because this is an increase in heating consumption due to the efficient lighting.

[504] This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Des Moines, Mason City, and Burlington, Iowa. Results of the Iowa study were judged to be equally applicable to Missouri.

[505] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference "HC6.9 Space Heating in Midwest Region.xls." In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

[506] Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

3.6 Motors

3.6.1 High Efficiency Pool Pumps

DESCRIPTION

Residential outdoor pool pumps can be single speed, two/multi-speed or variable speed. A federal standard (82 FR 5650) effective July 19, 2021 effectively requires pool pumps to be at least two speed.

Single speed pumps are often oversized, and run frequently at constant flow regardless of load. Single speed pool pumps require that the motor be sized for the task that requires the highest speed. As such, energy is wasted performing low speed tasks at high speed. Two- speed and variable speed pool pumps reduce speed when less flow is required, such as when filtering is needed but not cleaning, and have timers that encourage programming for fewer on-hours. Variable speed pool pumps use advanced motor technologies to achieve efficiency ratings of 90% while the average single speed pump will have efficiency ratings between 30% and 70%.[507] This measure is the characterization of the purchasing and installing of a new ENERGY STAR variable speed residential pool pump motor in place of a new baseline pump meeting the federal standard for Time of Sale and New Construction, or the early replacement of a standard single speed motor of equivalent horsepower.

This measure was developed to be applicable to the following program types: TOS, NC, and RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The high efficiency equipment is an ENERGY STAR® variable speed residential pool pump for in-ground pools.

DEFINITION OF BASELINE EQUIPMENT

For TOS and NC, the baseline equipment is a two speed residential pool pump meeting the Federal Standard, effective July 19, 2021 provided below:

Size Class	Baseline (Effective 7/19/2021)
Extra Small (hhp ≤ 0.13)	WEF ≥ 5.55
Small (hhp > 0.13 and < 0.711)	WEF ≥ -1.30 x ln (hhp) + 2.90
Standard Size (hhp ≥ 0.711)	WEF ≥ -2.30 x ln (hhp) + 6.59

For early replacement, the baseline is the existing single speed residential pool pump.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life for a two speed or variable speed pool pump is 10 years.[508]

DEEMED MEASURE COST

For TOS and NC, the incremental cost is estimated $314 for a variable speed motor.[509]

For early replacement, the actual cost of the measure should be used; if actual is unknown, use $549. [510]

LOADSHAPE

Pool Spa RES

Algorithm

CALCULATION OF ENERGY SAVINGS

[507] U.S. DOE, 2012. Measure Guideline: Replacing Single-Speed Pool Pumps with Variable Speed Pumps for Energy Savings. Report No. DOE/GO-102012-3534.
[508] The CEE Efficient Residential Swimming Pool Initiative, p18, indicates that the average motor life for pools in use year round is 5-7 years. For pools in use for under a third of a year, you would expect the lifetime to be higher so 10 years is selected as an assumption. This is consistent with DEER, 2014 and the ENERGY STAR® Pool Pump Calculator assumptions.
[509] ENERGY STAR® Pool Pump Calculator, using the difference between the two speed pool pump and variable speed pool pump incremental costs.
[510] ENERGY STAR® Pool Pump Calculator, estimated cost for a variable speed pool pump.

Electric Energy Savings[511]

For TOS and NC:

$$\Delta kWh = \left(Gallons * Turnovers * \left(\frac{1}{WEF_{base}} - \frac{1}{WEF_{ee}}\right) * Days\right) / 1{,}000 * ISR$$

For Early Replacement:

$$\Delta kWh = \left(Gallons * Turnovers * \left(\frac{1}{EF_{exist}} - \frac{1}{WEF_{ee}}\right) * Days\right) / 1{,}000 * ISR$$

Where:

Gallons = Capacity of the pool. Use actual, or if unknown assume 22,000.[512]

Turnovers = Desired number of pool water turnovers per day
= 2[513]

WEF_{base} = Weighted Energy Factor of baseline pump (gal/Wh)
= 4.6[514]

WEF_{ee} = Weighted Energy Factor of installed ENERGY STAR pump (gal/Wh)
= 6.31[515]

EF_{exist} = Energy Factor of existing single speed pump (gal/Wh)
= 2.3[516]

Days = Days per year of operation
= 122[517]

1,000 = Conversion factor from Wh to kWh

ISR = In Service Rate[518]

Summer Coincident Peak Demand Savings

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0002354459

NATURAL GAS SAVINGS
N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

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511 The methodology followed is consistent with the most recent version of the 2020 ENERGY STAR calculator (Pool_Pump_Calculator_2020.05.05_FINAL.xlsx), however this has not been updated to account for the new federal standard.
512 Consistent with assumption in the 2020 ENERGY STAR calculator (Pool_Pump_Calculator_2020.05.05_FINAL.xlsx).
513 Ibid.
514 Consistent with IL-TRM V10.0 assumption, which is based on applying the federal standard specifications to the average Curve-C rated hydraulic horsepower (hhp) from the ENERGY STAR Qualified Products List, accessed 3/31/2021.
515 Consistent with IL-TRM V10.0 assumption, which is based on applying the ENERGY STAR specifications to the average Curve-C rated hydraulic horsepower (hhp) from the ENERGY STAR Qualified Products List, accessed 3/31/2021.
516 Consistent with assumption in the 2020 ENERGY STAR calculator, assuming 1.5 HP pump (Pool_Pump_Calculator_2020.05.05_FINAL.xlsx).
517 Consistent with assumption in the 2020 ENERGY STAR calculator (Pool_Pump_Calculator_2020.05.05_FINAL.xlsx).
518 Ameren Missouri Efficient Products Evaluation: PY2019.

3.7 Building Shell

3.7.1 Air Sealing

DESCRIPTION

Thermal shell air leaks are sealed through strategic use and location of air-tight materials. An estimate of savings is provided in two ways. It is highly recommended that leaks be detected and pre- and post-sealing leakage rates measured with the assistance of a blower-door by qualified/certified inspectors.[519] Where this occurs, an algorithm is provided to estimate the site-specific savings. Where test in/test out has not occurred, a conservative deemed assumption is provided.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Air sealing materials and diagnostic testing should meet all eligibility program qualification criteria. The initial and final tested leakage rates should be assessed in such a manner that the identified reductions can be properly discerned, particularly in situations wherein multiple building envelope measures may be implemented simultaneously.

DEFINITION OF BASELINE EQUIPMENT

The existing air leakage should be determined through approved and appropriate test methods using a blower door. The baseline condition of a building upon first inspection significantly affects the opportunity for cost-effective energy savings through air sealing.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[520]

DEEMED MEASURE COST

The actual capital cost for this measure should be used.

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Test In / Test Out Approach

$$\Delta kWh = \Delta kWh\_cooling + \Delta kWh\_heating$$

Where:

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to air sealing

$$= \frac{\left(\frac{CFM50_{Pre} - CFM50_{Post}}{N_{cool}}\right) * 60 * 24 * CDD * DUA * 0.018 * LM}{(1000 * \eta Cool)}$$

$CFM50_{Pre}$ = Infiltration at 50 Pascals as measured by blower door before air sealing

= Actual[521]

$CFM50_{Post}$ = Infiltration at 50 Pascals as measured by blower door after air sealing

= Actual

[519] Refer to the Energy Conservatory Blower Door Manual for more information on testing methodologies.

[520] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007.

[521] Because the pre- and post-sealing blower door test will occur on different days, there is a potential for the wind and temperature conditions on the two days to affect the readings. There are methodologies to account for these effects. For wind – first, if possible, avoid testing in high wind, place blower door on downwind side, take a pre-test baseline house pressure reading, adjust house pressure readings by subtracting the baseline reading, and use the time averaging feature on the digital gauge, etc. Corrections for air density due to temperature swings can be accounted for with air density correction factors. Refer to the Energy Conservatory Blower Door Manual for more information.

N_{cool} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions
=Dependent on number of stories:[522]

Weather Basis (City based upon)	N_cool (by # of stories)			
	1	1.5	2	3
St Louis, MO	34.9	30.9	28.3	25.1

60 * 24 = Converts cubic feet per minute to cubic feet per day
CDD = Cooling Degree Days:[523]

Weather Basis (City based upon)	CDD 65
St Louis, MO	1646

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it)
= 0.75[524]
0.018 = Specific heat capacity of air (Btu/ft^3*°F)
1000 = Converts Btu to kBtu
ηCool = Efficiency (SEER) of air conditioning equipment (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate) - if unknown, assume the following:[525]

Age of Equipment	SEER Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

LM = Latent multiplier to account for latent cooling demand: [526]

Weather Basis (City based upon)	LM
St Louis, MO	3.0

ΔkWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to air sealing

$$= \frac{\frac{(CFM50_{Pre} - CFM50_{Post})}{N\_heat} * 60 * 24 * HDD * 0.018}{(\eta Heat * 3{,}412)}$$

N_heat = Conversion factor from leakage at 50 Pascal to leakage at natural conditions
= Based on building height:[527]

Weather Basis (City based upon)	N_heat (by # of stories)			
	1	1.5	2	3
St Louis, MO	24.0	21.3	19.5	17.3

[522] N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30-year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc" and calculation worksheets.
[523] Based on climate normals data with a base temperature of 65°F.
[524] This factor's source: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," p31.
[525] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.
[526] The LM is used to convert the sensible cooling savings calculated to a value representing sensible and latent cooling loads. The values are derived from the methodology outlined in Infiltration Factor Calculation Methodology by Bruce Harley, Senior Manager, Applied Building Science, CLEAResult 11/18/2015 and is based upon an 8760 analysis of sensible and total heat loads using hourly climate data.
[527] N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc" and calculation worksheets.

HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4486

ηHeat = Efficiency of heating system
= Actual - if not available refer to default table below:[528]

System Type	Age of Equipment	HSPF Estimate	ηHeat (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 and after	8.2	2.04
Resistance	N/A	N/A	1

3412 = Converts Btu to kWh

<u>Conservative Deemed Approach</u>

$$\Delta kWh = SavingsPerUnit * SqFt$$

Where:

SavingsPerUnit = Annual savings per square foot, dependent on heating / cooling equipment[529]

Building Type	HVAC System	SavingsPerUnit (kWh/ft)
Manufactured	Central Air Conditioner	0.062
Multifamily	Central Air Conditioner	0.043
Single Family	Central Air Conditioner	0.050
Manufactured	Electric Furnace/Resistance Space Heat	0.413
Multifamily	Electric Furnace/Resistance Space Heat	0.285
Single Family	Electric Furnace/Resistance Space Heat	0.308
Manufactured	Air Source Heat Pump	0.391
Multifamily	Air Source Heat Pump	0.251
Single Family	Air Source Heat Pump	0.308
Manufactured	Air Source Heat Pump - Cooling	0.062
Multifamily	Air Source Heat Pump - Cooling	0.043
Single Family	Air Source Heat Pump - Cooling	0.050
Manufactured	Air Source Heat Pump - Heating	0.329
Multifamily	Air Source Heat Pump - Heating	0.208
Single Family	Air Source Heat Pump - Heating	0.257

SqFt = Building conditioned square footage
= Actual

[528] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

[529] The values in the table represent estimates of savings from a 15% improvement in air leakage. The values are half those provided by Cadmus for the Iowa Joint Assessment, based on building simulations performed. While 30% savings are certainly achievable, this represents a thorough job in both the attic and basements and could not be verified without testing. The conservative 15% estimate is more appropriate for a deemed estimate. These values should be re-evaluated if EM&V values provide support for a higher deemed estimate.

Additional Fan savings

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time
= ΔTherms * F_e * 29.3

F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%[530]

29.3 = kWh per therm

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = Energy Savings as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805[531]

NATURAL GAS SAVINGS

Test In / Test Out Approach

If natural gas heating:

$$\Delta Therms = \frac{\frac{(CFM50_{Pre} - CFM50_{Post})}{N\_heat} * 60 * 24 * HDD * 0.018}{(\eta Heat * 100{,}000)}$$

Where:

N_heat = Conversion factor from leakage at 50 Pascal to leakage at natural conditions
= Based on building height:[532]

Weather Basis (City based upon)	N_heat (by # of stories)			
	1	1.5	2	3
St Louis, MO	24.0	21.3	19.5	17.3

HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4486

ηHeat = Efficiency of heating system
= Equipment efficiency * distribution efficiency
= Actual[533] - if not available, use 71%[534]

Other factors as defined above

Conservative Deemed Approach

$$\Delta kWh = SavingsPerUnit * SqFt$$

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530 F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See "Furnace Fan Analysis.xlsx" for reference.

531 Based on Ameren Missouri 2016 loadshape for residential HVAC end-use.

532 N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30-year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc" and calculation worksheets.

533 Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf - or by performing duct blaster testing.

534 This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

Where:

SavingsPerUnit = Annual savings per square foot, dependent on heating / cooling equipment[535]

Building Type	HVAC System	SavingsPerUnit (Therms/ft)
Manufactured	Gas Boiler	0.022
Multifamily	Gas Boiler	0.018
Single Family	Gas Boiler	0.016
Manufactured	Gas Furnace	0.017
Multifamily	Gas Furnace	0.012
Single Family	Gas Furnace	0.013

SqFt = Building square footage
= Actual

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

535 The values in the table represent estimates of savings from a 15% improvement in air leakage. The values are half those provided by Cadmus for the Iowa Joint Assessment, based on building simulations performed. While 30% savings are certainly achievable, this represents a thorough job in both the attic and basements and could not be verified without testing. The conservative 15% estimate is more appropriate for a deemed estimate. These values should be re-evaluated if EM&V values provide support for a higher deemed estimate.

3.7.2 Ceiling Insulation

DESCRIPTION

This measure describes savings from adding insulation to the attic/ceiling. This measure requires a member of the implementation staff evaluating the pre- and post-project R-values and to measure surface areas. The efficiency of the heating and cooling equipment in the home should also be evaluated if possible.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[536]

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

LOADSHAPE

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Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$$

Where

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to insulation

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{R_{Attic}}\right) * A_{attic} * (1 - FramingFactor_{Attic}) * CDD * 24 * DUA}{(1000 * \eta Cool)}$$

R_{Attic} = R-value of new attic assembly including all layers between inside air and outside air (ft^2.°F.h/Btu)

R_{Old} = R-value value of existing assembly and any existing insulation (Minimum of R-5 for uninsulated assemblies[537])

A_{Attic} = Total area of insulated ceiling/attic (ft^2)

$FramingFactor_{Attic}$ = Adjustment to account for area of framing = 7%[538]

CDD = Cooling Degree Days:[539]

Weather Basis (City based upon)	CDD 65
St Louis, MO	1646

24 = Converts days to hours

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it)

[536] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

[537] An estimate based on review of Madison Gas and Electric, Exterior Wall Insulation, R-value for no insulation in walls, and NREL's Building Energy Simulation Test for Existing Homes (BESTEST-EX).

[538] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1

[539] Based on climate normals data with a base temp of 65°F.

= 0.75[540]
1000 = Converts Btu to kBtu
ηCool = Seasonal energy efficiency ratio of cooling system (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate) - if unknown, assume the following:[541]

Age of Equipment	ηCool Estimate
Before 2006	10
2006 - 2014	13
Central AC after 1/1/2015	13
Heat Pump after 1/1/2015	14

kWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{R_{Attic}}\right) * A_{Attic} * (1 - FramingFactor_{Attic}) * HDD * 24 * ADJAttic}{(\eta Heat * 3{,}412)}$$

HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

ηHeat = Efficiency of heating system
= Actual - if not available, refer to default table below:[542]

System Type	Age of Equipment	HSPF Estimate	ηHeat (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.9
	2015 and after	8.2	2.0
Resistance	N/A	N/A	1.0

3,412 = Converts Btu to kWh
ADJ_{Attic} = Adjustment for attic insulation to account for prescriptive engineering algorithms consistently overclaiming savings.
= 74%[543]
ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time
= ΔTherms * F_e * 29.3
Where:
F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%[544]
29.3 = kWh per therm

[540] This factor's source: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," p31.
[541] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.
[542] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.
[543] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation," August 2012. See "Insulation ADJ calculations.xls" for details or calculation.
[544] F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See "Furnace Fan Analysis.xlsx" for reference.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0. 0004660805[545]

NATURAL GAS SAVINGS

ΔTherms (if Natural Gas heating)

$$= \frac{\left(\frac{1}{R_{old}} - \frac{1}{R_{attic}}\right) * A_{Attic} * (1 - FramingFactor_{Attic}) * HDD * 24 * ADJAttic}{(\eta Heat * 100{,}000)}$$

Where:

HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

ηHeat = Efficiency of heating system
= Equipment efficiency * distribution efficiency
= Actual.[546] If unknown, assume 71%.[547]

100,000 = Converts Btu to therms

Other factors as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[545] Based on Ameren Missouri 2016 loadshape for residential HVAC end-use.

[546] Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf) - or by performing duct blaster testing.

[547] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

3.7.3 Duct Insulation

DESCRIPTION

This measure describes evaluating the savings associated with performing duct insulation on the distribution system of homes with central cooling and/or a ducted heating system. While insulating ducts in conditioned space can help with control and comfort, energy savings are largely limited to insulating ducts in unconditioned space where the heat loss is to outside the thermal envelope. Therefore, for this measure to be applicable, at least 30% of ducts should be within unconditioned space (e.g., attic with floor insulation, vented crawlspace, unheated garages. Basements should be considered conditioned space).

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is insulated duct work throughout the unconditioned space in the home.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is existing duct work with at least 30% of the ducts within the unconditioned space in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years.[548]

DEEMED MEASURE COST

The actual duct insulation measure cost should be used.

LOADSHAPE

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Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric energy savings is calculated as the sum of energy saved when cooling the home and energy saved when heating the home.

$$\Delta kWh = \Delta kWh_{Cooling} + \Delta kWh_{Heating}$$

If central cooling, the electric energy saved in annual cooling due to the added insulation is

$$\Delta kWh_{Cooling} = \frac{\left(\frac{1}{R_{existing}} - \frac{1}{R_{new}}\right) * Area * EFLH_{cool} * \Delta T_{AVG,cooling}}{(1{,}000 * SEER)}$$

Where:

$R_{existing}$ = Duct heat loss coefficient with existing insulation ((hr-^{0}F-ft^2)/Btu)
= Actual

R_{new} = Duct heat loss coefficient with new insulation (hr-^{0}F-ft^2)/Btu)
= Actual

Area = Area of the duct surface exposed to the unconditioned space that has been insulated (ft^2)

$EFLH_{cool}$ = Equivalent Full Load Cooling Hours:

[548] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

Weather Basis (Ameren Missouri Average)	EFLHcool (Hours)
SF or MF	869[549]
MFc (comprehensive envelope)	632[550]

$\Delta T_{AVG,cooling}$ = Average temperature difference (^{0}F) during cooling season between outdoor air temperature and assumed 60^{0}F duct supply air temperature[551]

Weather Basis (City based upon)	$OA_{AVG,cooling}$ [°F][552]	$\Delta T_{AVG,cooling}$ [°F]
St Louis, MO	80.8	20.8

1,000 = Converts Btu to kBtu

SEER = Efficiency in SEER of air conditioning equipment

= Actual - If not available, use:[553]

Equipment Type	Age of Equipment	SEER Estimate
Central AC	Before 2006	10
	After 2006	13
Heat Pump	Before 2006	10
	2006-2014	13
	2015 on	14

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the added insulation is:

$$\Delta kWh_{HeatingElectric} = \frac{\left(\frac{1}{R_{existing}} - \frac{1}{R_{new}}\right) * Area * EFLH_{heat} * \Delta T_{AVG,heating}}{(3{,}412 * COP)}$$

Where:

EFLHheat = Equivalent Full Load Heating Hours:[554]

Weather Basis (Ameren Missouri Average)	EFLHheat (Hours)
SF or MF	1,496
MFc (comprehensive envelope)	509

$\Delta T_{AVG,heating}$ = Average temperature difference (^{0}F) during heating season between outdoor air temperature and assumed 115ºF duct supply temperature[555]

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549 Based on Full Load Hour assumptions (for St Louis and Kansas City) taken from the ENERGY STAR® calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) and reduced by 28.5% based on the evaluation results in Ameren territory suggesting an appropriate EFLH of 869.The other weather basis values are calculated using the relative climate normals cooling degree day ratios (at 65F set point).

550 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

551 Leaving coil air temperatures are typically about 55ºF. Therefore, 60ºF is used as an average temperature, recognizing that some heat transfer occurs between the ductwork and the environment it passes through.

552 National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3 http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html . Heating season defined as September 17th through April 13th, cooling season defined as May 20 through August 15th. For cooling season, temperatures from 8AM to 8PM were used to establish average temperatures as this is when cooling systems are expected to be loaded.

553 These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

554 Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

555 Forced air supply temperatures are typically 130ºF. 115ºF is used as an average temperature, recognizing that some heat transfer occurs between the ductwork and the environment it passes through.

Weather Basis (City based upon)	$OA_{AVG,heating}$ [°F][556]	$\Delta T_{AVG,heating}$ [°F]
St Louis, MO	43.2	71.8

3,412 = Converts Btu to kWh
COP = Efficiency in COP of heating equipment
= Actual - if not available, use:[557]

System Type	Age of Equipment	HSPF Estimate	COP (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1

If the building is heated with a gas furnace, there will be some electric savings in heating the building attributed to extra insulation since the furnace fans will run less.

$$\Delta kWhHeating_{Gas} = (\Delta Therms * Fe * 29.3)$$

Where:

ΔTherms = Therm savings as calculated in Natural Gas Savings
F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%[558]
29.3 = Converts therms to kWh

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWhCooling = Electric energy savings for cooling, calculated above
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805

NATURAL GAS SAVINGS

If home uses a gas heating system, the savings resulting from the insulation is calculated with the following formula.

$$\Delta Therms = \frac{\left(\frac{1}{R_{existing}} - \frac{1}{R_{new}}\right) * Area * EFLHheat * \Delta T_{AVG,heating}}{(100,000 * \eta Heat)}$$

Where:All factors as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[556] National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3 http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html . Heating season defined as September 17 through April 13, cooling season defined as May 20 through August 15. For cooling season, temperatures from 8AM to 8PM were used to establish average temperatures as this is when cooling systems are expected to be loaded.

[557] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

[558] F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e.

3.7.4 Floor Insulation

DESCRIPTION

Insulation is added to the floor above a vented crawl space that does not contain pipes or HVAC equipment. If there are pipes, HVAC, or a basement, it is desirable to keep them within the conditioned space by insulating the crawl space walls and ground. Insulating the floor separates the conditioned space above from the space below the floor and is only acceptable when there is nothing underneath that could freeze or would operate less efficiently in an environment resembling the outdoors. Even in the case of an empty, unvented crawl space, it is still considered best practice to seal and insulate the crawl space perimeter rather than the floor. Not only is there generally less area to insulate this way, but there are also moisture control benefits. There is a "Foundation Sidewall Insulation" measure for perimeter sealing and insulation. This measure assumes the insulation is installed above an unvented crawl space and should not be used in other situations.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no insulation on any surface surrounding a crawl space.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[559]

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

LOADSHAPE

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Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Where available, savings from shell insulation measures should be determined through a custom analysis. When that is not feasible, the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings:

$$\Delta kWh = (\Delta kWh\_cooling + \Delta kWh\_heating)$$

Where:

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to insulation

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{(R_{Added} + R_{Old})}\right) * Area * (1 - Framing\ Factor) * CDD * 24 * DUA}{(1000 * \eta Cool)}$$

R_{Old} = R-value value of floor before insulation, assuming 3/4" plywood subfloor and carpet with pad

= Actual -- if unknown, assume 3.96[560]

R_{Added} = R-value of additional spray foam, rigid foam, or cavity insulation.

Area = Total floor area to be insulated

Framing Factor = Adjustment to account for area of framing

= 12%[561]

24 = Converts hours to days

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[559] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007.

[560] Based on 2005 ASHRAE Handbook – Fundamentals: assuming 2x8 joists, 16" OC, ¾" subfloor, ½" carpet with rubber pad, and accounting for a still air film above and below: 1/ [(0.85 cavity share of area / (0.68 + 0.94 + 1.23 + 0.68)) + (0.15 framing share / (0.68 + 7.5" * 1.25 R/in + 0.94 + 1.23 + 0.68))] = 3.96.

[561] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1.

CDD = Cooling Degree Days

Weather Basis (City based upon)	Unconditioned Space CDD 75 [562]
St Louis, MO	762

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions call for it).
= 0.75[563]

1000 = Converts Btu to kBtu

ηCool = Seasonal energy efficiency ratio of cooling system (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate). If unknown, assume the following:[564]

Age of Equipment	ηCool Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

ΔkWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{(R_{Added} + R_{Old})}\right) * Area * (1 - Framing\ Factor) * HDD * 24 * ADJ_{Floor}}{(\eta Heat * 3{,}412)}$$

HDD = Heating Degree Days:

Weather Basis Zone (City based upon)	Unconditioned Space HDD 50 [565]
St Louis, MO	1911

ηHeat = Efficiency of heating system
= Actual -- if not available, refer to default table below:[566]

System Type	Age of Equipment	HSPF Estimate	ηHeat (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.9
	2015 and after	8.2	2.0
Resistance	N/A	N/A	1.0

ADJ_{Floor} = Adjustment for floor insulation to account for prescriptive engineering algorithms overclaiming savings.
= 88%[567]

Other factors as defined above

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time
= ΔTherms * F_e * 29.3

F_e = Furnace fan energy consumption as a percentage of annual fuel consumption

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562 The base temperature should be the outdoor temperature at which the desired indoor temperature stays constant in balance with heat loss or gain to the outside and internal gains. Since unconditioned basements are allowed to swing in temperature, are ground coupled, and are usually cool, they have a bigger delta between the two (heating and cooling) base temperatures. 75F for cooling and 50F for heating are used based on professional judgment. Five-year average cooling degree days with 75F base temp are provided from DegreeDays.net because the 30 year climate normals from NCDC are not available at base temps above 72F.

563 Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," p31.

564 These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

565 The base temperature should be the outdoor temperature at which the desired indoor temperature stays constant in balance with heat loss or gain to the outside and internal gains. Since unconditioned basements are allowed to swing in temperature, are ground coupled, and are usually cool, they have a bigger delta between the two (heating and cooling) base temperatures. 75F for cooling and 50F for heating are used based on professional judgment. National Climatic Data Center, calculated from 1981-2010 climate normals.

566 These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

567 Based upon comparing algorithm-derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation," August 2012. See "Insulation ADJ calculations.xls" for details or calculation. Note that basement wall is used as a proxy for crawlspace ceiling.

= 3.14%[568]
29.3 = kWh per therm

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0. 0004660805[569]

NATURAL GAS SAVINGS

ΔTherms (if Natural Gas heating)

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{(R_{Added} + R_{Old})}\right) * Area * (1 - Framing\ Factor) * HDD * 24 * ADJ_{Floor}}{(\eta Heat * 100{,}000)}$$

Where

ηHeat = Efficiency of heating system
= Equipment efficiency * distribution efficiency
= Actual[570] - If not available, use 71%[571]
100,000 = Converts Btu to therms
Other factors as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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[568] F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.
[569] Based on Ameren Missouri 2016 loadshape for residential building shell end-use.
[570] Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf - or by performing duct blaster testing.
[571] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

3.7.5 Foundation Sidewall Insulation

DESCRIPTION

Insulation is added to a basement or crawl space. Insulation added above ground in conditioned space is modeled the same as wall insulation. Below ground insulation is adjusted with an approximation of the thermal resistance of the ground. Insulation in unconditioned spaces is modeled by reducing the degree days to reflect the smaller but non-zero contribution to heating and cooling load. Cooling savings only consider above grade insulation, as below grade has little temperature difference during the cooling season.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no basement wall or ceiling insulation.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[572]

DEEMED MEASURE COST
The actual installed cost for this measure should be used in screening.

LOADSHAPE
Building Shell RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS
Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

$$\Delta kWh = (\Delta kWh\_cooling + \Delta kWh\_heating)$$

Where:

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to Insulation

$$= \frac{\left(\frac{1}{R_{OldAG}} - \frac{1}{(R_{Added} + R_{OldAG})}\right) * L_{BWT} * H_{BWAG} * (1 - FF) * CDD * 24 * DUA}{(1{,}000 \ * \ \eta Cool)}$$

R_{Added} = R-value of additional spray foam, rigid foam, or cavity insulation.

R_{OldAG} = R-value value of foundation wall above grade.
= Actual, if unknown assume 1.0[573]

L_{BWT} = Length (Basement Wall Total) of basement wall around the entire insulated perimeter (ft)

H_{BWAG} = Height (Basement Wall Above Grade) of insulated basement wall above grade (ft)

FF = Framing Factor, an adjustment to account for area of framing when cavity insulation is used
= 0% if spray foam or external rigid foam
= 25% if studs and cavity insulation[574]

24 = Converts hours to days

CDD = Cooling Degree Days
= Dependent whether basement is conditioned:

Weather Basis (City based upon)	Conditioned Space CDD 65 [575]	Unconditioned Space CDD 75 [576]
St Louis, MO	1646	762

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).
= 0.75[577]

1,000 = Converts Btu to kBtu

ηCool = Seasonal energy efficiency ratio of cooling system (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:[578]

Age of Equipment	ηCool Estimate
Before 2006	10

[572] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007.
[573] ORNL Builders Foundation Handbook, crawl space data from Table 5-5: Initial Effective R-values for Uninsulated Foundation System and Adjacent Soil, 1991, http://www.ornl.gov/sci/roofs+walls/foundation/ORNL_CON-295.pdf.
[574] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1
[575] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.
[576] The base temperature should be the outdoor temperature at which the desired indoor temperature stays constant, in balance with heat loss or gain to the outside and internal gains. Since unconditioned basements are allowed to swing in temperature, are ground coupled, and are usually cool, they have a bigger delta between the two (heating and cooling) base temperatures. 75F for cooling and 50F for heating are used based on professional judgment. Five year average cooling degree days with 75F base temp are provided from DegreeDys.net because the 30 year climate normals from NCDC are not available at base temps above 72F.
[577] This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," p31.
[578] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

ΔkWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

$$= \frac{\left(\begin{array}{c}\left(\left(\frac{1}{R_{OldAG}} - \frac{1}{(R_{Added} + R_{OldAG})}\right) * L_{BWT} * H_{BWAG} * (1 - FF)\right) + \\ \left(\left(\frac{1}{R_{OldBG}} - \frac{1}{(R_{Added} + R_{OldBG})}\right) * L_{BWT} * (H_{BWT} - H_{BWAG}) * (1 - FF)\right)\end{array}\right) * HDD * 24 * DUA * ADJ_{Basement}}{(3{,}412 * \eta Heat)}$$

Where

R_{OldBG} = R-value value of foundation wall below grade (including thermal resistance of the earth)[579]

= dependent on depth of foundation (H_basement_wall_total – H_basement_wall_AG):

= Actual R-value of wall plus average earth R-value by depth in table below

For example, for an area that extends 5 feet below grade, an R-value of 7.46 would be selected and added to the existing insulation R-value.

Below Grade R-value									
Depth below grade (ft)	0	1	2	3	4	5	6	7	8
Earth R-value (°F-ft^2-h/Btu)	2.44	4.50	6.30	8.40	10.44	12.66	14.49	17.00	20.00
Average Earth R-value (°F-ft2-h/Btu)	2.44	3.47	4.41	5.41	6.42	7.46	8.46	9.53	10.69
Total BG R-value (earth + R-1.0 foundation) default	3.44	4.47	5.41	6.41	7.42	8.46	9.46	10.53	11.69

H_{BWT} = Total height of basement wall (ft)

HDD = Heating Degree Days

= dependent on whether basement is conditioned:

Weather Basis (City based upon)	Conditioned Space HDD 65 [580]	Unconditioned Space HDD 50 [581]
St Louis, MO	4,486	1,911

ηHeat = Efficiency of heating system

= Actual. If not available refer to default table below:[582]

System Type	Age of Equipment	HSPF Estimate	ηHeat (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.9
	2015 and after	8.2	2.0
Resistance	N/A	N/A	1.0

$ADJ_{Basement}$= Adjustment for basement wall insulation to account for prescriptiveengineering algorithms overclaiming savings.

579 Adapted from Table 1, page 24.4, of the 1977 ASHRAE Fundamentals Handbook.

580 National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest. National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004.

581 The base temperature should be the outdoor temperature at which the desired indoor temperature stays constant in balance with heat loss or gain to the outside and internal gains. Since unconditioned basements are allowed to swing in temperature, are ground coupled, and are usually cool, they have a bigger delta between the two (heating and cooling) base temperatures. 75F for cooling and 50F for heating are used based on professional judgment. National Climatic Data Center, calculated from 1981-2010 climate normals.

582 These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

= 88%[583]

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time

= ΔTherms * F_e * 29.3

F_e = Furnace fan energy consumption as a percentage of annual fuel consumption

= 3.14%[584]

29.3 = kWh per therm

SUMMER COINCIDENT PEAK DEMAND

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0. 0004660805[585]

NATURAL GAS SAVINGS

If Natural Gas heating:

ΔTherms =

$$= \frac{\left(\left(\left(\frac{1}{R_{OldAG}} - \frac{1}{(R_{Added} + R_{OldAG})}\right) * L_{BWT} * H_{BWAG} * (1 - FF)\right) + \left(\left(\frac{1}{R_{OldBG}} - \frac{1}{(R_{Added} + R_{OldBG})}\right) * L_{BWT} * (H_{BWT} - H_{BWAG}) * (1 - FF)\right)\right) * HDD * 24 * ADJ_{Basement}}{(100{,}000 * \eta Heat)}$$

Where

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual[586] - If not available, use 71%[587]

100,000 = Converts Btu to therms

Other factors as defined above

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[583] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation," August 2012. See "Insulation ADJ calculations.xls" for details or calculation.

[584] F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

[585] Based on Ameren Missouri 2016 loadshape for residential building shell end-use.

[586] Ideally, the system efficiency should be obtained either by recording the AFUE of the unit or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf - or by performing duct blaster testing.

[587] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

3.7.6 Storm Windows

DESCRIPTION

Storm windows installed on either the interior or exterior of existing window assemblies can reduce both heating and cooling loads by reducing infiltration and solar heat gain and improving insulation properties. Glass options for storm windows can include traditional clear glazing as well as low-emissivity (Low-E) glazing. Low-E glass is formed by adding an ultra-thin layer of metal to clear glass. The metallic-oxide (pyrolytic) coating is applied when the glass is in its molten state, and the coating becomes a permanent and extremely durable part of the glass. This coating is also known as "hard-coat" Low-E. Low-E glass is designed to redirect heat back towards the source, effectively providing higher insulating properties and lower solar heat gain as compared to traditional clear glass. This characterization captures the savings associated with installing storm windows to an existing window assembly (retrofit).

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

An interior or exterior storm window installed according to manufacturer specifications.

DEFINITION OF BASELINE EQUIPMENT

The existing window assembly.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

20 years[588]

DEEMED MEASURE COST

The actual capital cost for this measure should be used when available and include both material and labor costs. If unavailable, the cost for a low-e storm window can be assumed as \$7.85/ft^2 of window area (material cost) plus \$30 per window for installation expenses.[588] For clear glazing, cost can be assumed as \$6.72/ft^2 of window area (material cost) plus \$30 per window for installation expenses.[589]

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

The following reference tables show savings factors (kBtu/ft^2) for both heating and cooling loads for each of the seven weather zones defined by the TRM.[590] They are used with savings equations listed in the electric energy and gas savings sections to produce savings estimates. If storm windows are left installed year-round, both heating and cooling savings may be claimed. If they are installed seasonally, only heating savings should be claimed. Savings are dependent on location, storm window location (interior or exterior), glazing type (clear or Low-E) and existing window assembly type.

[588] Task ET-WIN-PNNL-FY13-01_5.3: Database of Low-E Storm Window Energy Performance across U.S. Climate Zones. KA Cort and TD Culp, September 2013. Prepared for the U.S. Department of Energy by Pacific Northwest National Laboratory. PNNL-22864.

[589] A comparison of Low-E to clear glazed storm windows available at large national retail outlets showed the average incremental cost for Low-E glazing to be \$1.13/ft^2. Installation costs are identical.

[590] Savings factors are based on simulation results, documented in "Storm Windows Savings.xlsx."

St Louis, MO

Heating:

Savings in kBtu/ft²		Base Window Assembly			
		SINGLE PANE, DOUBLE HUNG	DOUBLE PANE, DOUBLE HUNG	SINGLE PANE, FIXED	DOUBLE PANE, FIXED
Storm Window Type	CLEAR EXTERIOR	47.7	13.3	48.5	12.3
	CLEAR INTERIOR	49.8	17.9	49.0	14.2
	LOW-E EXTERIOR	51.5	13.3	53.2	19.3
	LOW-E INTERIOR	57.7	20.3	55.9	17.5

Cooling:

Savings in kBtu/ft²		Base Window Assembly			
		SINGLE PANE, DOUBLE HUNG	DOUBLE PANE, DOUBLE HUNG	SINGLE PANE, FIXED	DOUBLE PANE, FIXED
Storm Window Type	CLEAR EXTERIOR	23.0	10.5	22.5	9.6
	CLEAR INTERIOR	23.9	10.7	24.4	9.8
	LOW-E EXTERIOR	29.5	15.4	29.3	9.3
	LOW-E INTERIOR	28.8	14.2	29.0	13.4

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$$

Where:

$\Delta kWh_{cooling}$ = If storm windows are left installed during the cooling season and the home has central cooling, the reduction in annual cooling requirement due to air sealing

$$= \frac{\Sigma_{cool} * A}{\eta Cool}$$

Σ_{cool} = Savings factor for cooling, as tabulated above.

A = Area (square footage) of storm windows installed.

ηCool = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate) - If unknown, assume the following:[591]

Age of Equipment	SEER Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

$\Delta kWh_{heating}$ = If electric heat (resistance or heat pump), reduction in annual electric heating due to air sealing

$$= \frac{\Sigma_{heat} * A}{\eta Heat * 3.412}$$

Σ_{heat} = Savings factor for heating, as tabulated above.

ηHeat = Efficiency of heating system

[591] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

= Actual - If not available refer to default table below:[592]

System Type	Age of Equipment	HSPF Estimate	ηHeat (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 and after	8.2	2.04
Resistance	N/A	N/A	1

3.412 = Converts kBtu to kWh

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh_cooling = As calculated above.

CF = Summer System Peak Coincidence Factor for Cooling

= 0. 0004660805[593]

NATURAL GAS SAVINGS

If Natural Gas heating:

$$\Delta Therms = \frac{\Sigma_{heat} * A}{\eta Heat * 100}$$

Where:

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual[594] - If not available, use 71%[595]

100 = Converts kBtu to therms

Other factors as defined above

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

[592] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

[593] Based on Ameren Missouri 2016 loadshape for residential building shell end-use.

[594] Ideally, the system efficiency should be obtained either by recording the AFUE of the unit or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf - or by performing duct blaster testing.

[595] This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

3.7.7 Kneewall and Sillbox Insulation

DESCRIPTION

This measure describes savings from adding insulation (for example, blown cellulose, spray foam) to wall cavities (this includes kneewall and sillbox areas). This measure requires a member of the implementation staff evaluating the pre- and post-project R-values and to measure surface areas. The efficiency of the heating and cooling equipment in the home should also be evaluated if possible.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be empty wall cavities.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[596]

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$$

Where

$\Delta kWh_{cooling}$ = If central cooling, reduction in annual cooling requirement due to insulation

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{R_{Wall}}\right) * A_{Wall} * (1 - FramingFactor_{Wall}) * CDD * 24 * DUA}{(1{,}000 * \eta Cool)}$$

R_{Wall} = R-value of new wall assembly including all layers between inside air and outside air ($ft^2.°F.h/Btu$)

R_{Old} = R-value value of existing assembly and any existing insulation ($ft^2.°F.h/Btu$)
(Minimum of R-5 for uninsulated assemblies[597])

A_{Wall} = Net area of insulated wall (ft^2)

$FramingFactor_{Wall}$ = Adjustment to account for area of framing
= 25%[598]

CDD = Cooling Degree Days:[599]

Weather Basis (City based upon)	CDD 65
St Louis, MO	1646

24 = Converts days to hours

[596] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007.
[597] An estimate based on review of Madison Gas and Electric, Exterior Wall Insulation, R-value for no insulation in walls, and NREL's Building Energy Simulation Test for Existing Homes (BESTEST-EX).
[598] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1.
[599] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temperature of 65°F.

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it)
= 0.75[600]

1,000 = Converts Btu to kBtu

ηCool = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate) - If unknown, assume the following:[601]

Age of Equipment	ηCool Estimate
Before 2006	10
2006 - 2014	13
Central AC after 1/1/2015	13
Heat Pump after 1/1/2015	14

$kWh_{heating}$ = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

$$= \frac{\left(\frac{1}{R_{Old}} - \frac{1}{R_{Wall}}\right) * A_{wall} * (1 - FramingFactor_{Wall}) * HDD * 24 * ADJWall}{(\eta Heat * 3{,}412)}$$

HDD = Heating Degree Days:[602]

Weather Basis (City based upon)	HDD 65
St Louis, MO	4486

ηHeat = Efficiency of heating system
= Actual - If not available, refer to default table below:[603]

System Type	Age of Equipment	HSPF Estimate	ηHeat (Effective COP Estimate) (HSPF/3.412)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.9
	2015 and after	8.2	2.0
Resistance	N/A	N/A	1.0

3412 = Converts Btu to kWh

ADJ_{Wall} = Adjustment for wall insulation to account for prescriptive engineering algorithms consistently overclaiming savings
= 63%[604]

$\Delta kWh_{heating}$ = If gas *furnace* heat, kWh savings for reduction in fan run time
= ΔTherms * F_e * 29.3
Where:
F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%[605]
29.3 = kWh per therm

[600] This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," p31.
[601] These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.
[602] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004.
[603] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.
[604] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation," August 2012. See "Insulation ADJ calculations.xls" for details or calculation.
[605] F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See "Furnace Fan Analysis.xlsx" for reference.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0. 0004660805[606]

NATURAL GAS SAVINGS

ΔTherms (if Natural Gas heating)

$$= \frac{\left(\frac{1}{R_{old}} - \frac{1}{R_{wall}}\right) * A_{wall} * (1 - FramingFactor_{Wall}) * HDD * 24 * ADJWall}{(\eta Heat * 100{,}000)}$$

Where:

HDD = Heating Degree Days:[607]

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual[608] - If not available, use 71%[609]

100,000 = Converts Btu to therms

Other factors as defined above

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

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606 Based on Ameren Missouri 2016 loadshape for residential building shell end-use.

607 National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004.

608 Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute - (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf - or by performing duct blaster testing.

609 This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

3.8 Miscellaneous

3.8.1 Home Energy Report

DESCRIPTION
These behavior/feedback programs send energy use reports to participating residential electric or gas customers in order to change customers' energy use behavior. Energy savings are evaluated by ex-post billing analysis comparing consumption before and after (or with and without) program intervention and require M&V methods that include customer-specific energy usage regression analysis and randomized controlled trial (RCT) experimental designs, among others (see national protocols developed under the sponsorship of the US Department of Energy[610]). As such, calculation of savings achieved by the program for the year is treated as a custom protocol.

Given that actual monitored energy use is needed, as an ex-post input for these custom calculations, estimates of program savings are used for program planning and goal setting at the beginning of the program cycles. Estimated, or ex ante, values are based on previous actual program performance developed through forecasting analysis from the program implementer, or taken from actual savings values from comparable programs delivered by other program administrators.

HER Program Estimated (Ex Ante) Savings Values

Utility Program	Year	Gross Electric Energy Savings (kWh/home)	Gross Demand Savings (kW/home) [A]
Ameren Missouri Home Energy Report	1	140.37 [B]	0.065422
	2	112.29	0.052337
	3	89.83	0.041870
	4	71.87	0.033496
	5	57.49	0.026797

[A] Demand savings are calculated as the product of the gross electric energy savings and the kW factor for the Building Shell RES end use.
[B] Value is based on the Ameren Missouri Home Energy Report Evaluation PY2021. First year annual energy savings are calculated as PY2021 HER Program Adjusted Net Annual Savings / Number of Customers Treated.

DEFINITION OF EFFICIENT CASE
The efficient case is a customer who receives a Home Energy Report.

DEFINITION OF BASELINE CASE
The baseline case is a customer who does not receive a Home Energy Report.

DEEMED LIFETIME OF PROGRAM SAVINGS
Year one savings represent ex post savings for the final year of treatment. Years two through five represent savings decay from the evaluated savings in year one. Once home energy reports cease, the savings persist for four additional years, with 20% savings decay each year. With this decay rate, second year savings are 80% of savings from the final year of treatment; third year savings are 64% of savings from the final year of treatment (80% of second year savings); fourth year savings are 51.2% of savings from the final year of treatment (80% of third year savings); fifth year savings are 40.96% of savings from the final year of treatment (80% of fourth year savings); and no savings persist beyond the fifth year.[611]

DEEMED MEASURE COST
It is assumed that most behavior changes in residential settings can be accomplished with homeowner labor only and without investment in new equipment. Therefore, without evidence to the contrary, measure costs in such residential programs focused on motivating changes in customer behavior may be defined as $0.

LOADSHAPE
Building Shell RES

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

[610] Evaluation, Measurement, and Verification (EM&V) of Residential Behavior-Based Energy Efficiency Programs: Issues and Recommendations; SEEAction (State and Local Energy Efficiency Action Network- EPA/DOE), 2012; The Uniform Methods Project: Methods for Determining Energy Efficiency Savings for Specific Measures; Residential Behavior Protocol, NREL/ DOE, 2015.
[611] Opinion Dynamics, MEMO: Recommendation for Ameren Missouri HER Program Persistence and EUL; August 2021.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

3.9 Residential Demand Response

3.9.1 Baseline Approach

DESCRIPTION

Residential demand response: For demand and energy savings associated with calling a demand response event, program participants will be randomly partitioned into two groups. In this scenario, on an event day, participants in one group receive a signal to initiate activity on the thermostat (treatment group), while the other group of participants would not receive this signal (control group). Demand impacts will be estimated from the average of the hours over all event periods. Energy savings impacts will be estimated from comparing the 24 hours of the control group for each event day to the 24 hours of actual kWh consumption for each event day. However, if it is not practical or plausible to use this approach (such as everyone is dispatched), other quasi experimental design approaches may be used.

3.9.2 Demand Response Advanced Thermostat

DESCRIPTION

This measure characterizes the energy and demand savings for an advanced thermostat enrolled in the Residential Demand Response (DR) Program. The program controls customer energy loads and also reduces energy usage by utilizing a continuous load shaping strategy during non-peak hours. Savings impacts are evaluated by ex-post analysis comparing demand and consumption with and without program intervention, utilizing field data which may be available through advanced thermostats' 2-way communication ability. The program will require M&V methods that include customer-specific energy usage regression analysis and randomized controlled trial (RCT) experimental designs, among others. As such, calculation of both demand and energy savings achieved by the program for the year are treated as a custom protocol.

Given that actual monitored field data is needed as ex post inputs for these custom calculations, estimates of program savings based on previous year evaluation results are used for program planning and goal setting at the beginning of the program cycles.

As advanced thermostats mature, some models include embedded optimization routines that achieve energy savings. The program differentiates between thermostats with "program-driven optimization," which achieve savings through program influence to operate optional optimization, and without "program-driven optimization," which achieve no energy savings due to either the default optimization baseline or no optimization routine employed.

Demand Response Smart Thermostat Deemed Savings Estimates for 2019-21 Planning

Utility Program	Gross Electric Savings (*Annual*) (kWh/thermostat)	Gross Demand Savings (*Event*) (kW/thermostat)
Demand Response Advanced Thermostat – with Program-Driven Optimization	97.49[612]	0.94[613]
Demand Response Advanced Thermostat – without Program-Driven Optimization	0.00	0.94[614]

This measure was developed to be applicable to the following program type: DR.

DEFINITION OF EFFICIENT CASE

The efficient case is a customer who participated in the DR program.

[612] Average energy savings per device based on Ameren Missouri PY2022 evaluation. See Ameren Missouri Program Year 2022 Annual EM&V Report, Volume 4: Demand Response Portfolio Report, Table 22. Comparison of Savings from Emerson Optimization Using Telemetry and AMI Data Pathways.

[613] Average demand impact per device based on Ameren Missouri PY2022 evaluation. See Ameren Missouri Program Year 2022 Annual EM&V Report, Volume 4: Demand Response Portfolio Report, Table 13. Residential DR Program: Resource Capability Impacts for the number of devices and Appendix Table 3 Residential DR Program: Resource Capability Impacts for the AMI per account kW impacts and number of accounts.

[614] Ibid.

Definition of Baseline Case

The baseline case is a customer who is not participating in the DR program and who has installed a thermostat with default enabled capability—or the capability to automatically—establish a schedule of temperature set points according to driving device inputs above and beyond basic time and temperature data of conventional programmable thermostats. This category of products and services is broad and rapidly advancing with regard to their capability, usability, and sophistication, but at a minimum the baseline customer must have installed a thermostat capable of two-way communication and exceed the typical performance of manual and conventional programmable thermostats through the automatic or default capabilities described above.

Deemed Lifetime of Program Savings

The expected measure life is assumed to be1 year.

Deemed Measure Cost

It is assumed that program-controlled changes in residential settings are accomplished without homeowner investment in new equipment. Therefore, without evidence to the contrary, measure costs in such residential programs focused on program controlled changes in customer behavior may be defined as $0.

Loadshape

HVAC RES (for optimization routines that save energy during the cooling and heating seasons)
Cooling RES (for optimization routines that save energy only during the cooling season)
Heating RES (for optimization routines that save energy only during the heating season)

Water Impact Descriptions and Calculation

N/A

Deemed O&M Cost Adjustment Calculation

N/A